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2	A reduced order computational model of a semi-active variable-stiffness foot prosthesis
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- 24 Introduction: Passive energy storage and return (ESR) feet are the current performance standard in lower limb
- 25 prostheses. A recently developed semi-active variable-stiffness foot (VSF) prosthesis balances the simplicity of a
- 26 passive ESR device with the adaptability of a powered design. The purpose of this study was to model and simulate
- the ESR properties of the VSF prosthesis. **Methods**: The ESR properties of the VSF were modeled as a lumped
- parameter overhung beam. The overhung length is variable, allowing the model to exhibit variable ESR stiffness.
 Foot-ground contact was modeled using sphere-to-plane contact models. Contact parameters were optimized to
- 30 represent the geometry and dynamics of the VSF and its foam base. Static compression tests and gait were
- 31 simulated. Simulation outcomes were compared to corresponding experimental data. **Results**: Stiffness of the model
- 32 matched that of the physical VSF (R²: 0.98, RMSE: 1.37 N/mm). Model-predicted resultant ground reaction force
- (GRF_k) matched well under optimized parameter conditions (R²: 0.98, RMSE: 5.3% body weight.) and unoptimized
- 34 parameter conditions (R: 0.90, mean RMSE: 13% body weight). Anterior-posterior center of pressure matched well
- 35 with $R^2 > 0.94$ and RMSE < 9.5% foot length in all conditions. **Conclusions**: The ESR properties of the VSF were
- 36 accurately simulated under benchtop testing and dynamic gait conditions. These methods may be useful for
- 37 predicting GRF₈ arising from gait with novel prostheses. Such data are useful to optimize prosthesis design
- 38 parameters on a user-specific basis.

39 1. Introduction

40 Individuals with lower limb loss exhibit distinct gait characteristics, which may limit mobility and decrease 41 quality of life. Those using lower limb prostheses may display gait asymmetry [1,2], elevated metabolic cost during 42 locomotion [3], and a variety of psychological disorders including anxiety and depression [4]. Sustained prosthesis 43 use may also induce overloading of intact joints and ultimately, musculoskeletal ailments [5]. Each of these issues 44 may be attenuated by improving user specificity in the design characteristics of foot prostheses. However, the effects 45 of foot prosthesis design parameters (e.g. stiffness) are not well characterized, and thus achieving meaningful 46 improvements in gait has proven arduous [6,7]. In order to achieve improvements, a robust understanding of the 47 relationships between anthropometry, gait mechanics, and prosthesis design are necessary.

48 One of the primary design goals of a lower limb prosthesis is to replace the coordinated energy absorption and 49 generation properties of a lost limb. Passive energy storage and return (ESR) foot prostheses are the current standard 50 for mimicking this functionality. However, the fixed stiffness behavior of these devices contrasts that of the healthy 51 foot-ankle complex, which modulates its behavior in response to varied gait conditions (e.g. velocity and terrain) 52 [8,9]. Glanzer and Adamczyk (2018) [10] recently developed a variable-stiffness foot (VSF) prosthesis designed 53 with an actuated keel support fulcrum to semi-actively control sagittal forefoot stiffness and thereby adapt to 54 different gait conditions with low power (Fig. 1). The ESR keel of the VSF is a composite leaf spring designed as an 55 overhung beam, which modulates the supported length (l) via an actuated keel support fulcrum (B). The total beam 56 length (L) is 229 mm, whereas the overhung length (a) is variable between 66-151 mm. By modulating overhung 57 length, the VSF's forefoot is capable of exhibiting roughly a three-fold range of forefoot stiffness values (10-32 58 N/mm). The heel component of the VSF has a consistent linear stiffness of 65 N/mm. The VSF's fulcrum position is 59 designed to be adjusted during swing phase, thus minimizing the power necessary for actuation. As such, the VSF 60 behaves primarily as a passive ESR prosthesis, which can adapt stiffness in response to variable gait conditions.

61 Simulations based on computational models can be powerful tools for evaluating potential biomechanical 62 interventions, such as the implementation of a novel ESR prosthesis. Recently, simulations have been used to aid in 63 the iterative design process and improve user-specificity [11–13]. Inverse simulations provide the ability to estimate 64 values that cannot be measured *in vivo* (e.g. socket-residual limb interface dynamics), whereas predictive 65 simulations suggest hypotheses regarding how humans may interact with and adapt to new prosthetic devices.

66 Computational modeling has been used to investigate the effects of prosthesis alignment [14] and a biarticular 67 clutched spring mechanism [15] on gait mechanics among persons with lower limb loss. However, these models do 68 not account for the ESR properties of the prosthetic foot, thus limiting their ecological validity. Other studies, which 69 did incorporate the force and torque contributions of ESR feet into gait models focused on characterizing 70 biomechanical and myophysiologic responses with prosthesis use, rather than validation of the prosthesis model 71 [16,17]. While these studies made important progress toward investigating the relationship between anthropometry, 72 gait mechanics, and prosthetic foot design, they had limited ability to verify simulation results in the context of 73 experimental values. Due to these limitations, the use of simulations to inform the design of ESR foot prostheses has 74 not been fully realized. The purpose of this study was to further couple experimental and simulation prosthesis data 75 by modeling and validating the mechanical stiffness properties and resulting ground reaction forces of a semi-active 76 VSF. 77

2. Methods

80 <u>2.1 Model design</u>

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81 A computational model of the VSF was developed in Simscape Multibody (Mathworks, Inc., Natick, MA). 82 The assembly, geometry, mass, and inertial properties were derived from SolidWorks (Dassault Systemes Inc., 83 Waltham, MA). A reduced order model of the VSF's variable-stiffness elastic keel was designed using the lumped 84 parameter approach for approximating flexible body dynamics. This approach involved discretizing the continuous 85 geometry of the keel into finite rigid segments coupled via revolute joints, springs, and dampers (Fig. 2). This 86 simplification of the original state space of the continuous elastic keel system to finite dimensions allows the partial 87 differential equations of the infinite-dimensional time-space states of the physical VSF to be represented by ordinary 88 differential equations with a finite number of parameters.

89 The keel of the VSF model was discretized into 16 segments (eight DoF). The most posterior segment is 66 90 mm in length, which matches the minimum possible fulcrum position. The rest of the keel consists of 11.64-mm 91 segments for a total beam length of 229 mm (Fig. 2). The stiffness and damping values for the revolute joints were 92 parameterized to represent the material properties of the VSF's G10/FR4 Garolite keel (flexural elastic modulus: 93 18.6 GPa, Poisson's ratio: 0.136). A MATLAB script controls continuous fulcrum position (i.e. variable stiffness).
94 The VSF model was rigidly attached to a prosthetic pylon and socket via a pyramid adapter, as the device would be
95 used *in vivo*. These connections were modeled as weld joints. Each segment is independently scalable, allowing the
96 model to be integrated into an anatomically scaled computational gait model.

97 Foot-ground contact consists of 24 sphere-to-plane contact models [18] parameterized to represent the 98 geometry and dynamics of the VSF's foam base. Each of these models estimates normal (F_{i}) and frictional (F_{i}) 99 forces associated with the collision of a viscoelastic sphere (a massless spring and damper system) and a rigid plane 100 (Fig. 3). The overall foot contact model was divided into five zones; the sphere-to-plane models were parameterized 101 by zone (Fig. 3, Table 1). The heel of the VSF model is comprised of three zones; this choice was motivated by the 102 sensitivity of contact parameters when few spheres are in contact with the walking plane (e.g. the heel of the foot 103 early in stance phase). Contact parameters are less sensitive when many spheres are in contact with the walking 104 plane (e.g. the midfoot and forefoot late in stance phase). The foam base of the physical VSF undergoes 105 compression throughout stance phase. To account for these effects, a modified Kelvin-Voigt nonlinear spring and 106 damper force law (eq. (1)) was implemented to represent contact between the VSF and walking plane:

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$$F_n = \begin{cases} (k \times \delta^n) + y(b \times \dot{\delta}) & \delta > 0, \dot{\delta} > 0\\ k \times \delta & \delta > 0, \dot{\delta} < 0\\ 0 & \delta < 0 \end{cases}$$

109 F_{a} : normal force

110 *k*: contact stiffness

111 δ : penetration depth

112 *n*: penetration exponent

113 y: damping force scaling factor

114 **b**: contact damping coefficient

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116 The spring force increases exponentially as the sphere penetrates the contact plane. The damping force is multiplied 117 by a scaling factor (y), which increases from zero to one as a polynomial as it approaches a user-defined value for

full damping. Frictional force (eq. (2)) is the product of the normal force and coefficient of friction (μ). A stick-slip

friction law defines the transition between static (μ_{match}) and kinetic (μ_{match}) coefficients of friction based on a velocity threshold (ν_{match}):

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$$(F_n \times \mu_{static} \quad v_{noc} < v_t)$$

122 $F_f = \begin{cases} F_n \times \mu_{static} & v_{poc} < v_{threshold} \\ F_n \times \mu_{kinetic} & v_{poc} > v_{threshold} \end{cases}$

123 F_{i} : frictional force

124 μ : coefficient of friction

125 v_{poc} : velocity at point of contact

126 $v_{threshold}$: velocity threshold 127

128 Static and kinetic coefficients of friction were set to 0.5 and 0.3 with a velocity threshold of 0.1 m/s. Resultant 129 ground reaction force (GRF_{*}) was derived by summing and low-pass filtering (4^a order Butterworth, f: 40 Hz) the 130 normal and frictional forces arising from each contact sphere.

131 In order to improve GRF_{i} predictions, contact model parameterization was formulated as a least-squares 132 optimization problem with the objective of minimizing the sum of squared errors between model-predicted and 133 experimentally measured GRF_s(see "Model Validation"). Initial parameter settings at the outset of the optimization 134 were derived by increasing stiffness until the contact spheres were able to support the weight of the model. Initial 135 damping coefficients (N·s/mm) were set to half the numerical value of stiffness (N/mm). Penetration exponents and 136 penetration for full damping values were initialized at 1 and 1 mm, respectively. These initial values were used as 137 inputs to the problem. Latin hypercube sampling (LHS) was applied to generate simulation scenarios with pseudo-138 random sets of parameters. The LHS approach is a method of stratified sampling, which divides parameter values 139 into equal strata based on an assumed normal distribution and constrained by user-defined bounds. Random 140 parameter values are sampled from within these strata to generate a simulation scenario with a pseudo-random set of 141 parameters. The LHS technique effectively samples the search space, while providing the randomness required to 142 explore the efficacy of a range of variable values to minimize the objective function. The objective function value of 143 each iteration is compared to the previous iteration; the parameter scenario which best minimizes the objective is

144 passed to the next iteration of the algorithm. The optimization algorithm proceeds for 100 iterations or until an

(2)

(1)

objective function tolerance of 0.1 N is reached (i.e. convergence). If the optimization algorithm did not meet any of
the termination criteria, the initial parameter values were updated using the results of the first run, and an additional
run was initiated. Parameter tolerances were set to 0.001 (varying units) in order to avoid false minima.

149 <u>2.2 Model validation</u>

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151 2.2.1 Static compression testing

152 The operational stiffness range of the physical VSF was determined through static compression testing 153 (TestResources, Shakopee, MN) (Glanzer and Adamczyk 2018). Load was applied at a constant speed of 50 154 mm/min to a point 30 mm proximal to the anterior tip of the VSF (i.e. supported beam length = 199 mm). To 155 validate the ESR properties of the VSF model, a simulated materials testing system (MTS) was developed in 156 Simscape Multibody. The MTS simulator consists of a massless body, which translates vertically according to a 157 user-defined time-position vector (Fig. 2). Simulated static compression tests were performed as in Glanzer and 158 Adamczyk (2018). Contact was maintained throughout VSF deflection. Contact dynamics between the VSF and 159 MTS were estimated using a sphere-to-sphere contact model. Stiffness (k) (eq. (3)) was computed as the average 160 slope of the load-displacement data for loads above 200 N. 161

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$$k = \frac{\overline{\Delta}_{\text{Load}}}{\overline{\Delta}_{\text{Displacement}}}$$
 for loads 200 N to F_{max}

(3)

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Deformation for loads under 200 N was considered to arise primarily from foam compression, rather than keel
 displacement. Mid-range keel displacement was also calculated for the VSF model as the displacement of the keel at
 50 percent of the maximal load applied during the static compression test.

Static compression tests were simulated at five discrete fulcrum positions (66, 87, 108, 129, and 151 mm),
which span the full continuous range of possible positions. Simulation-derived values were compared to those from
static compression tests of the physical VSF via coefficient of determination and root mean squared error (RMSE).
Simulations were calculated in Simscape Multibody using the *ode15s* solver profile with variable step size.

172 2.2.2 Gait conditions

173 Model-predicted GRF₈ was validated under two scenarios: static and dynamic gait conditions. For both 174 validations, the VSF model was integrated into a seven-segment, 28-DoF anatomically-scaled gait model of a 175 subject with a unilateral transtibial amputation. Three-dimensional optical motion capture data (Optitrack, Natural 176 Point, Inc. Corvallis, OR) of a male subject (181 cm, 78.0 kg) with a right side transtibial amputation walking with 177 the physical VSF were used as inputs to the model. Retroreflective marker coordinates from a static motion capture trial were used to estimate and scale limb dimensions for the pelvis, leg, intact shank, residual shank, and intact foot. 178 179 Within the gait model, the residual shank was encapsulated in a prosthetic socket and welded to the pyramid adapter 180 of the VSF model (Fig. 2). The interface between the prosthetic socket and residual limb was modeled as a high-181 stiffness 6-DoF bushing joint, similar to previous work by LaPrè et al. (2018). The rotational and translational 182 stiffness as well as displacement and velocity constraints were designed according to previous gait experiments [19] 183 and finite element analysis [20]. The mass and inertial properties of the lower limbs and pelvis were modeled as 184 conical frusta and an ellipsoid, respectively. Segment masses were estimated according to De Leva (1996).

185 For the static condition, the model was simulated with anatomically neutral joint angles for ten seconds. Model-186 predicted GRF₈ was averaged over the course of the trial and compared to the mass of the subject. Dynamic gait 187 simulations were calculated based on experimental motion capture trials of the subject walking over ground between 188 1.0 and 1.2 m/s with the VSF under low, medium and high stiffness configurations (fulcrum positions: 66, 108, and 189 151 mm). Three trials were collected for each stiffness configuration for a total of nine trials. Three-axis pelvis, hip, 190 knee, and ankle angles were calculated from three-dimensional marker coordinate data [22,23] and used as inputs to 191 drive the corresponding joints of the model. Motion at the socket-limb interface was considered to be passive based 192 on the aforementioned velocity and displacement constraints. The pyramid adapter-pylon interface was assumed to 193 be rigid.

194 Contact model-derived GRF_{*} prediction was optimized for a single trial at the 66-mm fulcrum position. The 195 GRF_{*} error resulting from this trial represents the theoretical optimal performance of the comprehensive VSF-ground 196 contact model. The transferability of the optimized parameter values was determined by simulating the two 197 remaining low stiffness trials and the three remaining trials each for the medium and high stiffness configurations.

Joint kinematics and GRF_{R} data were low-pass filtered (4^a order Butterworth: f_{e} : 6 Hz and 40 Hz, respectively). Simulation and experimental GRF_{R} were time locked and indexed to 0.25 s before and 0.25 s after stance phase. Including the brief period before and after stance phase provides insights regarding how the contact

201 model behaves outside of stance phase and whether or not key gait events (e.g. heel strike and toe off) occur at 202 similar time points in the simulated and experimental data. Resultant ground reaction force time series were re-

similar time points in the similated and experimental data. Resultant ground reaction force time series were resampled to 101 data points via cubic spline interpolation to allow for comparison between stance phases of differing

204 lengths. Ensemble curves (mean \pm SD) were generated for each condition. The impulse of GRF_R was calculated to

205 assess the simulation's ability to predict GRF_{s} trajectory.

Anterior-posterior center of pressure (CoP_{xy}) position was calculated as the weighted sum of each contact sphere's predicted force multiplied by its anterior-posterior position (*x*). Raw normal forces arising from each sphere during stance phase were low-pass filtered (4^a order Butterworth: *f*.: 40 Hz) and summed. Anterior-posterior CoP position was calculated across stance phase (eq. (4)).

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$$\operatorname{CoP}_{AP} = \frac{\sum_{i=1}^{N} x_i F_{n_i}}{\sum F_n}$$
(4)

212 **CoP**_{AP}: Anterior-posterior center of pressure position

213 *x*: Anterior posterior coordinate of contact sphere214

The CoP_{av} time series data were low-pass filtered (4* order Butterworth: f_{av} : 6 Hz) and re-sampled to 101 data points via cubic spline interpolation to allow for comparison between stance phases of differing lengths. Joint kinematics, GRF_a, and CoP_{av} data measured during experimental gait trials were compared to those derived from the simulations using coefficient of determination and RMSE.

3. **Results**

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222 <u>3.1 Static compression tests</u>

223 Simulated VSF stiffness effectively reproduced experimental stiffness across the five fulcrum 224 configurations ($R^2 > 0.98$, RMSE = 1.37 N/mm) (Fig. 4, Table 2). Simulated mid-range displacement also matched 225 well ($R^2 > 0.99$) with small offset from experimental displacement in each condition (RMSE = 0.45 mm). 226 Experimental load-displacement relationships were most linear in the 66 and 87 mm fulcrum configurations, as 227 indicated by variance in the slope of the relationship. The stiffest three conditions exhibited curvilinear relationships. 228 Simulated load-displacement data were linear in all conditions due to the linear spring and damper force parameters 229 for the revolute joints in the lumped parameter keel model.

231 <u>3.2 Resultant ground reaction force predictions</u>

In the static condition, model-predicted subject mass was $2.6 \pm 0.0\%$ less than measured mass. In the dynamic conditions, simulated joint angles matched experimental joint angles well, but exhibited a small phase lag (mean RMSE: 1.9 ± 1.0 deg, mean R:: 0.98 ± 0.02). Simulated and experimental GRF₈ data agreed well in the time domain (Fig. 6). Amplitude discrepancies, quantified via RMSE, were least in low stiffness configuration and greatest in the high stiffness configuration. Coefficient of determination values were similar for the low and medium stiffness conditions and lower for the high stiffness condition. Impulse was similar in the low and high stiffness conditions and lower for the medium stiffness condition (Table 3).

239 Optimization of the single low stiffness trial resulted in a GRF_{*} RMSE of 5.3% body weight (BW) and R² of 240 0.98 across stance phase. Impulse also matched well (RMSE: 0.01 BW·s, R² > 0.99) (Fig. 5). In the time domain, 241 model-predicted heel contact preceded experimental heel contact by 0.02 s, resulting in a 0.02-s longer stance phase. 242 Simulating the two additional low stiffness trials with the optimized contact parameters resulted in average RMSE 243 and R² values of 0.10 ± 0.05 BW and 0.93 ± 0.05 for GRF_{*} and 0.02 ± 0.01 BW·s and > 0.99 ± 0.01 for GRF_{*} 244 impulse (Fig. 6, Table 3).

245 Experimental GRF_{*} and GRF_{*} impulse responses were similar in the time and amplitude domains across the 246 three stiffness conditions (Fig. 6). On average, stance phase time was 0.05 ± 0.03 s longer in the simulations across 247 the stiffness conditions. Time errors were least in the low stiffness condition and greatest in the high stiffness. 248 Variability for GRF_{*} was greatest during the first 25% of stance phase for all conditions. Variability for GRF_{*} 249 impulse was greatest near the end of stance phase. The ability of the contact parameters optimized for the low 250 stiffness condition transferred well across the other two conditions, which is evident by the similar RMSE values for 251 GRF₈ (Table 3). Resultant ground reaction force RMSE and R² values were better in the medium stiffness 252 configuration, whereas RMSE and R² were better in the high stiffness condition for GRF₈ impulse. The medium 253 stiffness condition demonstrated the least variability for the GRF₈ response, whereas the low and high stiffness

254 conditions showed similarly low variability for GRF₈ impulse (Table 3).

Anterior-posterior CoP trajectory during stance phase was similar between simulated and experimental data (Fig. 7). Root mean squared errors were 8.9 ± 1.0 , 9.5 ± 0.9 , and 5.7 ± 1.4 percent foot length for the low, medium, and high stiffness conditions, respectively (Table 3). Simulated data correlated well with experimental data across all conditions. Coefficient of determination values were 0.95 ± 0.01 , 0.94 ± 0.01 , and 0.97 ± 0.01 for the low, medium, and high stiffness conditions.

4. Discussion

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263 The goal of this study was to develop a reduced-order computational model of a semi-active variable-264 stiffness foot prosthesis. Results from simulated static compression tests showed good agreement with experimental 265 data. These outcomes suggest that the variable-stiffness ESR properties of the VSF were modeled with high fidelity 266 using a reduced order lumped parameter approach for approximating flexible body dynamics. One of the goals of 267 reduced order modeling is to capture a structure's dynamic behavior in a computationally inexpensive way. A 268 common benchmark for reduced-order models is the ability to simulate at or near real-time [24,25], which contrasts 269 with more computationally expensive methods such as mesh-based finite element modeling. Including initialization 270 time, static compression simulations computed 3.3 ± 0.8 times faster than real-time (i.e. the length of time required 271 to complete the experimental static compression test) on computer with a four core 4.0 GHz processor. Initialization 272 time, which includes model compiling and building, can be minimized using "Accelerator" and "Fast Restart" 273 modes in Simscape Multibody. Using these tools, simulations computed 39 ± 16 times faster than real time. This 274 computational efficiency is useful if the model is to be simulated iteratively, for example in parameter optimization 275 or machine learning frameworks.

276 The range of forefoot stiffness values exhibited by the physical VSF and captured by the VSF model 277 represent a range of stiffness values available in many commercially-available prosthetic feet [26,27]. Accurate 278 characterization of this range is important, should this model be used to inform the design and/or prescription of 279 prosthetic feet. Further, this model can be easily re-parameterized to exhibit a different range of stiffness values, 280 which could aid in the selection of keel dimensions or material properties to meet design goals. Two primary 281 limitations are present for the static compression testing simulations. Experimental load-displacement data were 282 only available for positive loading conditions, and thus a comparison of the model's hysteresis behavior was not 283 possible. Similarly, experimental data were only available for the 50 mm/min loading rate. A robust characterization 284 of the VSF's stiffness behavior under a range of loading rates would likely improve the model's behavior under 285 dynamic conditions. Experimental load-displacement data could also be influenced by imperfections in maintaining 286 a constant contact point with the prosthesis.

287 Under dynamic gait conditions, simulated joint angles agreed well with experimental values, indicating that 288 the model is numerically stable when actuated by joint kinematics measured during gait with the VSF. Joint angles 289 were strongly correlated, but exhibited a small phase lag, possibly due to ODE solver settings and numerical 290 integration. This phase lag may be also be present in the kinetic data, but masked by the larger inherent variability of 291 the simulated GRF_s. Total simulation times were 8.95 ± 3.92 , 12.7 ± 0.67 , and 46.2 ± 1.19 times slower than real 292 time for the low, medium, and high stiffness configurations respectively. Execution times were 3.12 ± 0.10 , $3.40 \pm$ 293 0.67, and 38.4 ± 1.19 times slower than real time. Increased execution times for the stiff conditions may reflect the 294 need for small time-steps in solving a rapidly-evolving, stiff differential equation.

295 Optimization of the GRF_{*} for the low stiffness configuration achieved a RMSE of 5.3% BW and R^a of 0.98. 296 These values are similar to those reported in previous biomechanical contact modeling work [28–30]. However, 297 those studies focused on quantification of foot-ground contact during gait for individuals with intact limbs. Direct 298 comparison of these data was limited to work in intact limb biomechanical modeling due to a lack of studies 299 reporting validation data for prosthesis-ground contact modeling in gait biomechanics. The strong correlation and 300 low error for GRF_{*} impulse indicates that the contact model is able to predict the shape and trajectory of the GRF_{*} 301 arising from gait kinematics. Accurate predictions of GRF₈ impulse is important for capturing whole-body energetics 302 throughout gait. The concomitant agreement for both kinematics and kinetics further suggests that these methods are 303 viable for simulating whole-body energetics during gait.

The transferability of the optimized contact model parameters from the low stiffness condition was assessed by simulating two additional low stiffness trials and three trials each with medium and high stiffness configurations. Compared to the optimized trial, simulation-derived GRF₈ predictions did not perform as well in the unoptimized trials. Mean GRF₈ RMSE and R² were $12.7 \pm 1.44\%$ BW and 0.91 ± 0.02 for the remaining low stiffness trials. These values were similar for the medium and low stiffness trials (Table 3). The impulse of these data matched well across the unoptimized trials (RMSE: 0.03 ± 0.02 BW·s, R²: 0.98 ± 0.01). Variability of the model's performance was similar across the unoptimized conditions for all outcome measures. It is possible that the contact model parameters were over-fitted to the specific conditions of a single trial, resulting in decreased
 generalizability. Future work should assess the balance between optimization specificity and generalizability.

The amplitude and shape of experimental GRF₈ waveforms were similar across the three stiffness conditions. However, stance phase times did vary by condition for the subject tested. The medium stiffness condition resulted in the longest stance phase time $(0.79 \pm 0.01 \text{ s})$, high stiffness resulted in the shortest $(0.71 \pm 0.02 \text{ s})$ s), and low stiffness $(0.73 \pm 0.02 \text{ s})$ was in the middle. The same pattern was present in the simulated data, although stance phase times were 0.05 ± 0.03 s longer on average compared to the experimental data. Stance phase times derived from simulations were correlated with experimental times (R² = 0.65). More data are necessary to discern the strength, repeatability, and significance of these relationships.

Simulated \hat{CoP}_{xv} values agreed well with experimental values. The RMSE values achieved using this model were similar to those reported in previous work involving subject-specific biomechanical contact modeling for individuals with intact limbs [31]. Accurate mapping of CoP_{xv} throughout stance phase is vital for simulating the effects of variable prosthesis stiffness on joint forces and moments during gait. Errors in model-predicted CoP_{xv} may be reduced by increasing the density of contact spheres distributed on the plantar surface of the foot, which would improve the resolution of CoP_{xv} predictions. However, this would likely result in increased execution time for simulations and also increase complexity of the contact parameter optimization problem.

327 The present data show promise for predicting GRF_{*} arising from a semi-active VSF prosthesis. These 328 methods may be applied to the design and prescription of lower limb prostheses and forward dynamics simulations 329 in robotics and biomechanics. Within biomechanics, future work could integrate the VSF model into a gait model of 330 an individual with lower limb loss. Gait simulations could be formulated as an optimal control problem in which 331 prosthesis stiffness is tuned to minimize a biomechanical cost function such as joint loading or metabolic cost. 332 Further optimization of the VSF-ground contact model may be necessary for simulation scenarios with error 333 tolerances less than 12% BW. Similar improvements may be required if the mean difference between simulation 334 conditions is less than the error of the model. Reducing error in model-predicted GRF₈ may be accomplished by 335 evaluating the objective function under a variety of conditions and choosing the parameter set that achieves the best 336 minimization across several conditions. A deformable contact model, such as presented in Jackson, Hass, and Fregly 337 (2016), may also be a viable means of representing foam deformation throughout stance phase and thus reducing 338 error.

339 These methods assume accurate estimation of segment length, joint centers, and joint angles which were 340 derived from marker-based motion capture data. Each of these metrics likely suffers from small errors due to marker 341 placement, localization, and coordinate system design. Such errors would contribute to decrements in contact model 342 performance. The components and joints of the prosthetic limb were also modeled as rigid, which may not be 343 completely accurate to represent the physical limb. This discrepancy would manifest as small differences in 344 kinematics and energy transfer between the components of the prosthetic limb. Nevertheless, simulated motions 345 were consistent with experimental data of subjects walking with the VSF and other previously reported data of 346 spatiotemporal gait patterns among persons with lower limb loss [33,34]. Another limitation is inherent to the 347 reduced order design of the lumped parameter VSF keel, which constrains keel motion to the sagittal plane. While 348 this design is computationally efficient compared to more robust finite element models, it fails to account for small 349 torsional keel motions that would be possible under ecological gait conditions with the physical VSF.

5. Conclusions

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353 The present study demonstrates that the ESR properties of a semi-active VSF can be modeled with high 354 fidelity. Foot-ground contact models were used to estimate GRF₈ with 5.3% BW error in an optimized gait trial, 355 which translated to mean errors of 13% for unoptimized trials. The contact models also predicted COP_{st} with mean 356 error of 9.3% foot length. This model performance may be sufficient for gait simulations among persons with lower 357 limb loss. Such simulations may be used to aid in the prosthesis design and prescription process in order to improve 358 user mobility. These methods may also be helpful to identify other important prosthesis design parameters, which 359 can be modified to optimize gait. Further contact model optimization and error reduction may be required for 360 simulation-based comparisons of varied prosthesis stiffness, where differences in GRF_{μ} magnitude may be nuanced. Statement of acknowledgements: The authors would like to thank Evan Glanzer for his work in developing and
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С	c	7
3	σ	1

Acronyms widely used	in text
BW	Body weight; M*g
CoP	Center of pressure
DOF	Degrees of Freedom
ESK	Energy storage and return
$\mathrm{GRF}_{\scriptscriptstyle{\mathbb{R}}}$	Resultant Ground Reaction Force, N; $\sqrt{GRF_x^2 + GRF_y^2 + GRF_z^2}$
LHS	Latin Hypercube Sampling
MTS	Material Testing System
ode15s	Ordinary differential equation 15 solver
SD	Standard Deviation
RMSE	Root Mean Square Error; $\sqrt{\frac{\sum_{i=1}^{N} (Experimental_i - Simulation_i)^2}{N}}$
VSF	Variable Stiffness Foot
Abbreviations	
a	Overhung length, mm
b	Damping coefficient, N·s/mm
В	Support fulcrum position, mm
D	Displacement, mm
F	Force, N
k	Linear stiffness, N/mm
	I otal beam length, mm
l	Supported length, mm
n \mathbf{P}_{2}	Coefficient of determination
N ²	Coefficient of friction
μ V	Linear velocity
V V	Scaling factor
δ	Penetration depth mm
ś	Penetration velocity, mm/s
ω	Angular velocity, rad/s
Supersorints and subs	
CoP	Anterior posterior (Center of Pressure)
	Simulation (Displacement)
D D	Experimental (Displacement)
E_{ep}	Frictional force N
F.	Normal force. N
GRF	Resultant ground reaction force, N
k _{sim}	Simulation (stiffness), N/mm
$k_{\scriptscriptstyle exp}$	Experimental (stiffness), N/mm
\mathcal{V}_{poc}	Linear velocity at point of contact, mm/s
$\mathcal{V}_{ ext{threshold}}$	Linear velocity threshold, m/s
μ_{kinetic}	Coefficient of kinetic friction
μ_{static}	Coefficient of static friction

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- 448



- 451 452 453 Figure 1: Overhung cantilever beam model of the VSF. The schematic illustrates keel length (*L*) pinned at A and simply supported at B, with a force applied at C. Overhung length (*a*) = L - l (supported length). Image reproduced with permission from Glanzer and Adamczyk (2018).

454



456 Figure 2: Modeled VSF, pylon, socket, and materials testing system (MTS). The MTS translates vertically, contacting the VSF
 457 30 mm proximal to the end of the keel (Glanzer and Adamczyk 2018).

458



461 Figure 3: Schematic of a single sphere-to-plane contact model (A) and contact model plantar (B) and lateral (C) perspectives of
 462 the VSF sphere-to-plane contact models. Heel contact spheres vary in color by zone.

463

464 Table 1: Summary of sphere-to-plane contact model parameters for the VSF.

			Penetration for full	
Location	<i>k</i> (N/mm)	b (N·s/mm)	damping (mm)	Penetration exponent
Zone 1	90.16	3.525	7.474	297.7
Zone 2	91.11	390.9	2.000	458.4
Zone 3	18.01	292.9	2.900	3.152
Zone 4	1003	252.1	0.765	0.977
Zone 5	123.8	476.7	1.700	0.754

465 466

k: stiffness, b: damping





468 Figure 4: Load-displacement relationships for simulation (dashed) and experimental data (solid). Data are best fit \pm 95% confidence interval. Displacement offset (Δ_D), example depicted with a bracket (I–I), is the difference between simulated and experimental mid-range displacement (eq. (3)).

72 Table 2: Comparative summary of experimental and simulated stiffness and mid-range displacement.

Fulcrum position (mm)	$k_{\scriptscriptstyle{ m exp}}$ (N/mm)	k (N/mm)	473 Displacement offset (mm) 474
66	10.43 ± 0.07	10.94 ± 0.00	^{0.02} 475
87	14.17 ± 0.08	13.62 ± 0.00	-0.46
108	19.45 ± 0.10	18.52 ± 0.00	0.23
129	24.83 ± 0.16	23.04 ± 0.00	0.32 477
151	31.59 ± 0.24	29.41 ± 0.00	0.79 478

479 **Displacement (D) offset:** $D_{sim} - D_{exp}$. Data are mean \pm SD.

Tables and figures







483

 $\begin{array}{ll} \textbf{484} & \mbox{Figure 6: Ensemble curves for GRF_{R} (top) and GRF_{R} Impulse (bottom) for the low, medium, and high stiffness conditions (left, middle, and right). \end{array}$

Tables and figures



487 Figure 7: Ensemble curves for COP_{ar} position for the low, medium, and high stiffness conditions (left, middle, and right).

Table 3: Summary of GRF_R, GRF_R impulse, and COP_{AP} comparison between simulated and experimental data.

Stiffness	G	GRF _R		GRF _* Impulse		COP
Configuration	\mathbf{R}^2	RMSE (BW)	\mathbf{R}^2	RMSE (BW·s)	\mathbf{R}^2	RMSE (% FL)
Low	0.93 ± 0.05	0.10 ± 0.04	$> 0.99 \pm 0.01$	0.02 ± 0.01	0.95 ± 0.01	8.93 ± 0.99
Medium	0.92 ± 0.01	0.13 ± 0.02	0.96 ± 0.02	0.05 ± 0.01	0.94 ± 0.01	9.45 ± 0.92
High	0.87 ± 0.07	0.14 ± 0.07	$>0.99\pm0.01$	0.02 ± 0.01	0.97 ± 0.01	5.68 ± 1.39

BW: Body weight, **COP**_{*x*}: Anterior-posterior center of pressure, **FL**: Foot length, Data are mean ± SD