



22 Although the structure of retractable roofs has a long history, it was not until recent  
23 decades that it gained widespread attention and became an essential category of  
24 building structures [1–4]. The various forms of retractable roofs not only enrich the  
25 urban landscape but also endow new functionalities to building structures.

26 The retractable roof structure is a type of deployable structure, and as such, its  
27 research is closely related to that of truss, plate, and membrane unit systems. The  
28 concept of deployable structures was first introduced by American architect  
29 Buckminster Fuller in 1960 [5,6]. Inspired by his work, Spanish architect E.P. Pineo  
30 proposed a scissor joint unit system that connected two bar members through a central  
31 pin joint. This system was first applied to the design of a deployable theater, marking  
32 the first use of deployable structures in such a context [7]. Hoberman, based on the  
33 scissor joint unit, further developed a unique deployable unit system, which served as  
34 the basis for designing a new kind of centrally-symmetric radial retractable roof  
35 structure [8].

36 British scholars You and Pellegrino, building on the research of Hoberman, further  
37 expanded the research by developing a new type of fundamental scissor joint unit and  
38 constructed a more general deployable structure known as Foldable Bar structure, FBS  
39 [9]. Additionally, utilizing the scissor joint unit as the fundamental element, You and  
40 Pellegrino, at the same time, proposed a radial retractable structure system [10].

41 In practice, structural performances, particularly mechanical properties, play a  
42 crucial role in the design and analysis of retractable roof structures. Chen and his

43 colleagues [11] conducted a study on the structural performance of deployable planar  
44 truss structures and found that the low stiffness of the structure may lead to reduced  
45 efficiency in large-span truss structures (due to the significant bending moments to  
46 internal forces). To overcome this issue, they proposed adding diagonal bracing and  
47 lower chords that are compatible with the original geometry of the scissor joint units,  
48 thereby improving the mechanical performance of the structure [11]. Professor Cai [12]  
49 has extensively researched various new deployable structures, including angular scissor  
50 joint units, folding truss structures, and tensioned cable structures, utilizing  
51 theoretical analysis, nonlinear finite element simulations, and physical model  
52 experiments. His research focused on the geometric composition, motion process, and  
53 mechanical performance of these structures, resulting in a set of highly practical  
54 conclusions. In short, a substantial amount of work [3,13,14] has been done on the  
55 development, theoretical analysis, and optimization of deployable structures.  
56 Nevertheless, there is still a need for further integration and application of these  
57 deployable structures in specific engineering areas (for instance, civil engineering) , to  
58 fully realize their potential.

59 Origami has been widely applied in engineering, particularly in deployable  
60 structures, due to its diverse configurations. Mechanisms based on the principles of  
61 origami are an excellent choice for structures that undergo significant changes in  
62 characteristics, such as area, in different states. Origami has been used as a tool for  
63 mathematical and scientific research since the end of the 19th century. In the 1990s,

64 research on the mathematics and theory of origami gained popularity in many countries  
65 worldwide. De Focatiis et al. [15] used the folding model of the oak leaf as a basis to  
66 propose various high-compactness folding methods by connecting several leaf units in  
67 different ways, which provided new avenues for the deployable membrane structure.  
68 Origami structures have found critical applications in the aerospace field. For instance,  
69 a solar panel based on Flasher origami [16], composed of a series of triangles,  
70 rectangles, and trapezoids arranged according to specific rules, was developed by the  
71 Jet Propulsion Laboratory of NASA, and can be readily deployed into a hexagonal  
72 shape. The folding and unfolding ratios of this new structure satisfy the requirement of  
73 the aerospace industry for solar panels in both the un- and folded states. Utilizing the  
74 unique morphological adaptability of origami, Professor Miura Koryo [17,18], from the  
75 Institute of Space and Astronautical Science at the University of Tokyo, invented an  
76 origami technique called Miura-ori, which not only saves space but also minimizes  
77 losses during the un- and folding processes (reducing the volume of an object by 25  
78 times and increasing its energy density by 14 times). Currently, researchers are  
79 exploring the potential of origami techniques in the field of architecture and civil  
80 engineering [19–21]. Nevertheless, the application of origami structures in this area,  
81 overall, is still relatively limited and primarily focused on temporary structures such as  
82 lightweight exhibition halls, galleries, and tents. Expanding the application of origami  
83 structures in civil engineering, particularly in large and permanent building structures  
84 such as sports stadiums, is a research direction that requires further exploration and

85 development.

86 We, in this paper, propose a novel retractable roof structure based on origami and  
87 conduct an in-depth investigation into its kinematic mechanism, aiming to provide more  
88 design options for the first party and foster innovation in building structural design by  
89 enabling novel functional and structural possibilities for buildings. We first introduce  
90 the geometric principles underlying the Miura-ori, including its arched and fan-shaped  
91 variations, and provide an overview of the concepts of rigid origami and the thick plate  
92 analysis ([Section 2](#)). Further, we utilize scissor joint rectangular units to develop three  
93 novel retractable roof structures, derive the compatibility conditions for the coordinated  
94 motion of each new retractable roof structure, and propose a corresponding  
95 improvement strategy ([Section 3](#)). After realizing the deployment of retractable roof  
96 structures, a detailed analysis of the structural changes during the motion process is  
97 conducted ([Section 4](#)). Finally, the conclusion is summarized, while the limitations of  
98 the study and the prospects for future research are also described ([Section 5](#)).

## 99 **2. Retractable roof based on the principle of origami mechanism**

100 This section aims to identify suitable origami structures that can be applied to  
101 retractable roof structures. Firstly, we elaborate on the fundamental kinematic  
102 characteristics of the classic Miura-ori, as well as its arched and fan-shaped variations.  
103 Additionally, a detailed introduction is given to the expandable scissor joint unit and its  
104 variants. Subsequently, a comprehensive analysis of the thick-plate transformation for  
105 the retractable roof is presented. Finally, the parameterization modeling method used in

106 this paper is explained using the classic Miura-ori structure as an example.

## 107 **2.1 Miura-ori and its variations**

108 The Miura-ori is a classic single-degree-of-freedom origami structure with identical  
109 parallelogram-shaped units [17]. The classic Miura-ori, when fully expanded, features  
110 longitudinal zigzag creases. Based on the classic Miura-ori, adjusting the longitudinal  
111 angle of the zigzag creases to make adjacent longitudinal creases unparallel (with a  
112 reasonable angle) while spaced longitudinal creases remain parallel, to get the arched  
113 variant of the Miura-ori [22–24]. If the creases that lie along straight lines in the  
114 transverse direction are altered to become non-parallel, a different form of Miura-ori,  
115 known as the fan-shaped variant, can be achieved [25].

116 The uniformity of the basic components in the Miura-ori enables us to derive the  
117 kinematic behavior of the entire mechanism by just analyzing a single unit. Utilizing  
118 the principles of Babuška-Brezzi's spherical trigonometry as well as the constraints  
119 governing the mechanism [26–28], more generalized kinematic patterns can be deduced.  
120 This approach is further applied to derive the corresponding kinematic patterns for the  
121 arched and fan-shaped variants of the Miura-ori.

## 122 **2.2 Expandable Scissor Joint Unit**

123 We introduce expandable scissor joint units to enhance the strength of the structure  
124 and to achieve more convenient control [29,30]. Several scissor joint units, consisting  
125 of two bars of equal length hinged together at the center, are connected by hinges at the  
126 ends of the bars, which form a rectangular mechanism with a single degree of freedom.

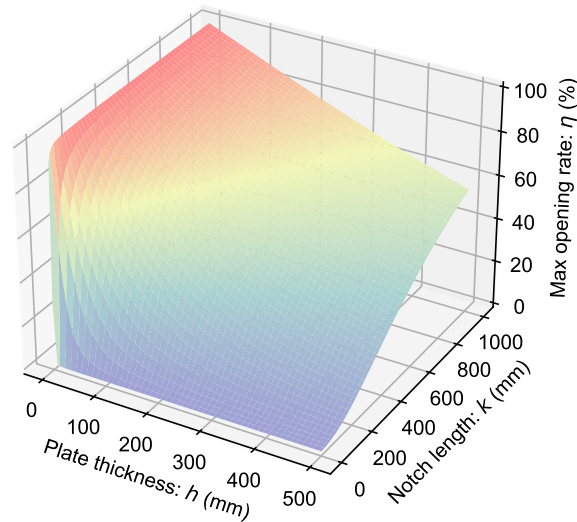
127 Furthermore, by altering the hinge positions in a single scissor joint unit and satisfying  
128 certain conditions, an arched scissor joint unit can also be formed [30]. A novel  
129 retractable roof structure can be obtained by combining scissor joint units with  
130 deployable grid structures.

### 131 **2.3 Thick plate retractable roof structures analysis**

132 Rigid origami maintains its shape as a rigid plane throughout the un- and folding  
133 process, making it particularly suitable for origami-derived structures with significant  
134 thickness [31–33]. We draw inspiration from origami and thick plate theories, and  
135 utilize the principles of origami mechanisms to design retractable roof panels. Various  
136 thick plate methods, including adjusted the crease position, adjusted the shape, and  
137 expanding the gap, etc., have been proposed by different researchers for planar  
138 symmetric quadrilateral origami structures [31,32,34–36]. The adjusted shape method  
139 is applied in our study to obtain the corresponding thick plates for the retractable roof  
140 panels. The calculation formula for the opening ratio of the retractable roof panels is  
141 given by Eq. (1) (for a detailed derivation, please refer to [Appendix A](#)):

$$142 \quad \eta = 1 - \sin\left(\arctan \frac{h}{k}\right) \quad (1)$$

143 where  $\eta$  denotes the opening rate,  $h$  represents the thickness of the panels being trimmed  
144 on both sides of the crease, and  $k$  is the length of one side of the symmetrical notches.  
145 Furthermore, by visualizing Eq. (1), the variation spectrum of the opening rate can be  
146 obtained, as depicted in Fig. 1, which vividly demonstrates the relationship between the  
147 variables of plate thickness and notch length with max opening rate.



148

149

Fig. 1 The variation spectrum of the maximal opening rate.

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As shown in Fig. 1, when the notch length is fixed and its value is relatively small,

151

the opening rate decreases with the increase in panel thickness, and the reduction

152

gradually weakens. Meanwhile, when the notch length is fixed and its value is relatively

153

large, the opening rate decreases in a nearly linear manner. Eq. (1), essentially, indicates

154

that the maximum opening ratio is solely determined by the trimmed thickness  $h$  and

155

the notch length  $k$ , and is irrelevant to factors such as the shape and size of the panel.

156

Therefore, the calculation of the opening rate, for the arched and fan-shaped retractable

157

roof structures, is the same as Eq.(1).

158

#### 2.4 Parametric modeling of retractable roof structure

159

Parametric modeling steps using Grasshopper, a powerful parametric modeling

160

software widely used in building structure areas, can be briefly summarized as follows:

161

First, establish a zero-thickness Miura-ori, without considering factors such as the

162

thickness of the panels and the composition of the bars, which is used to determine the

163

shape of the roof unit at different stages of the retractable roof. Then, based on the zero-



164 thickness Miura-ori mechanism, construct a grid structure with thickness. Thereafter, a  
165 practical and applicable retractable roof structure is established for specific analysis. A  
166 comprehensive guide to the parameterized modeling process can be found in [Appendix](#)  
167 [B](#).

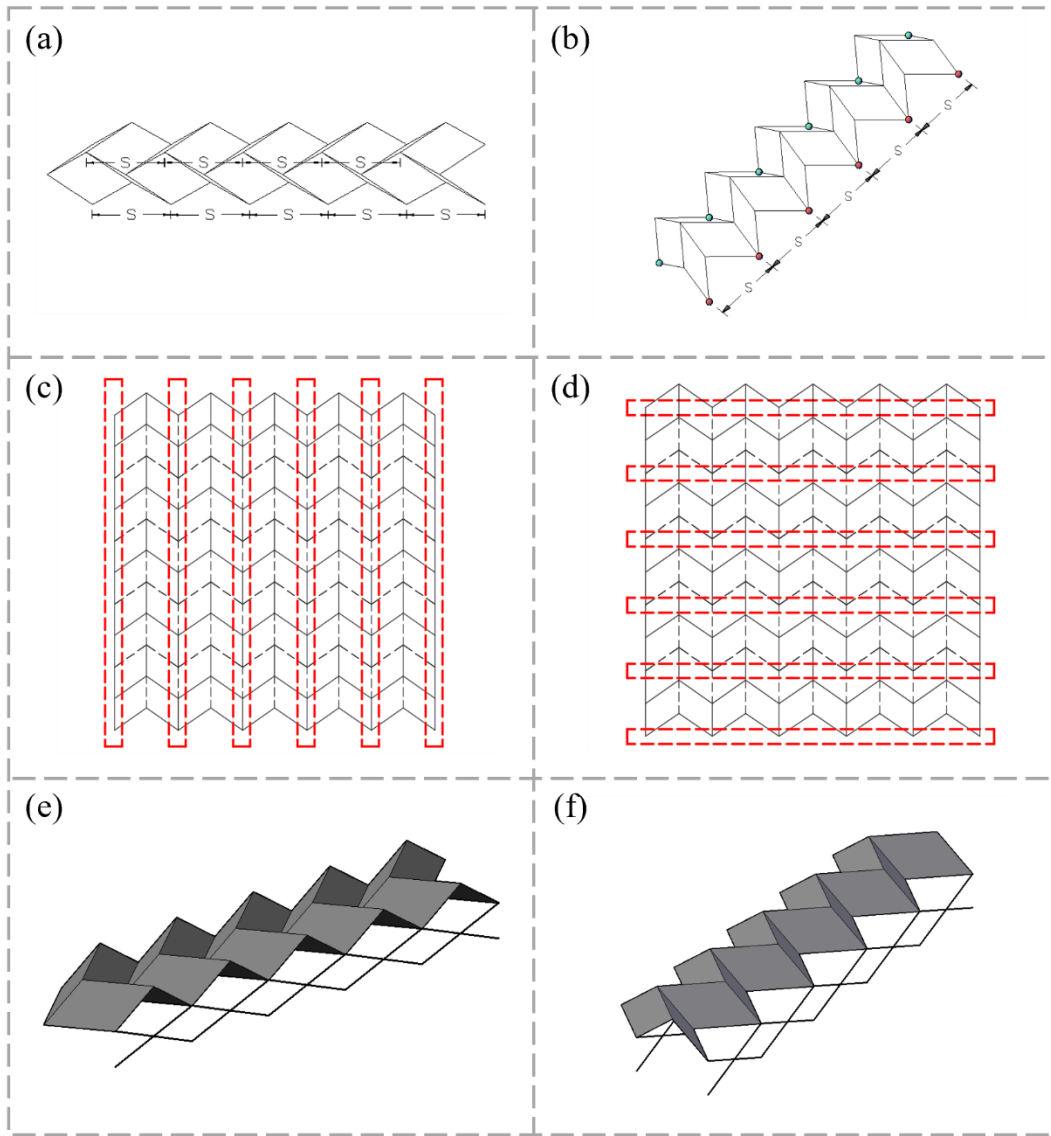
### 168 **3. Layout of retractable roof structures**

169 This section further elaborates on the different deployments and forms of the  
170 retractable roofs based on the principle of Miura-ori (as well as its arched and fan-  
171 shaped variants). First, the deployment of scissor joint units is based on the co-linear  
172 and coplanar points that are distributed in a relatively regular manner during the un-  
173 and folding process. Geometric formulas are then further employed to validate whether  
174 the upper and lower parts of the structure can work synchronously and coordinately.  
175 Ultimately, the corresponding optimal methods are proposed for cases where  
176 coordinated motion can not be achieved.

#### 177 **3.1 Miura-ori retractable roof**

178 When the Miura-ori mechanism is fully extended, all the mountain and valley  
179 vertices of the transverse (span direction, parallel to the x-axis) creases are collinear  
180 and located in the same plane (Fig. 2c) . The distances between the mountain and valley  
181 vertices in the transverse direction keep unchanged during the deploying process for  
182 they share the same basic unit (Fig. 2a). By placing the scissor-jointed rectangular  
183 mechanism on the transverse direction of the Miura-ori (the red outline section in Fig.  
184 2c) where the ends of the upper sides of the rectangular mechanism are hinged with the

185 valley vertices of the Miura-ori (Fig. 2e), the motion of these two parts are coordinated  
 186 in the transverse direction.



187

188 Fig. 2 Miura-ori retractable roof. (a) and (b) denote the transverse and longitudinal  
 189 origami units, respectively; (c) and (d) depict the location of the transverse and  
 190 longitudinal scissor joint units; (e) and (f) are the axonometric projection in the  
 191 transverse and longitudinal direction of scissor joint units.

192 Similarly, all the vertices of the longitudinal (parallel to the y-axis) creases are  
 193 collinear throughout the deploying process, and the distances between them remain

194 equal (the distance between the blue dots and the distance between the red dots in Fig.  
195 2b). Additionally, the longitudinally arranged creases, which are spaced apart, remain  
196 parallel to each other, and maintain in the same plane during the folding process. By  
197 placing the scissor-jointed rectangular mechanism at the inflection point of the  
198 longitudinal creases of the Miura-ori, as shown in Fig. 2d, where the upper ends of both  
199 sides of the rectangular mechanism are hinged with the vertices of the inflection points  
200 of the creases of the Miura-ori (Fig. 2f), the motion of these two parts are coordinated  
201 in the longitudinal direction.

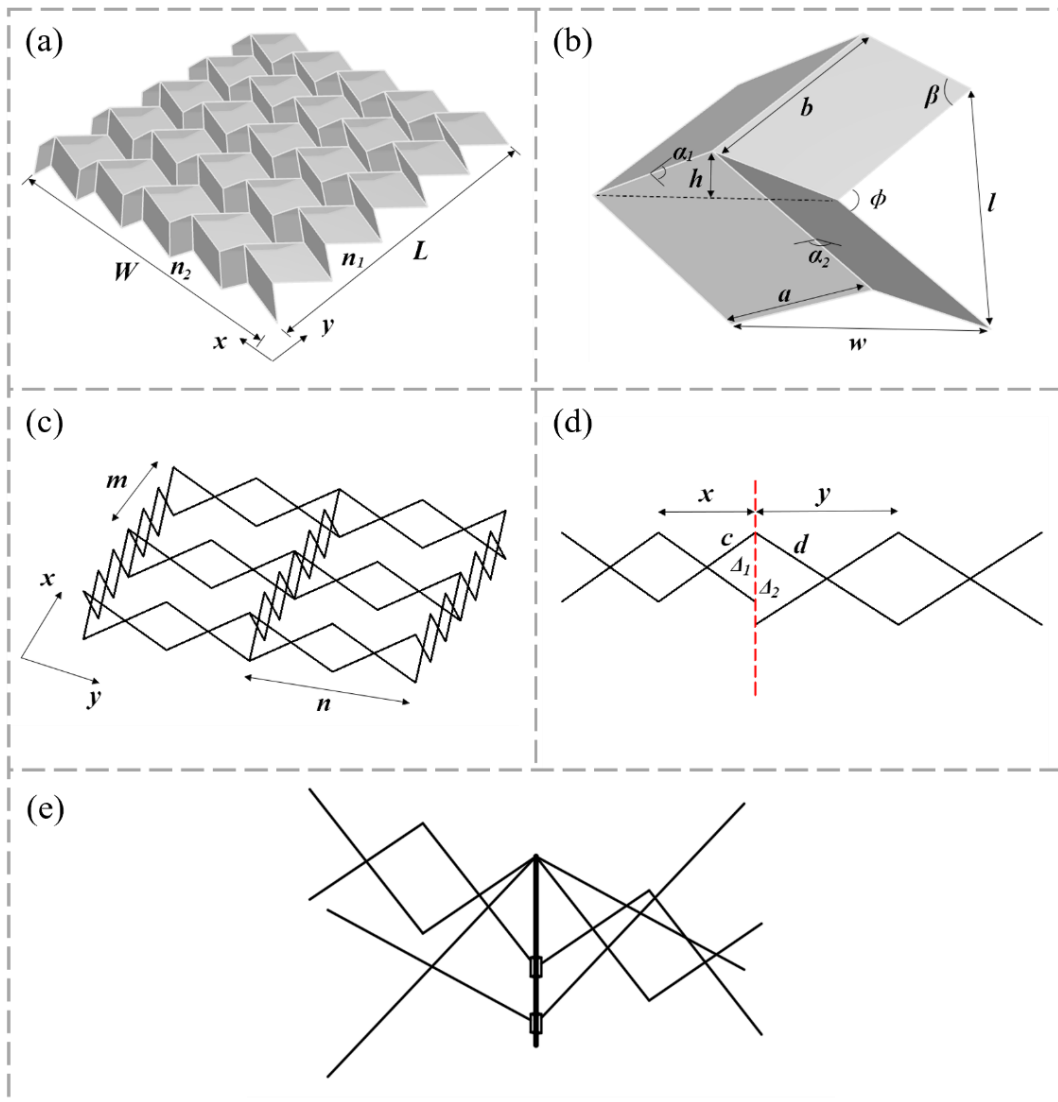
202 The two conditions for the coordinated motion of the scissor-jointed rectangular  
203 mechanisms in both x- and y-direction are as follows:

204 (1) The ratio between the distance of the upper ends of the rectangular mechanism  
205 in the x- and y-direction remains constant and is equivalent to the ratio between the  
206 distance of the crease vertices of the Miura-ori mechanism in both the x- and y-direction.

207 (2) The ratio of the relative velocity between the upper ends of the rectangular  
208 mechanism in the x- and y-direction is equivalent to the ratio of the relative velocity  
209 between the crease vertices of the Miura-ori mechanism in both the x- and y-direction,  
210 and it remains constant.

211 As mentioned above, the distance between the ends of the scissor joint units in both  
212 x- and y-direction is always equal to the distance between the corresponding crease  
213 vertices. Therefore, the ratio of the distances of the former is equal to the ratio of the  
214 distances of the latter, which satisfies condition (1). Nevertheless, for condition (2),

215 further discussion is needed.



216

217 Fig. 3 Geometric parameters of the Miura-ori retractable roof. (a) The overall layout  
 218 of Miura-ori; (b) Axonometric projection of a panel unit; (c) The overall layout of  
 219 scissor joint units; (d) Simplified diagram of two scissor joint units; (e) The schematic  
 220 diagram of the improved scissor joint units. ( $W$  and  $L$  represent the length along the x-  
 221 and y-direction, respectively.  $n_1$  and  $n_2$  represent the number of units in the two  
 222 directions.)

223 All the critical geometric parameters of the retractable roof structure, formed by the

224 classic Miura-ori, are diagrammed holistically in Fig. 3. Once the side lengths ( $a$ ,  $b$ )  
 225 and acute angle ( $\beta$ ) of a sub-parallelogram panel unit (Fig. 3b), with four other identical  
 226 sub-parallelograms, are given, the shape of the basic unit can be determined. The  
 227 motion of the entire unit is controlled by  $\phi$ , the angle between the two longitudinal  
 228 creases. According to [37], the length ( $l$ ), width ( $w$ ), and height ( $h$ ) of the basic unit can  
 229 be expressed as follows:

$$230 \quad l = 2b \sin(\phi / 2) \quad (2)$$

$$231 \quad w = 2a \frac{\cos \beta}{\cos(\phi / 2)} \quad (3)$$

$$232 \quad h = \frac{a \sqrt{\sin^2 \beta - \sin^2(\phi / 2)}}{\cos(\phi / 2)} \quad (4)$$

233 The angles formed between the panels of the parallelogram, denoted  $\alpha_1$  and  $\alpha_2$ , can  
 234 be represented as follows:

$$235 \quad \alpha_1 = \cos^{-1} \left[ 1 - 2 \frac{\sin^2(\phi / 2)}{\sin^2 \beta} \right] \quad (5)$$

$$236 \quad \alpha_2 = \cos^{-1} \left[ 1 - 2 \cot^2 \beta \tan^2(\phi / 2) \right] \quad (6)$$

237 Fig. 3c-d depicts the three-dimensional scissor joint units mechanism and its  
 238 essential geometric parameters. To form the whole structure with a single degree of  
 239 freedom, the scissor joint units require the length of the upper and lower portions of the  
 240 two scissor joint units at the red dashed line to be equal (Fig. 3d), namely  $\Delta_1 = \Delta_2$ , which  
 241 can be expressed as:

$$242 \quad 2\sqrt{c^2 - \left(\frac{x}{2}\right)^2} = 2\sqrt{d^2 - \left(\frac{y}{2}\right)^2} \quad (7)$$

243 where  $c$  and  $d$  are half of the length of the scissor joint unit in the x- and y-direction,  $x$

244 and  $y$  denote the distance between the upper ends of the scissor joint units in the  $x$ - and  
 245  $y$ -direction, respectively. We, further, formulate the values of  $x$  and  $y$  using Eqs. (8) and  
 246 (9):

$$247 \quad x = \frac{w}{m} = \frac{2a}{m} \cdot \frac{\cos \beta}{\cos(\phi / 2)} \quad (8)$$

$$248 \quad y = \frac{l}{n} = \frac{2b}{n} \cdot \sin(\phi / 2) \quad (9)$$

249 where  $m$  and  $n$  respectively denote the number of scissor joint units for a single Miura-  
 250 ori in the  $x$ - and  $y$ -direction. Substituting the above two equations into Eq. (7) then, we  
 251 can obtain:

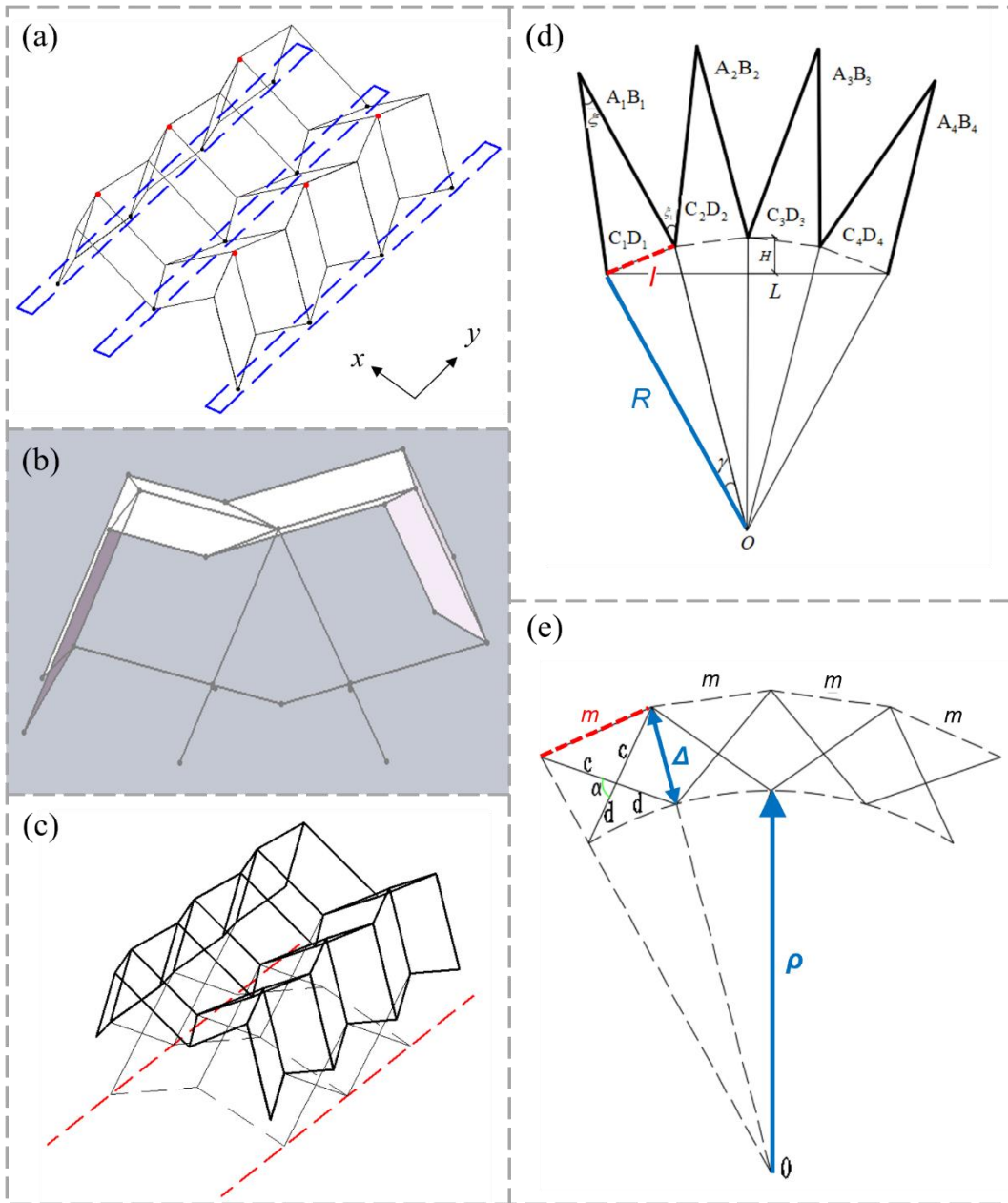
$$252 \quad \left(\frac{a^2}{m^2} \cos^2 \beta\right) \cdot \frac{1}{\cos^2(\phi / 2)} - \frac{b^2}{n^2} \cdot \sin^2(\phi / 2) = c^2 - d^2 \quad (10)$$

253 The parameters  $\beta$ ,  $a$ ,  $b$ ,  $m$ , and  $n$  above are constants, and the un- and fold of the  
 254 entire structure is controlled by the parameter  $\phi$ . Since the left side of the equation is a  
 255 variable and the right side is a constant, it is not feasible for the three-dimensional  
 256 scissor joint units connected to the Miura-ori to create a structure with only one degree  
 257 of freedom. Therefore, an improved design is needed. Fig. 3e is a schematic diagram of  
 258 the improved scissor joint units. A vertical bar is added to the top of the original scissor  
 259 joint units to connect with the crease vertex of the Miura-ori. The scissor joint units in  
 260 both directions are connected at the top, and the bottom ends of the bars are connected  
 261 to the vertical bar through two sliding joints. In this way, free deploying in the  $x$ - and  
 262  $y$ -direction can be achieved readily, and it is coordinated with the movement of the  
 263 Miura-ori mechanism at the top.

### 264 3.2 Arched variant of retractable roof

265 In the un- and folding process of the arched variant (Fig. 4a) , the crease vertices  
266 at the longitudinal inflection points (parallel to the y-axis) are always collinear. Since  
267 the basic units are identical, the distance between the crease vertices in the longitudinal  
268 direction (the distance between the blue dots) remains unchanged during the opening  
269 and closing process. When arranging the scissor-jointed rectangular mechanism at the  
270 longitudinal crease inflection point of the arched variant (the blue outline section in  
271 Fig. 4a) , the upper ends of the scissor-jointed units on both sides of the rectangular  
272 mechanism are hinged with the crease vertices at the inflection points of the arched  
273 variant. Taking only the longitudinal direction, the motion of the upper and lower  
274 mechanism is coordinated.

275 During un- and folding, likewise, the crease vertices along the x-axis are coplanar  
276 and the mountain and valley crease vertices in the same plane are located on a single  
277 arc segment (red points in Fig. 4a), which creates a uniform distance between the  
278 mountain and valley crease vertices in the transverse direction due to the consistent  
279 basic units.



280

281 Fig. 4 The arched variant of the retractable roof. (a) The longitudinal distribution of  
 282 arched variant origami and scissor joint units distribution; (b) The layout of the  
 283 transverse arched scissor joint unit; (c) The schematic diagram of the improved lower  
 284 mechanism; (d) Geometric of the upper origami; (e) Geometric parameters of the  
 285 lower arched scissor joint mechanism.

286 On the arched variant of the Miura-ori (Fig. 4b), the arched scissor joint units are



287 arranged in a transverse orientation, with the upper portion of the scissor-jointed  
 288 mechanism hinged to the crease vertices of the arched variant panel. To ensure proper  
 289 coordination between the scissor-jointed arched mechanism and the upper arched  
 290 variant of the Miura-ori, two conditions below must be met:

291 (1) The distance between the ends of the upper bars is equal to the distance between  
 292 the crease intersections of the upper arched variant of the Miura-ori.

293 (2) The arc radius between the arched scissor-jointed mechanism and the upper  
 294 arched variant should maintain consistency throughout the un- and folding process.

295 Condition (1) necessitates that the distance between the crease intersections at the  
 296 lower end of the arched variant (denoted as  $l$ ) must be equal to the distance between the  
 297 upper bar ends of the arched scissor-jointed mechanism (represented by  $m$ ), as  
 298 expressed by Eq. (11) and (12), respectively.

$$299 \quad l = 2a \cdot \sin \frac{\xi}{2} \quad (11)$$

$$300 \quad m = 2c \sin \left( \frac{\pi}{2} - \frac{\alpha}{2} \right) = 2c \cos \frac{\alpha}{2} \quad (12)$$

301 In Eq. (11),  $\xi$  represents the apex angle of the arched variant (Fig. 4d), while in Eq.  
 302 (12),  $\alpha$  denotes the angle formed between the two bars of the arched scissor-jointed  
 303 mechanism (Fig. 4e). Condition (1) can be met when  $\alpha$  and  $\xi$  satisfy Eq. (13):

$$304 \quad \sin \frac{\xi}{2} = \cos \frac{\alpha}{2} \quad (13)$$

305 Condition (2) requires that  $R = \rho + \Delta$ , where  $R$  represents the arc radius of the top  
 306 arched variant of Miura-ori, and  $\rho + \Delta$  represents the radius of the upper arc of the arched  
 307 scissor-jointed mechanism (Fig. 4e). It can be derived from [38] that:

$$308 \quad \frac{a \cdot \sin \frac{\xi}{2}}{\sin \frac{\xi_1 - \xi}{4}} = \frac{\Delta}{r-1} + \Delta = \frac{\sqrt{c^2 + d^2 - 2cd \cos \alpha}}{\frac{c}{d} - 1} + \sqrt{c^2 + d^2 - 2cd \cos \alpha} \quad (14)$$

309 In Eq. (14), introducing a new variable  $K$  as shown in Eq. (15), then substituting  $K$   
 310 into Eq. (14) and conducting simplification, we get Eq. (16).

$$311 \quad K = \frac{\frac{c}{d}}{a \cdot (\frac{c}{d} - 1)} = \frac{c}{ac - ad} \quad (15)$$

$$312 \quad \frac{\sin^2 \frac{\xi}{2}}{\sin^2 (\frac{\xi_1 - \xi}{4})} + 2cd \cos \alpha = K^2 \cdot (c^2 + d^2) \quad (16)$$

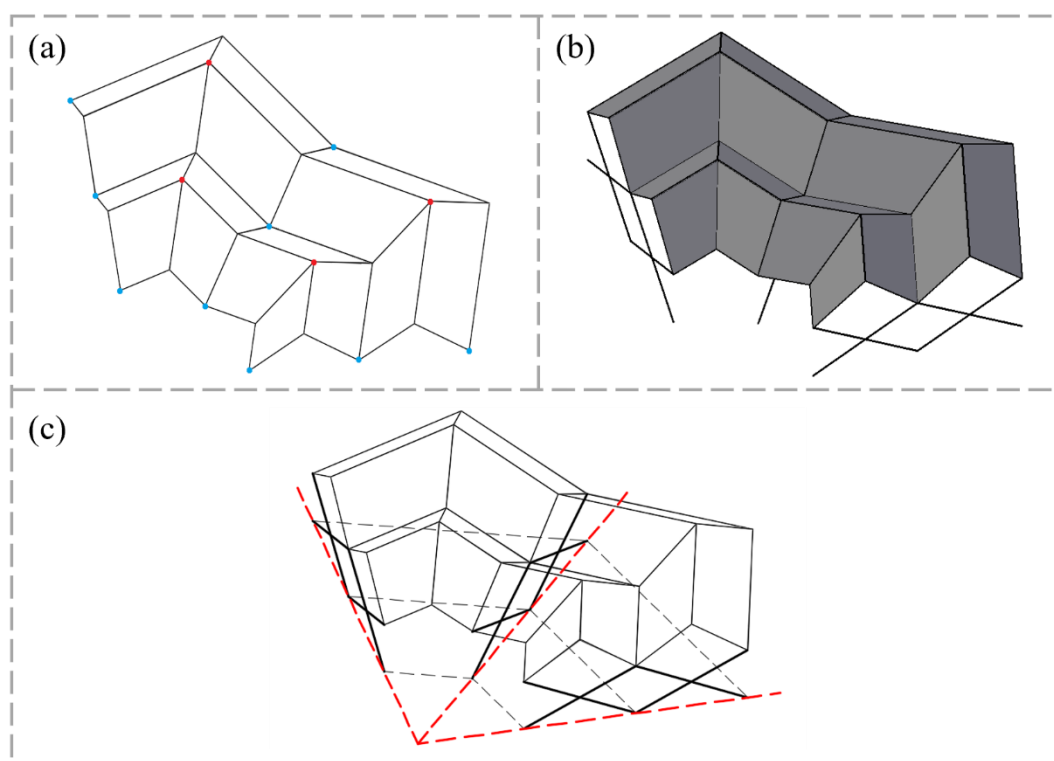
313 Eq. (16) features the constants  $a$ ,  $c$ , and  $d$ , making  $K$  a constant. The un- and folding  
 314 of the entire mechanism are regulated by  $\xi$ ,  $\xi_1$ , and  $\alpha$ . The right side of the equation is  
 315 a constant, while the left side is indeterminate, which suggests that the three-  
 316 dimensional scissor-jointed units, linked to the arched variant of the Miura-ori, cannot  
 317 create a mechanism with a single degree of freedom.

318 We present an enhanced design for the arched variant of the Miura-ori, depicted in  
 319 Fig. 4c, in which the longitudinal arrangement remains unaltered, but the arched  
 320 scissor-jointed units in the transverse direction are substituted with prestressed steel  
 321 cables (or telescopic bars). This modification facilitates unrestricted un- and folding in  
 322 both the x- and y-direction, while guaranteeing the coordinated motion of the lower  
 323 arched scissor-jointed mechanism and the upper arched variant of the Miura-ori.

### 324 3.3 Fan-shaped variant of retractable roof

325 During the un- and folding of the fan-shaped variant mechanism, the crease vertices

326 in the radial direction remain coplanar, while within the same plane, all mountain crease  
327 vertices and valley crease vertices are collinear. The geometric properties of the fan-  
328 shaped variant in the transverse direction make it convenient to arrange the scissor-  
329 jointed mechanism. The distances between the mountain crease vertices (represented in  
330 red) and valley crease vertices (shown in blue) in the radial direction remain constant  
331 during the folding process, by virtue of the identical basic units (Fig. 5a). As a result,  
332 arched scissor joint units similar to those used in the longitudinal direction can be  
333 utilized in the radial direction, allowing for the hinging and assembly of the upper  
334 structure (Fig. 5b).



335  
336 Fig. 5 The fan-shaped variant of the retractable roof. (a) The layout of the origami; (b)  
337 The arrangement of the scissor joint units; (c) The schematic diagram of the improved  
338 fan-shaped variation.

339 Similarly, additional prestressed steel cables or rotatable tracks (Fig. 5c) can be  
340 added between the lower bar ends of adjacent scissor-jointed rectangular units in the  
341 radial direction, which not only ensures coordination with the upper fan-shaped variant  
342 mechanism, but also helps to improve the overall integrity and stability of the structure.

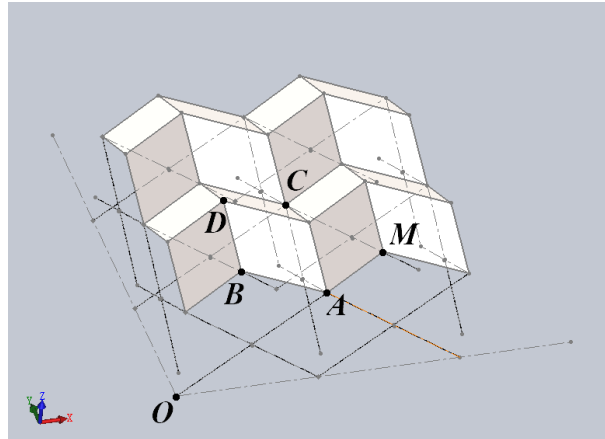
#### 343 **4. Motion simulation of retractable roof based on origami principles**

344 In this section, the structure models of the retractable roof structures based on  
345 origami principles are simplified, and then SolidWorks, a three-dimensional simulation  
346 software, is adopted to simulate its motion. After verifying that the roof panel with  
347 thickness can un- and fold smoothly, the variation law of the structure during its motion  
348 is analyzed. The kinematic analysis of the three new retractable forms is investigated  
349 exhaustively.

##### 350 **4.1 Miura-ori retractable roof**

351 We assume that, as shown in Fig. 6, the Miura-ori retractable roof model consists  
352 of four basic units, with two units longitudinally and transversely each. The length of  
353 each bar in the lower scissor joint unit is  $2c = 3100$  mm and  $2d = 3570$  mm.

354 Considering the thickness of the upper roof panel and the dimensions of the lower  
355 scissor hinge on the structural motion, two angles, namely  $\phi_1 = 20^\circ$  and  $\phi_2 = 120^\circ$ ,  
356 which correspond to the fully folded and unfolded states, respectively, are adopted to  
357 prevent interference phenomena. Throughout the entire opening and closing process,  
358 the angular variation between the roof panels totals  $100^\circ$ . Detailed parameters of the  
359 model are presented in Table 1.



360

361 Fig. 6 Miura-ori retractable roof model. A, B, C, and D denote reference points; point

362 O represents the motion center of the structure; point M denotes the place where the

363 rotational driving motor is set.

364 Table 1 The detailed parameters of Miura-ori retractable roof

$a$ (mm)	$b$ (mm)	$\beta$ (°)	$\phi_1$ (°)	$\phi_2$ (°)	$c$ (mm)	$d$ (mm)
1500	1732	60	20	120	1550	1785

365 Note: the parameters are illustrated in Fig. 3.

366 To simulate the motion of adjacent roof panels rotating 100 degrees using

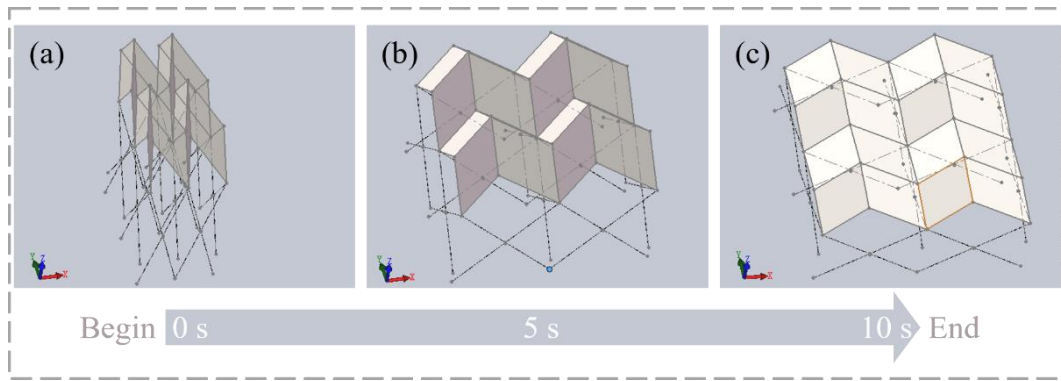
367 SolidWorks, we incorporated a rotational driving motor at point M, the intersection of

368 the bottom edges of the two panels, as illustrated in Fig. 6. The angle between the panels

369 increases uniformly at a constant angular velocity of 10 °/s for 10 seconds. Our

370 simulation, depicted in Fig. 7, demonstrates that the whole mechanism can successfully

371 un- and fold as an entirety.

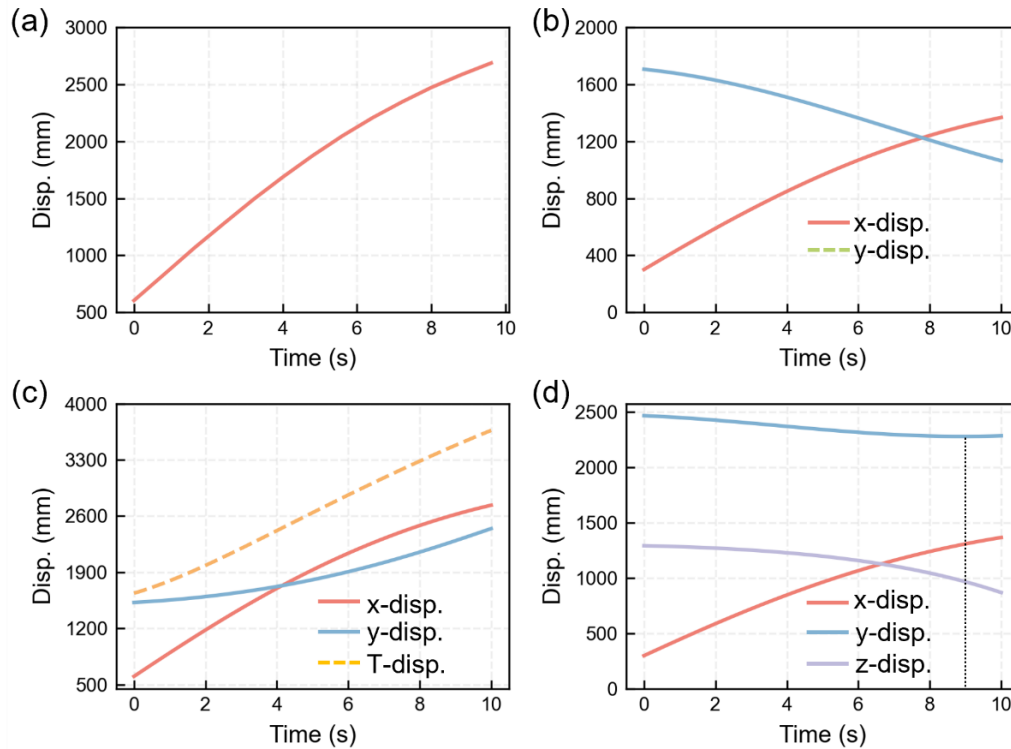


372

373 Fig. 7 The motion of the Miura-ori retractable roof. (a), (b), and (c) respectively

374 denote the 0th, 5th, and 10th seconds during the course of motion.

375 The displacement-time curves of reference points A, B, C, and D relative to the  
 376 center point O of the orbit are illustrated in Fig. 8. Upon comparative analysis, it can be  
 377 observed that the displacement of points B and D in x-direction is equivalent, as is the  
 378 displacement of points A and C in the same direction, which suggests that during the  
 379 process of un- and folding the roof, these two sets of points always remain coplanar  
 380 with the scissor joint units, ensuring coordinated motion of the structure. As the entire  
 381 structure unfolds, the displacement of all reference points in x-direction exhibits a  
 382 smooth and gradual increase, reaching its peak value at  $t = 10$  s. The panel comprising  
 383 points O, B, and D rotates around the x-axis and y-axis, resulting in a relatively minor  
 384 overall displacement in the y-direction for point D, which is only 182.33 mm. Moreover,  
 385 the displacement in z-direction of point D showcases the gradual decrease in the overall  
 386 height of the structure as it unfolds.



387

388 Fig. 8 The displacement-time curves of the Miura-ori retractable roof. (a), (b), (c), and

389 (d) denote the displacement-time relationship of points A, B, C, and D, respectively.

390 (x-, y-, and z-disp. represent the displacement components of a reference point in the

391 three principal axis directions while T-disp. is the total displacement of the

392 corresponding point.)

393 Fig. 9 depicts the velocity-displacement curves of points A, B, C, and D. The upper

394 roof comprises four identical parallelograms, resulting in comparable velocities of

395 points A and C, as well as B and D, at the onset of the motion. Specifically, the velocity

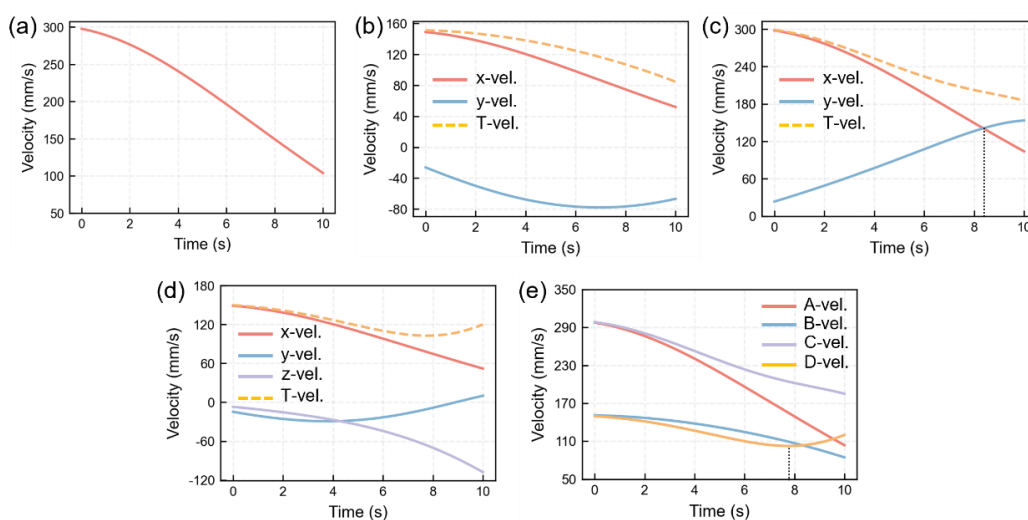
396 of points A and B is roughly twice that of C and D (more vividly demonstrated in Fig.

397 9e). For a significant period, the velocity of point C in x-direction consistently exceeds

398 that in y-direction, although the disparity between the two gradually diminishes. At  $t =$

399 8.36 s, the x- and y-direction of the velocity at point C are both 140.80 mm/s.

400 Subsequently, the y-direction of the velocity at point C overtakes the x-direction. This  
 401 indicates that, during the motion process, the structure initially moves predominantly  
 402 in x-direction and gradually shifts towards movement in y-direction. The velocity of  
 403 point D, as depicted in Fig. 9e, gradually decreases over time, reaching a minimum  
 404 value of 102.68 mm/s at  $t = 7.76$  s. Afterward, it gradually increases and eventually  
 405 surpasses the velocities of points A and B.



406

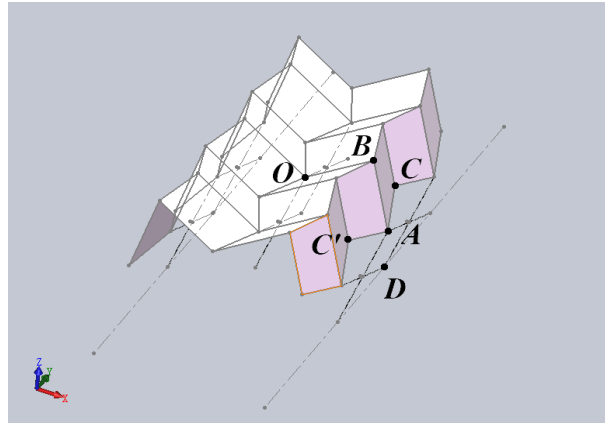
407 Fig. 9 The velocity-time curves of the Miura-ori retractable roof. (a), (b), (c), and (d)  
 408 denote the velocity-time relationship of points A, B, C, and D, respectively; (e)  
 409 depicts the velocity comparison of four reference points. (x-, y-, and z-vel. represent  
 410 the velocity components of a reference point in the three principal axis directions  
 411 while T-vel. is the total velocity of the corresponding point.)

#### 412 4.2 The arched variation of retractable roof

413 The arched variant retractable roof model, as shown in Fig. 10, comprises three  
 414 arched scissor-jointed mechanisms, consisting of six basic origami units. The length of  
 415 each bar in the lower scissor joint unit measures 2100 mm ( $2c$ ). To ensure smooth and



416 uninterrupted movement of the entire structure, we have set the angles between the  
 417 retractable panels to be  $\zeta_1 = 45^\circ$  and  $\zeta_2 = 180^\circ$ , corresponding to the fully un- and folded  
 418 states of the arched variant of the retractable roof. For further details regarding the  
 419 specific parameters of the retractable roof model, kindly refer to Table 2.



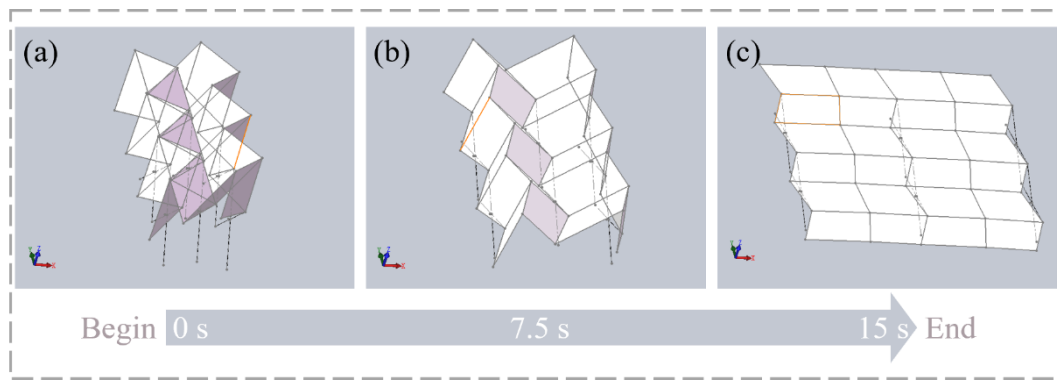
420  
 421 Fig. 10 The Arched variation of the Miura-ori retractable roof model.

422 Table 2 The specific parameters of the arched variation of Miura-ori retractable roof

$a$ (mm)	$h$ (mm)	$\beta_1$ ( $^\circ$ )	$\beta_2$ ( $^\circ$ )	$\zeta_1$ ( $^\circ$ )	$\zeta_2$ ( $^\circ$ )	$c$ (mm)
1500	1000	80	70	45	180	1050

423 Note: the parameters are illustrated in Fig. 4 and Fig. A. 1.

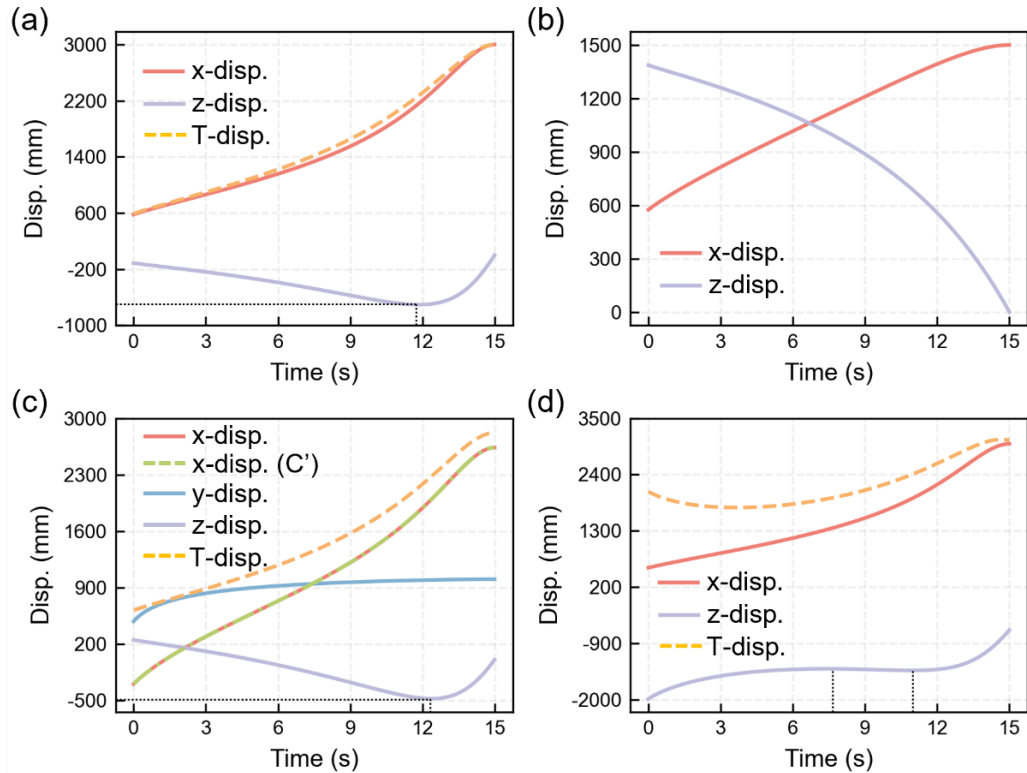
424 We utilize revolute connections in SolidWorks to link the bars of the transverse and  
 425 longitudinal scissor joint units. Through path coordination, the retractable roof moves  
 426 along a predetermined motion trajectory line (the dash lines in Fig. 10) and is propelled  
 427 by a motor. A rotational driving motor is positioned at point O, the center point of the  
 428 entire structure (as depicted in Fig. 10). This motor uniformly increases the included  
 429 angle between adjacent roof panels at an angular velocity of  $9^\circ/\text{s}$  and for 15 seconds.  
 430 The simulation, presented in Fig. 11, demonstrates that the entire mechanism can  
 431 smoothly achieve opening and closing.



432

433 Fig. 11 The motion of the arched variation of the Miura-ori retractable roof. (a), (b),  
 434 and (c) respectively denote the 0th, 5th, and 15th seconds during the course of motion.

435 Fig. 12 illustrates the displacement-time curves of reference points A, B, C, and D  
 436 concerning the center point O of the structure. As the structure is in motion, the  
 437 displacement of all reference points in x-direction increases smoothly and attains their  
 438 maximum values at  $t = 15$  s. Point B follows circular motion around the center point O  
 439 in y-direction, with its displacement in this direction gradually decreasing over time.  
 440 Point D represents the intersection of the lower ends of two scissor joint units and  
 441 experiences no displacement in y-direction. Nevertheless, its displacement in z-  
 442 direction gradually increases (i.e., moving upward) over time, then begins to decrease  
 443 slowly at  $t = 7.40$  s, before moving upward again at  $t = 10.96$  s. Notably, during the  
 444 period of  $t = 7.40 \sim 10.96$  s, the structure primarily unfolds in x-direction, with a  
 445 relatively minor degree of unfolding in y-direction. Point D ultimately reaches its  
 446 highest point at  $-640.31$  mm. Simultaneously, the displacements of points C and C' in  
 447 x-direction remain precisely the same, indicating that the entire structure moves in a  
 448 coordinated manner.



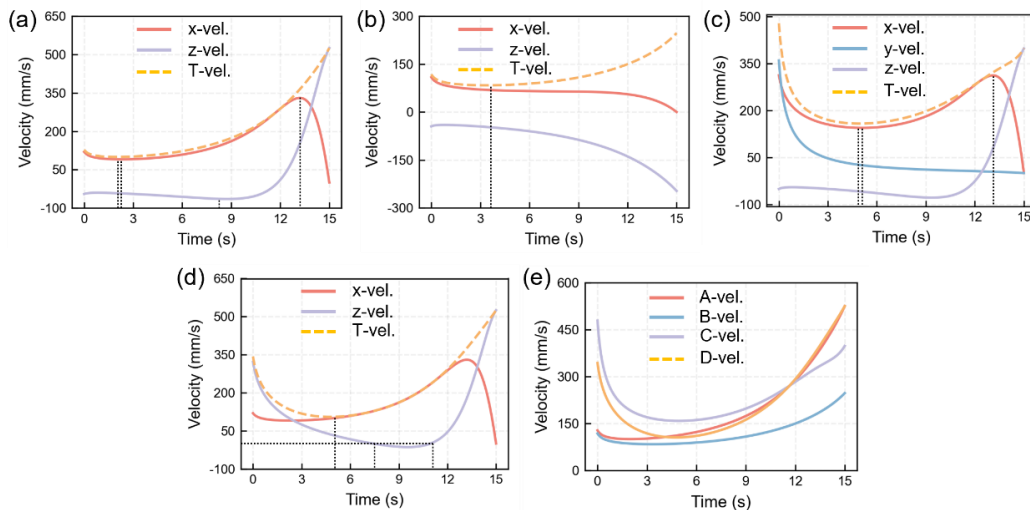
449

450 Fig. 12 The displacement-time curves of the arched variation of retractable roof. (a),  
 451 (b), (c), and (d) denote the displacement-time relationship of points A, B, C, and D,  
 452 respectively. (x-, y-, and z-disp. represent the displacement components of a reference  
 453 point in the three principal axis directions while T-disp. is the total displacement of  
 454 the corresponding point.)

455 The velocity-displacement curves of the key reference points, as shown in Fig. 13,  
 456 reveal that the motion process can be segmented into three stages. During the initial  
 457 stage ( $t = 0 \sim 4.76$  s), the velocity components of each reference point in all directions  
 458 are relatively uniform. In the middle stage ( $t = 4.76 \sim 13.20$  s), the total velocity and x-  
 459 component velocity of the four reference points are nearly equivalent, indicating that  
 460 the movement is primarily in x-direction. In the subsequent stage ( $t = 13.20 \sim 15.0$  s),  
 461 the x-component velocities of points A, C, and D decrease rapidly to zero,

462 demonstrating that they move primarily in the z-direction. In contrast, point B  
 463 undergoes circular motion around the center point O, with a relatively lower velocity  
 464 compared to other points but with a smooth movement.

465 Points C and D exhibit a significant decrease in velocity, followed by a rapid  
 466 increase, with the times of minimum velocity being identical at  $t = 4.96$  s and  $t = 4.76$   
 467 s, respectively. In the early stage, points A and B experience similar velocity changes,  
 468 with a substantial reduction in velocity during the initial stage (at  $t = 2.0$  s and  $t = 3.4$  s,  
 469 respectively). The decrease in velocity for point A is more significant than that of point  
 470 B. Throughout the motion process, the velocity of point B remains the lowest among  
 471 the four key reference points. During the period of  $t = 0$  to 4.12 s,  $v_D$  is greater than  $v_A$ ,  
 472 but the difference gradually decreases, and at  $t = 4.12$  s, the instantaneous velocities of  
 473 these two points are the same. Subsequently, their velocities become closer and exceed  
 474 the velocity of point C at  $t = 11.68$  s.

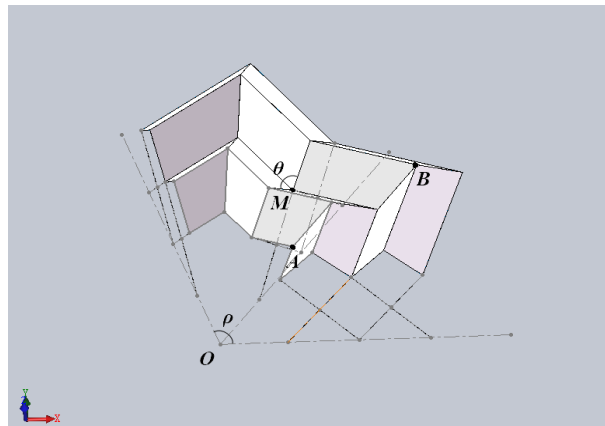


475  
 476 Fig. 13 The velocity-time curves of the arched variation of the Miura-ori retractable  
 477 roof. (a), (b), (c), and (d) denote the velocity-time relationship of points A, B, C, and

478 D, respectively; (e) depicts the velocity comparison of four reference points. (x-, y-,  
 479 and z-vel. represent the velocity components of a reference point in the three principal  
 480 axis directions while T-vel. is the total velocity of the corresponding point.)

### 481 4.3 Fan-shaped variant of retractable roof

482 The fan-shaped variant retractable roof model (depicted in Fig. 14) comprises four  
 483 basic units, with two units oriented in the circumferential and radial directions,  
 484 respectively, and the length of the bars in the lower scissor joint units is  $l = 4500$  mm.  
 485 To ensure smooth movement of the entire structure, the angles between the retractable  
 486 roof panels are  $\theta_1 = 30^\circ$  and  $\theta_2 = 180^\circ$  for the fully folded and unfolded states of the  
 487 fan-shaped variant, respectively. For additional information regarding the specific  
 488 parameters of the retractable roof model, kindly refer to Table 3.



489

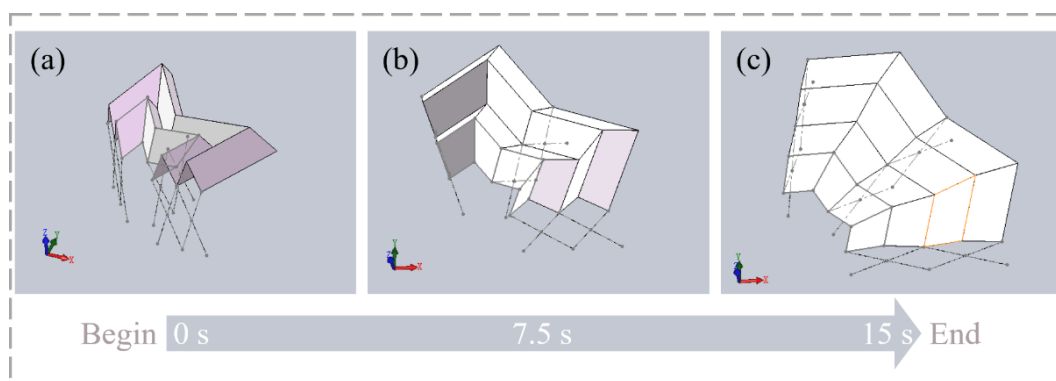
490 Fig. 14 The fan-shaped variation of the Miura-ori retractable roof model.

491 Table 3 The specific parameters of the fan-shaped variation of the Miura-ori retractable  
 492 roof

$b_1$ (mm)	$a_c$ (mm)	$\varphi_1$ (°)	$\varphi_2$ (°)	$\theta_1$ (°)	$\theta_2$ (°)	$l$ (mm)
1500	2000	80	60	30	180	4500

493 Note: the parameters are illustrated in Fig. A. 2.

494 The simulation software employs revolute connections to link the bars of the  
 495 transverse and longitudinal scissor joint units at the connection points. By path  
 496 coordination, the retractable roof moves along the trajectory line at the lower end of the  
 497 scissor joint units (the dash lines in Fig. 14) and is driven by a motor. In Fig. 14, a  
 498 rotational driving motor is installed at point M, which denotes the center point of the  
 499 upper panel structure. This motor uniformly increases the angle between two adjacent  
 500 panels at an angular velocity of  $10^\circ/\text{s}$  and persists for 15 seconds. The simulation,  
 501 presented in Fig. 15, illustrates that the entire mechanism achieves smooth opening and  
 502 closing.

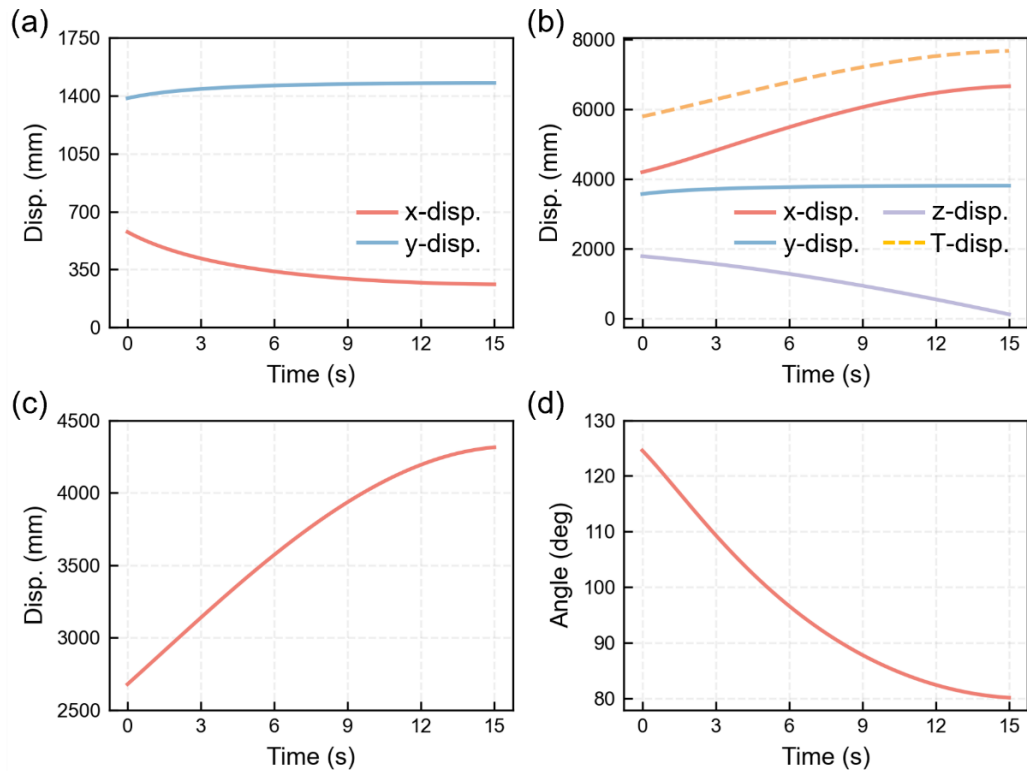


503  
 504 Fig. 15 The motion process of the fan-shaped variation of the Miura-ori retractable  
 505 roof.

506 The displacement-time curves of the motion reference points A and B during the  
 507 movement process are shown in Fig. 16a-b. The displacements of the retractable roof  
 508 in y-direction undergo minimal change, with a displacement increase of only 6.68% as  
 509 the roof unfolds. During the motion process, point A follows a counterclockwise path  
 510 around the center point O, with its displacement in x-direction gradually decreasing as  
 511 time  $t$  increases. Since point B shares the same polar angle as point A in the projection  
 512 plane, their displacements and velocities are identical in y-direction, as evident from

513 the trend of the curves in Fig. 16b. Nevertheless, due to the outward expansion  
514 movement of the retractable roof, the displacement of point B exhibits an opposite trend  
515 to that of point A in x-direction. As the motion of the retractable roof continues, the  
516 displacement of point B in x-direction gradually increases, but the rate of displacement  
517 change remains comparable to that of point A. The displacement of point B in z-  
518 direction reflects the variation pattern of the height of the retractable roof, which  
519 decreases gradually with time  $t$  during the unfolding process, exhibiting a relatively  
520 minor rate of change, and its curve is approximately linear.

521 Fig. 16c presents the curve of the distance (polar radius) from point A to the polar  
522 axis in the projection plane as a function of time. The polar radius of point A increases  
523 from an initial value of 2678.54 mm to 4313.43 mm, and its rate of change gradually  
524 decreases, which indicates that during the unfolding process, the inner edge of the fan-  
525 shaped variant roof panel gradually moves away from the center of the structure, and  
526 the space in the middle increases as the structure unfolds. Fig. 16d depicts the variation  
527 of the maximum angle  $\rho$  (as shown in Fig. 14) of the structure. The  $\rho$  gradually  
528 decreases with time  $t$ , with its rate of change also gradually decreasing. Throughout the  
529 unfolding process, the angle  $\rho$  decreases from  $124.57^\circ$  to  $80.15^\circ$ , representing an  
530 opening efficiency of 35.66% in the circumferential direction. Overall, the movement  
531 of the entire structure is well-coordinated.



532

533 Fig. 16 Parameters at each reference point of the fan-shaped variant retractable roof.

534 (a) and (b) denote the displacement-time relationship of points A and B, respectively;

535 (c) depicts the distance of point A from the polar; (d) is the maximum angle  $\rho$  of the

536 retractable roof. (x-, y-, and z-disp. represent the displacement components of a

537 reference point in the three principal axis directions while T-disp. is the total

538 displacement of the corresponding point.)

## 539 5. Conclusion

540 Three different retractable roof structure systems, inspired by the concept of

541 origami, are proposed in this paper, and the process of conducting thick plate analysis

542 and parametric modeling, adopting Grasshopper, is elaborately described. Theoretical

543 analysis of the three retractable roof structures is then carried out using geometric

544 formulas, followed by a detailed motion simulation analysis. Our main contributions



545 and conclusions:

546 (1) Three novel retractable roof structure systems are proposed. We propose three  
547 new types of retractable roof structure systems by combining the geometric principles  
548 of the Miura-ori mechanism and its arched and fan-shaped variations with the scissor  
549 hinge rectangular unit. The necessary conditions for the coordinated motion of the three  
550 new retractable roof structure systems in two directions are further derived, and the  
551 compatibility solution is proposed for cases where coordinated motion cannot be  
552 achieved between the scissor joint and origami mechanisms.

553 (2) The effect of thickening on the maximum opening ratio of the structure is  
554 presented. The thickening method based on the shape adjustment technique is proposed,  
555 and the formula for the maximum opening ratio of the new retractable roof system is  
556 derived accordingly. The analysis results demonstrate that the effect of panel thickness  
557 on the maximum opening ratio shows different patterns as the value of the notch length  
558 varies. Specifically, as the depth of the cut increases while keeping the panel thickness  
559 constant, the maximum opening ratio also increases. When it comes to the case where  
560 the panel thickness is fixed and the depth of the cut varies, a consistent pattern is  
561 observed: the maximum opening ratio increase as the length of the notch enlarges.

562 (3) Parametric modeling steps are elaborately described. The rigid origami truss  
563 structure, based on the principles of origami mechanism, is established by adopting a  
564 “two-step” approach that uses Grasshopper and isogeometric segmentation). The  
565 modeling steps and parameter control methods are described in exhaustive detail.

566 (4) The kinematic patterns of the new retractable roof structures are investigated.  
567 The simulation demonstrates that all three types of structures can be un- and folded  
568 smoothly without any occurrence of motion interference: 1) During the motion process  
569 of the Miura-ori retractable roof structure, the primary movement is transverse, which  
570 gradually shifts to longitudinal movement. It is sufficient to control the movement  
571 during the initial phase of motion as the speed of motion slows down in the later stages.  
572 2) As to the arched variant, the velocity relationship between the four key reference  
573 points is such that  $v_A \approx v_D > v_C > v_B$ , which suggests that the overall motion speed of the  
574 edge ends and scissor joint units is greater than that of the middle part of the structure.  
575 Therefore, in practical applications, it is crucial to prioritize control of the speed of the  
576 far end. 3) As the fan-shaped variant retractable roof structure unfolds, the z-component  
577 of point B reflects a nearly constant rate decrease in the structure height. Additionally,  
578 the included angle  $\rho$  of the structure gradually decreases with increasing time  $t$ , with a  
579 progressively slowing rate of decrease.

580 We, in this paper, primarily focus on the development of new structural systems and  
581 the theoretical analysis and motion simulation of their coordinated motions. The  
582 following aspects, nevertheless, require further research. Firstly, there is still much  
583 exploration needed for retractable roof structure systems that draw inspiration from  
584 origami while also incorporating more practical designs. Moreover, future research is  
585 recommended to concentrate on refining the design of the nodes and examining their  
586 impact on the structural motion and mechanical performance, for we streamlined the

587 design of connection nodes in the current study. Lastly, it is of utmost importance to  
588 carry out practical experiments to verify the workability of the structures and the  
589 precision of the theoretical investigations, owing to the cross-disciplinary nature of this  
590 research.

### 591 **Declaration of competition interest**

592 All authors have no conflict of interest to declare.

### 593 **Acknowledgment**

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597 Project of the Jiangsu Higher Education Institutions of China.

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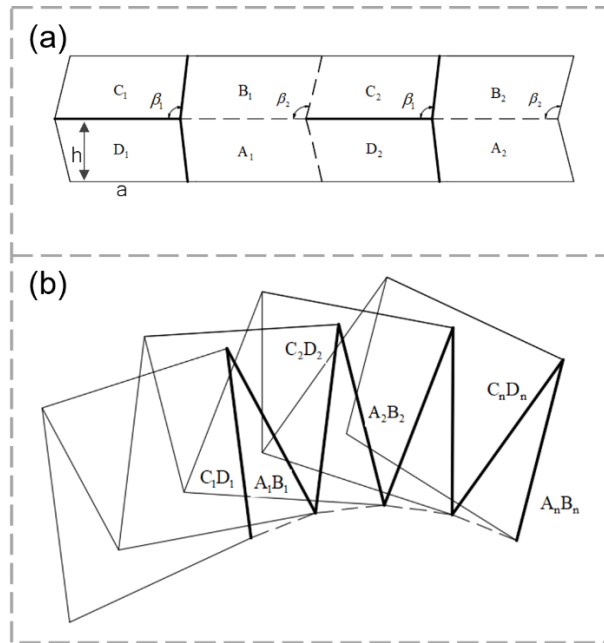
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675

676 **Appendix A.**

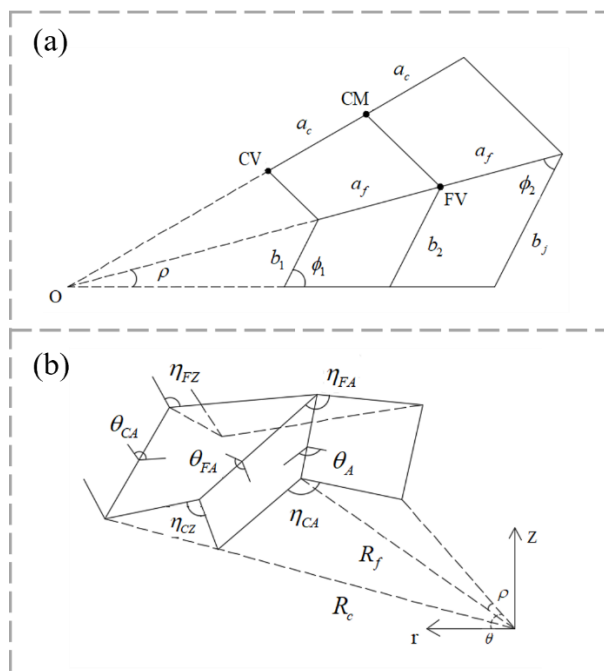


677

678 Fig. A. 1 Geometric parameters of the arched variant. (a) the arched variant unit; (b)

679

The arched variant unit after folding.



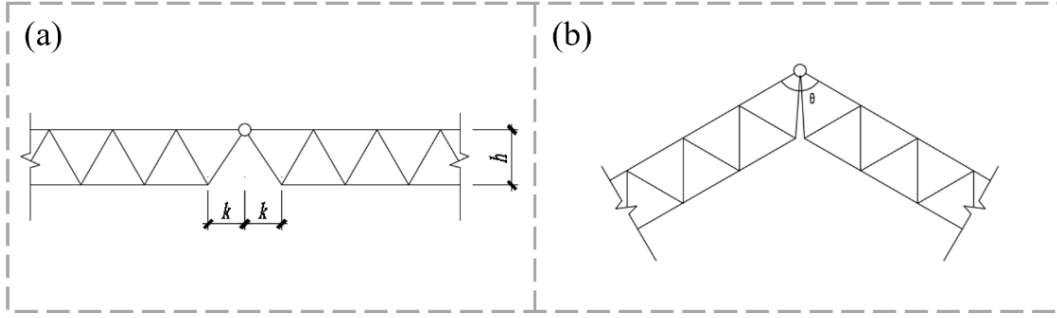
680

681 Fig. A. 2 Geometric parameters of the fan-shaped variant. (a) The fan-shaped variant

682

unit; (b) The fan-shaped variant unit after folding.

683 **Appendix B. Derivation of the equation for thick plate**



684

685 Fig. B. 1 The simplified diagram of the thick plate analysis. (a) Unfolded state; (b)

686

Folded state.

687

The simplified diagram of the thick plate analysis, illustrated in Fig. B. 1, involves

688

cutting the panels on both sides of the crease with a thickness of  $h$  to create symmetrical

689

notches with a length defined as  $k$ . The fold angle,  $\theta$ , is defined as the angle between

690

adjacent panels. The minimum value of  $\theta_{\min}$ , when the panels are fully folded, is related

691

to both  $h$  and  $k$ . The relationship between these three can be expressed as:

692

$$\theta_{\min} = 2 \arctan \frac{h}{k} \quad (\text{B.1})$$

693

To control the maximum opening rate of the Retractable roof structure designed

694

with origami principles, the values of  $h$  and  $k$  (related to plane thickness and notch

695

length) require to be analyzed and studied. The formula of the opening rate is

696

$$\eta = 1 - \frac{S_o}{S_c} \quad (\text{B.2})$$

697

where  $S_o$  and  $S_c$  are the areas of the retractable roof when fully open and closed,

698

respectively.  $S_o$  represents the projection area of the retractable roof when the origami

699

mechanism is wholly folded, while  $S_c$  denotes the projection area when it is totally

700

unfolded. For a Miura-ori retractable roof with  $n_1$  and  $n_2$  units in the longitudinal ( $y$ )

701

and transverse ( $x$ ) directions, respectively, if the acute angle of the parallelogram unit



702 is  $\beta$  and its corresponding sides lengths are  $a$  and  $b$ , respectively, then expressions for  
 703  $S_o$  and  $S_c$  can be obtained accordingly:

$$704 \quad S_c = n_1 \cdot n_2 \cdot (4ab \cos(\frac{\pi}{2} - \beta)) = 4abn_1n_2 \sin \beta \quad (B.3)$$

$$705 \quad S_o = n_1 \cdot n_2 \cdot (4ab \cos(\frac{\pi}{2} - \beta) \cdot \sin \frac{\theta_{\min}}{2}) = 4abn_1n_2 \sin \beta \sin \frac{\theta_{\min}}{2} \quad (B.4)$$

706 By substituting Eq. (B.1), Eq. (B.3), and Eq. (B.4) into Eq. (B.2), Eq. (B.5) can be  
 707 obtained.

$$708 \quad \eta = 1 - \frac{S_o}{S_c} = 1 - \frac{4abn_1n_2 \sin \beta \sin \frac{\theta_{\min}}{2}}{4abn_1n_2 \sin \beta} = 1 - \sin \frac{\theta_{\min}}{2} = 1 - \sin(\arctan \frac{h}{k}) \quad (B.5)$$

### 709 **Appendix C. Parametric modeling procedure for the retractable roof structure**

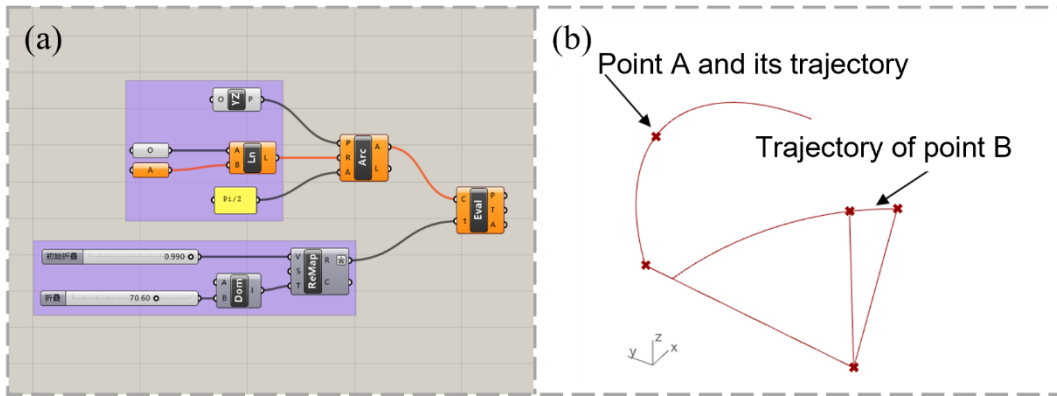
710 The Grasshopper modeling process can be broadly classified into two parts. In this  
 711 study, we demonstrate the parameterization of the retractable roof model using the  
 712 classic Miura-ori as an example.

#### 713 (1) Modeling of zero-thickness origami mechanism

714 To determine the shape of the retractable roof panel in various states, a basic Miura-  
 715 ori structure is established, without considering the influence of thickness, members,  
 716 and other factors.

717 Step 1: Construct the motion trajectory and label the vertices.

718 By defining the side lengths  $a$  and  $b$  of a parallelogram using coordinates and  
 719 utilizing the “Arc” command, the vertices movement paths of the parallelogram  
 720 throughout its entire deployment process can be constructed. The “Construct Domain”  
 721 command allows for the control of the position of vertex A at any given time on its  
 722 movement path, as shown in the figure below.



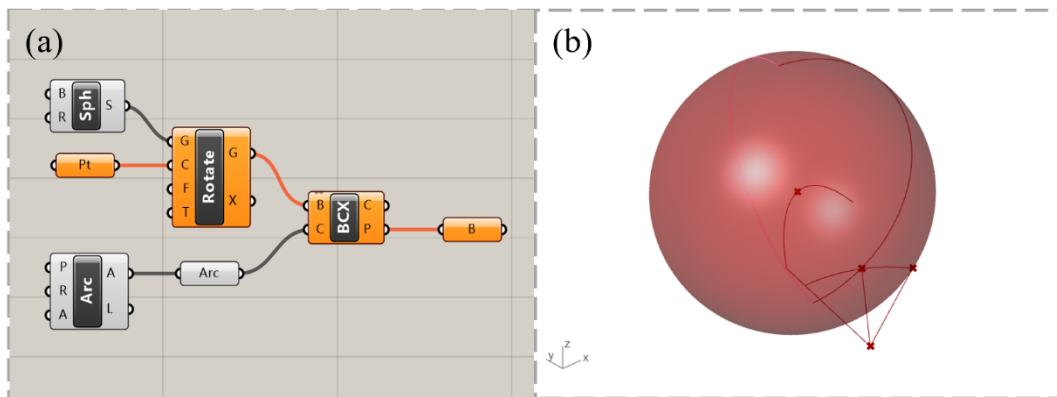
723

724 Fig. C. 1 Construct the motion trajectory and label the vertices. (a) Grasshopper

725 command; (b) Corresponding graphics in Rhino.

726 Step 2: Create a single degree of freedom connection point.

727 Create a reference sphere with a center at point A using the “Sphere” command, and  
 728 then use “Brep/Curve” command to automatically calculate the intersection points  
 729 between the reference sphere and the motion path to ensure that the position of point A  
 730 always determines the position of point B. All vertices are related to point A, and the  
 731 entire structure has, therefore, a single degree of freedom.



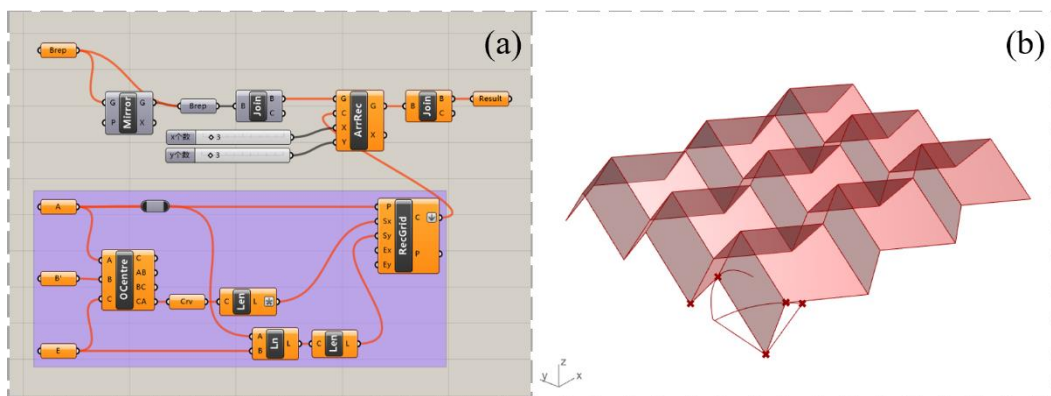
732

733 Fig. C. 2 Create a single degree of freedom connection point. (a) Grasshopper

734 command; (b) Corresponding graphics in Rhino.

735 Step 3: Build the zero-thickness panel.

736 By utilizing vector displacement and mirroring commands on the key points A and  
 737 B identified in the previous steps, a parallelogram panel with controllable position and  
 738 orientation is formed, which further forms a set of panel units with adjustable  
 739 parameters. Lastly, all panel units, using the “Brep Join” command, are integrated into  
 740 a single entity, as illustrated in. The entire mechanism can achieve a single degree of  
 741 freedom un- and folding using the “Construct Domain” progress bar introduced in step  
 742 1.



743 Fig. C. 3 Build the zero-thickness panel. (a) Grasshopper command; (b)  
 744

745 Corresponding graphics in Rhino.

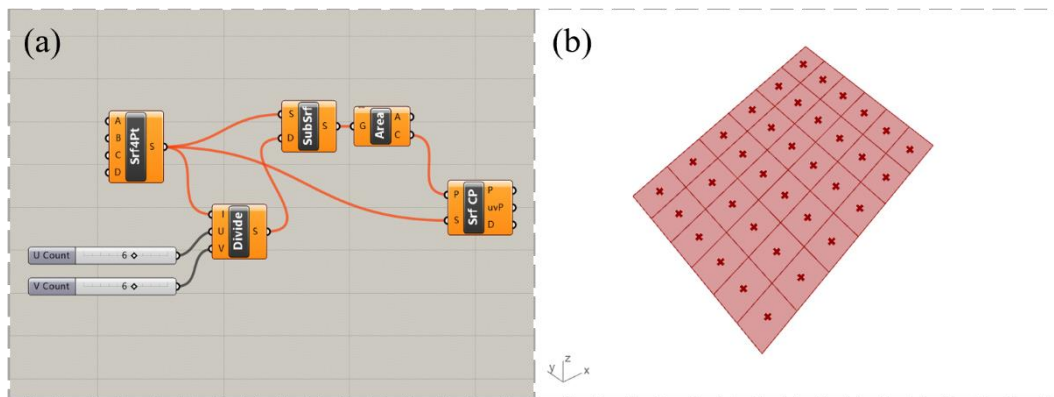
746 (2) Establishing retractable roof grid structure with thickness.

747 A planar truss structure with thickness, based on the previously built zero-thickness  
 748 panel element, is constructed to establish a retractable roof structure suitable for  
 749 practical analysis.

750 Step 1: Divide the panel to get the meshes.

751 To begin, set the number of mesh lines along the u- and v-direction of the single  
 752 roof panel to the corresponding numbers. Next, divide the panel into several small  
 753 parallelogram meshes along the u- and v-direction using the “Divide Domain”

754 command. The center points of each small mesh are then extracted using the “Area”  
755 and “Srf CP” commands.



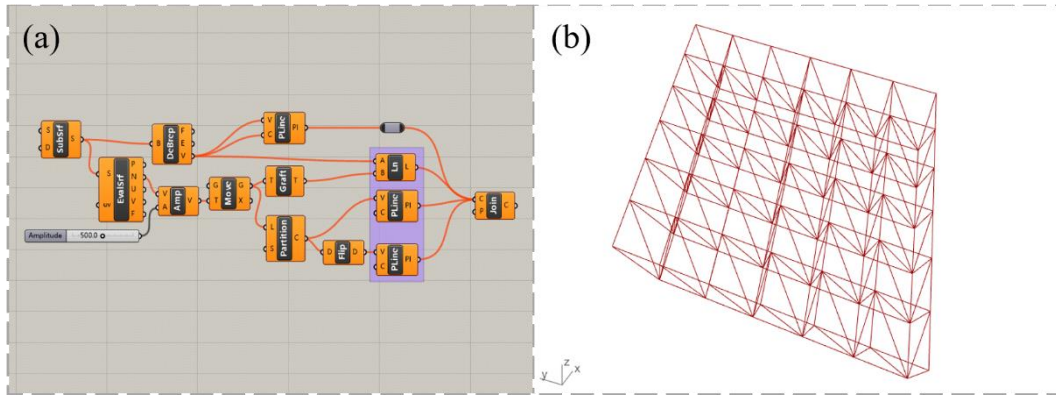
756

757 Fig. C. 4 Divide the panel to get the mesh. (a) Grasshopper command; (b)

758 Corresponding graphics in Rhino.

759 Step 2: Construct single-piece trusses.

760 The normal direction of each mesh plane is determined using the “Eval Srf”  
761 command. The center points of meshes, extracted from step 1, are then offset a certain  
762 distance, along the normal direction, which is set as the thickness of the truss structure.  
763 The “Graft Tree” and “Flip” commands are then used to adjust the coding of each point  
764 based on the characteristics of the mesh distribution of the truss structure. The upper  
765 and lower chords, as well as the web members, are subsequently connected according  
766 to different coding arrangements. Finally, the “Joint Curves” command is used to join  
767 them together to form a complete truss structure.



768

769

Fig. C. 5 Construct single-piece trusses. (a) Grasshopper command; (b)

770

Corresponding graphics in Rhino.

771

Step 3: Create a retractable roof structure with rigid folding capability.

772

Several single truss structures are integrated to form a retractable roof structure with

773

rigid folding capability based on the number and position of the previously described

774

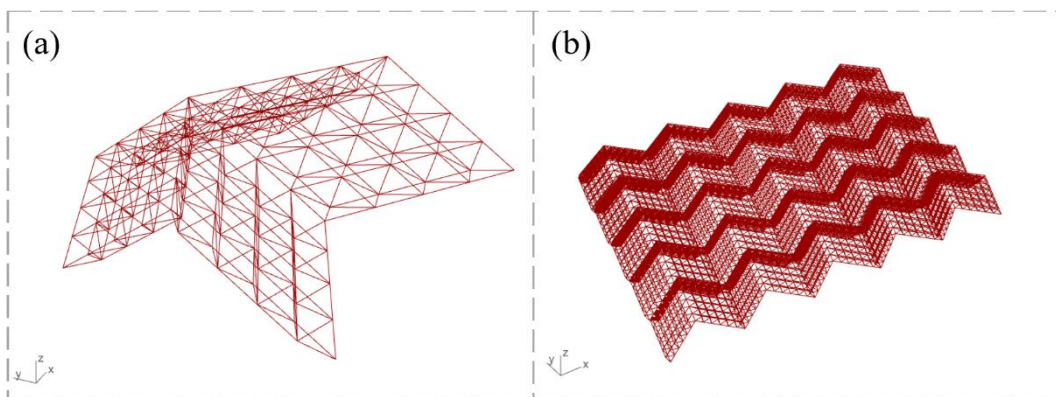
zero-thickness folding panels. To achieve retractable roof structures with varying

775

heights, shapes, and folding states, one can simply adjust the corresponding parameter

776

values in the key steps outlined above.



777

778

Fig. C. 6 Create a retractable roof structure with rigid folding capability: (a)

779

Grasshopper command; (b) Corresponding graphics in Rhino.