

1 Optimising speed and frequency of a scale model smooth vibrating drum roller 2 on lunar highland regolith simulant

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8 9 **ABSTRACT**

10
11 As part of NASA's Artemis program, establishing a sustainable lunar base near the Moon's
12 South Pole is a key objective. Understanding the effects of compaction on the lunar surface is
13 essential to support infrastructure development and long-term exploration activities. This study
14 presents the results of laboratory experiments conducted using a scaled lunar vibrating drum
15 roller (LVDR-1) to investigate the densification behaviour of lunar highland regolith simulant.
16 LVDR-1 used in this study had two different roller masses: 2.92 kg and 4.45 kg. Besides mass,
17 experiments conducted evaluated the effects of varying roller speed and motor frequency. Key
18 metrics such as surface settlement and roller-induced vertical pressure were measured to assess
19 compaction effectiveness. The vertical pressure results were most influenced by the frequency
20 at which the LVDR-1 operated. For the 2.92 kg mass, the highest peak pressures were observed
21 at 2 km/h and 40 Hz, whilst for the 4.45 kg mass, 6 km/h and 40 Hz were found to be optimal,
22 confirming that the optimal combination of speed and frequency differs with roller mass.

23
24 **KEYWORDS:** lunar construction, vibrating smooth drum roller, mass, frequency, speed.

25 26 **1. INTRODUCTION**

27
28 The National Aeronautics and Space Administration (NASA) seeks to establish the Moon as a
29 strategic hub for sustained deep space exploration. A central objective of this initiative is the
30 construction of a lunar base in the highland region near the Moon's South Pole. However, the
31 lunar surface is covered by an unconsolidated layer of fine debris, known as regolith (Carrier
32 III et al., 1991), which presents significant engineering challenges.

33 Translating terrestrial construction practices to the lunar environment requires a
34 comprehensive understanding of the Moon's unique geotechnical and environmental
35 conditions. Compared to Earth, the Moon has a gravitational acceleration approximately one-
36 sixth that of Earth's, lacks both water and atmospheric pressure, and experiences extreme
37 temperature fluctuations that occur between lunar day and night (Vaniman et al., 1991). These
38 conditions significantly influence soil behaviour and consequently construction
39 methodologies. Lunar infrastructure will require the development of roads, landing pads and
40 structural foundations that can support heavy loads transported from landing zones to
41 operational bases. The regolith thickness varies across the surface, ranging from 3 to 5 metres
42 in the lunar maria region to 10 to 20 metres in the highlands region (Oravec 2009). A clear

43 understanding of lunar regolith compaction behaviour is, therefore, essential for designing
44 stable and durable foundations that meet lunar mission requirements.

45 Agarwal et al. (2024) conducted scaled laboratory tests using the LVDR-1 vibrating
46 smooth drum roller (VSDR) with a mass of 2.92 kg to assess its effectiveness in compacting
47 lunar highland regolith simulant. In a subsequent study, Agarwal et al. (2025, under review)
48 extended this work by comparing the performance of the LVDR-1 with a four-sided scale
49 model impact roller. This work demonstrated that vibration was a more effective ground
50 improvement technique than impact for compacting the lunar highland simulant tested.

51 The present study goes beyond conventional parametric variation by revealing
52 previously unexplored dynamic interactions among mass, speed and vibration frequency for
53 compaction of lunar highland regolith simulant. Rather than treating these parameters
54 independently, this work identifies the interdependent relationship between these parameters
55 that governs densification efficiency. A comprehensive experimental framework is developed
56 to capture non-linear compaction behaviour by a vibrating smooth drum roller, which has not
57 been reported in prior studies. The findings establish new mechanistic insights that directly
58 inform the optimisation of lunar surface compaction systems.

59 This study advances the design of lunar regolith compaction strategies for in situ
60 resource utilisation (ISRU), thereby contributing scientific understanding and engineering
61 relevance to sustainable lunar infrastructure development.

62

63 **2. BACKGROUND**

64

65 In terrestrial soil compaction, air voids are typically reduced via a mechanical process
66 involving static, dynamic, impact or vibratory loading. Agarwal et al. (2024) evaluated
67 compaction methods suitable for lunar applications by comparing six different compaction
68 techniques that are widely used within the terrestrial construction industry. The review
69 identified impact rollers and vibrating smooth drum rollers (VSDRs) as the most promising
70 techniques for lunar application. Building upon this study, Agarwal et al. (2025, under review)
71 conducted a comparative analysis between scale models of a 4-sided impact roller and a VSDR.
72 The experiments demonstrated that the VSDR achieved greater surface settlement, higher peak
73 pressures across multiple depth intervals, larger densities, and a greater depth of influence than
74 the impact roller.

75 The compaction performance of VSDRs has been widely studied in terrestrial
76 applications; however, investigations specific to extraterrestrial environments, such as the lunar
77 surface, remain limited. Wersäll (2016) highlighted the significance of vibration frequency in
78 influencing soil compaction outcomes. In the context of lunar infrastructure development, a
79 thorough understanding of the behaviour of lunar highland regolith simulant under varying
80 vibratory loading conditions is essential.

81 In terrestrial applications, the selection of roller mass is a critical factor that directly
82 influences the depth, uniformity and effectiveness of soil compaction. VSDRs are typically
83 available in a range of operating masses, varying from 7,000 kg to over 20,000 kg, depending
84 on the model and ballast configuration (Caterpillar, 2025). These varying masses are
85 specifically designed to suit different soil types and compaction requirements in the field. To

86 investigate how mass affects the compaction behaviour of lunar highland regolith simulant,
87 two distinct masses are tested: 2.92 kg and 4.45 kg.

88 In the present study, as well as in recent investigations by Scott et al. (2023), Agarwal
89 et al. (2024, 2025(under review)), the same lunar highland regolith simulant has been
90 employed. Owing to the limited availability of actual lunar highland regolith samples,
91 simulants are routinely adopted for experimental research (Toklu & Akpınar, 2022). This study
92 utilises LHS-1E (engineering-grade lunar highland simulant), commercially produced by
93 Space Resource Technologies (Space Resource Technologies, 2021), previously known as
94 Exolith Lab. LHS-1E is composed of 75% anorthosite and 25% basalt, designed to closely
95 replicate the mineralogical composition of the actual lunar highlands regolith. Previous work
96 by Agarwal et al. (2023) established that the geotechnical characteristics of LHS-1E are a
97 credible surrogate for actual lunar highland regolith, particularly in the context of geotechnical
98 and construction-related studies.

99 This study presents a comprehensive parametric investigation of scaled VSDRs,
100 focusing on the compaction performance of a lunar highland regolith simulant. The current
101 study integrates a systematic analysis of roller mass, speed and motor frequency to identify
102 optimal combinations for efficient compaction. This investigation used surface settlement
103 measurements and real-time monitoring using buried earth pressure cells, enabling a detailed
104 assessment of compaction behaviour. These results establish a performance baseline while
105 providing novel insights into operational parameters and roller mechanics, with direct
106 implications for lunar and planetary surface construction applications.

107

108 3. METHODOLOGY

109

110 Two bins (Bin A and Bin B), each measuring 750 mm in width, 1060 mm in length,
111 and 380 mm in depth, were used to contain the lunar highland regolith simulant, as shown in
112 Figure 1. The lunar vibrating smooth drum roller (LVDR-1) model (Figure 2) is mobilised
113 along the test lane via a chain-driven carriage system, powered by a variable-speed electric
114 motor.

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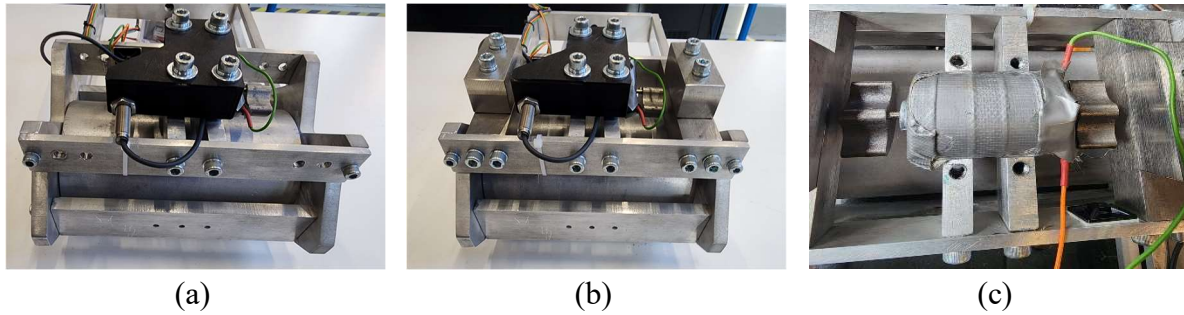
Figure 1. Test facility setup after lunar highland regolith simulant placement.

118 To investigate the compaction response of the simulant with depth under terrestrial
119 laboratory conditions, a series of translational speed tests were undertaken. Seven distinct
120 operating speeds in 2 km/h increments, ranging from 2 to 14 km/h, were adopted. At each
121 speed, three motor frequencies of 20, 30, and 40 Hz were applied, resulting in 21 unique
122 combinations of speed and drum vibration frequency.

123 A minimum speed of 2 km/h was selected based on the findings of Henrich (1987), who
124 identified the optimal compaction speed range for vibratory rollers was between 1 to 4 km/h.
125 A maximum speed of 14 km/h was chosen as it is a multiple of the speed increment and it is
126 close to the maximum speed of Astrolab's FLEX (Flexible Logistics and Exploration) rover, a
127 lunar surface mobility platform capable of reaching speeds of up to 15 km/h (Patrascu, 2022).

128 An externally mounted electric motor fitted with an eccentric mass was integrated into
129 the scale model to control vibrational frequency. The rotating eccentric mass generates periodic
130 inertial forces, resulting in dynamic loading that can be precisely regulated through adjustment
131 of the motor speed. This configuration allows for repeatable and consistent frequency
132 conditions during testing. With regard to vibration frequency, three discrete values of 20, 30,
133 and 40 Hz were selected to span the range of the vibratory motor, which was limited due to
134 mechanical constraints. Although Henrich (1987) recommends a frequency range of 25–45 Hz,
135 the three chosen values provided sufficient resolution to examine the influence of vibration on
136 the lunar simulant while remaining within the limits of the equipment. It is also important to
137 acknowledge that the compaction behaviour of actual lunar regolith may vary under the Moon's
138 unique environmental conditions, which include low gravity, vacuum exposure, and the
139 presence of electrostatically charged particles.

140 As mentioned, these tests were conducted using two different roller masses: 2.92 kg
141 and 4.45 kg, producing a total of 42 test combinations. Figure 2 presents the scale model of the
142 VSDR utilised in the compaction experiments carried out on the test lane. The initial
143 configuration featured a drum mass of 2.92 kg (Figure 2a), and an additional mass was
144 subsequently affixed to the top of the model to increase the mass by approximately 50%,
145 resulting in a total drum mass of 4.45 kg (Figure 2b). In this configuration, four rectangular
146 blocks were mounted by screws on the sides of the motor—two on each side—within the frame,
147 without contacting the drum. It should be noted that this represents a limitation of the scale
148 model design, as increasing the mass is only possible by placing additional mass on top of the
149 drum in lieu of placing eccentric masses inside the drum roller. The scale model incorporates
150 a drum of width 165 mm and a diameter of 100 mm, closely representing the scaled dimensions
151 of full-sized terrestrial rollers. Vibration is generated via an eccentrically-loaded electric motor
152 mounted on the drum axle (Figure 2c), effectively emulating the vibratory mechanism found
153 in conventional field compaction equipment. To confirm the operational frequency of the
154 motor, a Fast Fourier Transform (FFT) analysis was performed on earth pressure cells data
155 (discussed below) for each of the 42 test combinations. This analysis verified the actual
156 frequency applied in each combination.



157 **Figure 2. LVDR-1 scale model: (a) 2.92 kg version; (b) 4.45 kg version; (c) Electric**
 158 **vibration motor.**
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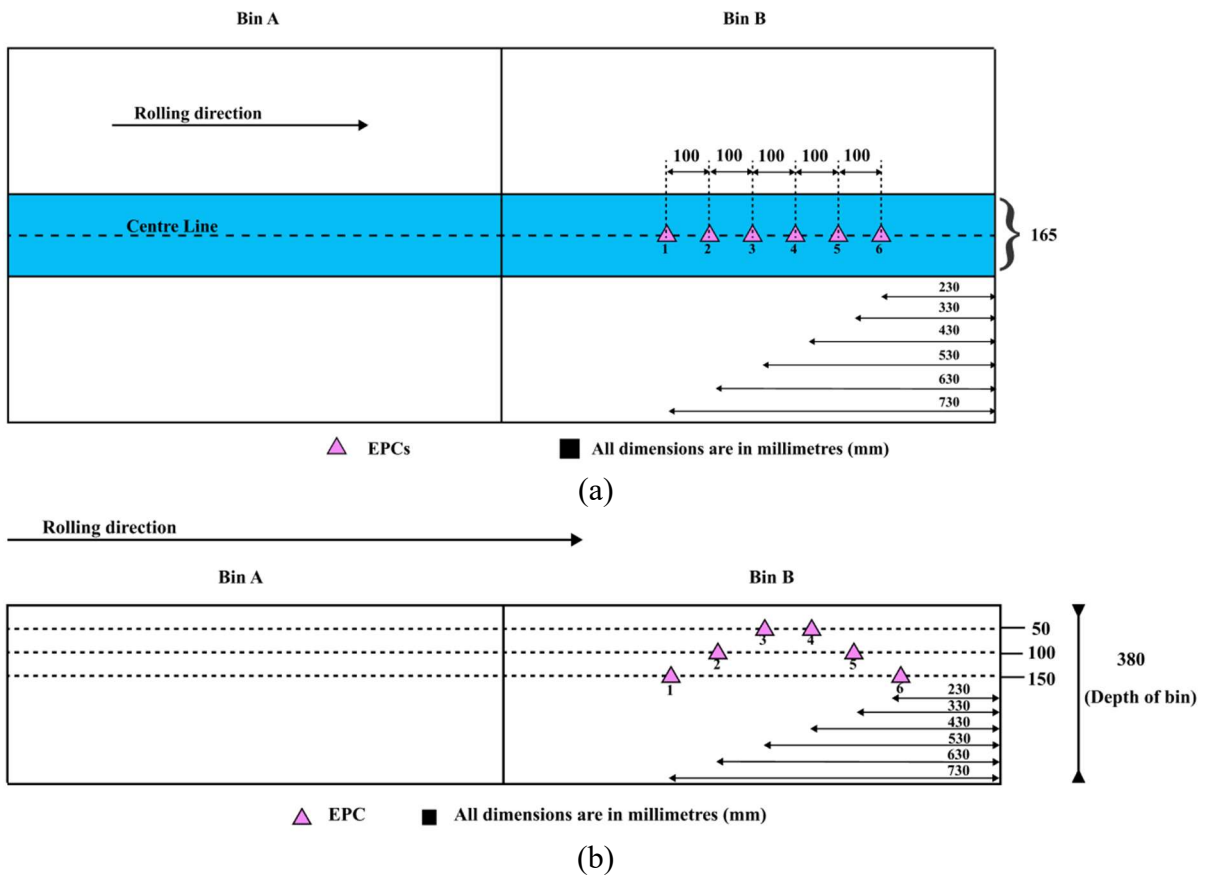
160 The adopted mini earth pressure cells (EPCs), as shown in Figure 3, have a diameter of
 161 19 mm and are equipped with integrated tri-axis accelerometers. Conventional EPCs with
 162 larger diameters have been widely used in geotechnical engineering for in situ earth pressure
 163 measurements (e.g., Rinehart & Mooney 2009; Scott et al. 2019). A 3-axis accelerometer was
 164 fixed to each EPC to provide real-time information regarding the compaction of the lunar
 165 highland regolith simulant. All EPCs and accelerometers were connected to a custom-built data
 166 acquisition system, integrated with a bespoke LabVIEW interface. To accurately capture peak
 167 pressures and ground accelerations during compaction, a sampling frequency of 7,500 Hz was
 168 employed.
 169



170 **Figure 3. Earth pressure cell: (a) Plan view; (b) Internal 3-axis accelerometers.**
 171

172 **Error! Reference source not found.** illustrates the rolling direction of and placement
 173 of EPCs within Bin B. The EPCs were installed at 100 mm intervals to ensure uniform data
 174 acquisition across the test section. As shown in **Error! Reference source not found.**a, the
 175 LVDR-1 unit traversed the designated test lane, which is located in the centre of the bin to
 176 minimise boundary effects, with the distance from the edge of the bin to the centreline of the
 177 roller lane being maintained at 375 mm. In order to achieve a homogenous target density in
 178 each bin, the simulant was weighed and carefully placed into each bin using 10-litre buckets,
 179 and pluviated through a 2.36 mm sieve, so that the simulant fell from a height no greater than
 180 200 mm. To achieve the most uniform placement density possible, care was taken in placing
 181 the simulant in layers not exceeding 50 mm in thickness. During the placement of six EPCs
 182 within Bin B, the EPCs sensors were embedded in the lunar highland regolith simulant at a
 183 lateral spacing of 100 mm and vertical intervals of 50 mm, as illustrated in Figure 4b. Two
 184 EPCs were installed at each depth of 50, 100 and 150 mm, and the average readings from each
 185 pair were used to represent the peak pressure at that corresponding depth. All pressure cells
 186 were zeroed to account for the effects of overburden pressure such that only roller-induced
 187 pressures are reported. This configuration was designed to mitigate the influence of sensor

188 shadowing (Dave & Dasaka, 2011) beneath the centreline of the roller module operating in the
 189 test lane.
 190



195 **Figure 4. Placement of the EPCs: (a) Plan view; (b) Elevation view showing embedment**
 196 **depths.**
 197

198 Prior to testing the speed-frequency combinations, an initial 60 passes of the vibrating
 199 smooth drum roller were carried out at a baseline speed of 4 km/h and a vibration frequency of
 200 30 Hz to induce the majority of surface settlement. Following this, each of the 21 unique
 201 combinations of operating speed and vibration frequency were tested. For each combination,
 202 the vibrating smooth drum roller executed 10 passes over the test lane. This procedure was
 203 repeated for both roller masses 2.92 kg and 4.45 kg, resulting in a total of 42 distinct test
 204 combinations as outlined in Table 1. The sequence of vibration frequency during testing was
 205 systematically varied to assess the influence across a full range of roller speeds. The initial
 206 vibration frequency was set to 30 Hz, at which all seven speeds were tested. After the
 207 completion of all speeds (i.e., after the 130th pass), the frequency was increased to 40 Hz, and
 208 the same speed sequence repeated. Subsequently, after the 200th pass, the frequency was
 209 reduced to 20 Hz, completing all frequency and speed testing at this roller mass.

210 Following the 270th pass, the roller mass was increased to 4.45 kg. A similar testing
 211 pattern was adopted: the vibration frequency was initially maintained at 30 Hz from passes 270
 212 to 340, then increased to 40 Hz until the 410th pass, and finally reduced to 20 Hz for the
 213 remaining passes as summarised in Table 1.

214 The different speeds at which the model traversed the bins, measured in mm/s, were
 215 converted to their full-scale equivalents in km/h by using appropriate scaling laws and
 216 principles (Altae & Fellenius, 1994), as presented in Table 1. Further details regarding the
 217 scale model test rig used in this study are provided by Chung et al. (2017).

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 219

Table 1. Compaction parameters at varying speeds and vibration frequencies.

Combination	Roller mass (kg)	Speed (km/h)	Speed (mm/sec)	Vibrating frequency (Hz)	Passes
0	2.92 kg	4	85	30	0-60
1		4	85	30	61-70
2		6	128	30	71-80
3		8	171	30	81-90
4		10	214	30	91-100
5		12	256	30	101-110
6		14	299	30	111-120
7		2	43	30	121-130
8		4	85	40	131-140
9		6	128	40	141-150
10		8	171	40	151-160
11		10	214	40	161-170
12		12	256	40	171-180
13		14	299	40	181-190
14		2	43	40	191-200
15		2	43	20	201-210
16		4	85	20	211-220
17		6	128	20	221-230
18		8	171	20	231-240
19		10	214	20	241-250
20		12	256	20	251-260
21	14	299	20	261-270	
22	4.45 kg	4	85	30	271-280
23		6	128	30	281-290
24		8	171	30	291-300
25		10	214	30	301-310
26		12	256	30	311-320
27		14	299	30	321-330
28		2	43	30	331-340
29		4	85	40	341-350
30		6	128	40	351-360
31		8	171	40	361-370
32		10	214	40	371-380
33		12	256	40	381-390
34		14	299	40	391-400
35		2	43	40	401-410
36		2	43	20	411-420
37		4	85	20	421-430
38		6	128	20	431-440
39		8	171	20	441-450
40		10	214	20	451-460
41		12	256	20	461-470
42	14	299	20	471-480	

220

221 Surface settlements were measured along the centre of the test lane. A standard tape
222 measure was used at 150 mm intervals along the test rig track. The test rig provided a stable
223 and repeatable datum for consistent measurements throughout the experiment. As mentioned
224 above, for each unique combination of roller speed and vibration frequency, a total of 10 passes
225 were performed, after which six settlement readings were recorded, and the mean value
226 calculated to represent the surface settlement corresponding to that test combination. The initial
227 measurements, corresponding to zero passes, were considered the baseline for assessing the
228 impact of subsequent compaction passes.

230 4. RESULTS

231
232 Figure 5. Surface settlement versus number of passes. presents the surface settlement results as
233 a function of the number of compaction passes using the VSDR on the centre lane. As expected,
234 the most significant settlement was recorded during the initial passes, consistent with previous
235 findings (Agarwal et al., 2024, 2025(under review)), which attributed early-stage deformation
236 to the loose packing of the initially placed simulant.

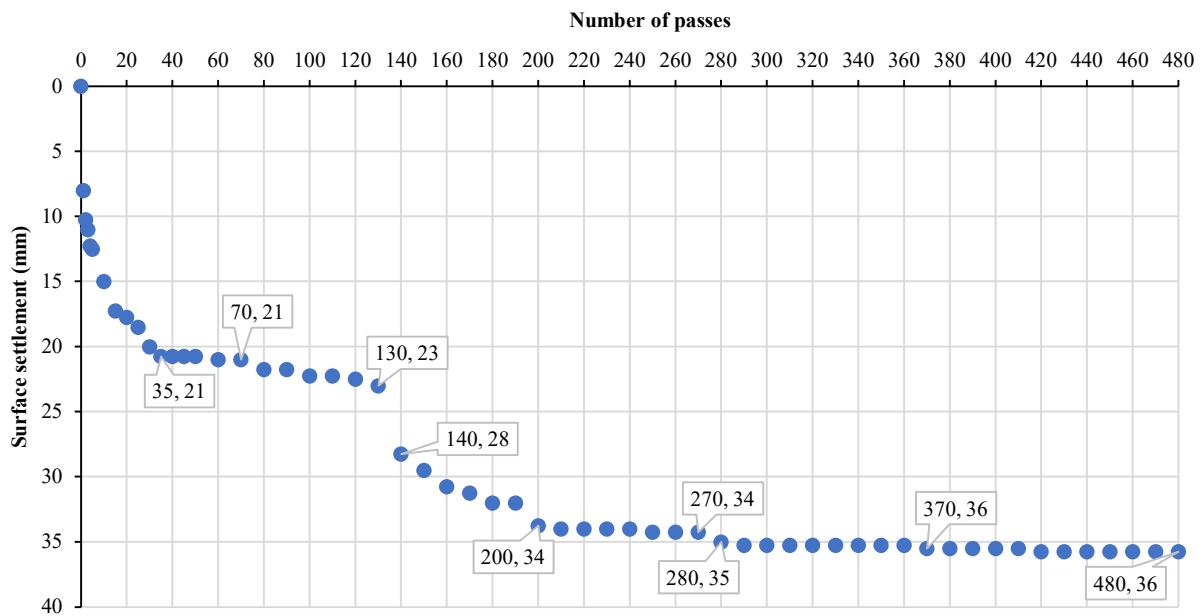
237 A surface settlement of approximately 21 mm was observed within the first 35 passes,
238 after which the rate of settlement substantially declined, indicating the onset of compaction
239 stabilisation. As mentioned above, 60 passes were conducted at a fixed speed of 4 km/h and a
240 vibration frequency of 30 Hz. Beyond this point, different combinations of speed and frequency
241 were implemented to further examine the compaction behaviour. Between the 60th and 130th
242 passes, VSDR operation at 30 Hz with varying roller speeds produced negligible settlement.

243 A notable increase in surface settlement was observed at the 140th pass, with an
244 additional 5 mm recorded relative to the measurement at the 130th pass. This behaviour
245 coincided with an increase in motor frequency from 30 Hz to 40 Hz, suggesting a potential
246 correlation between higher excitation frequency and enhanced surface deformation at this stage
247 of compaction. Between the 140th and 190th passes, only minor variations in settlement were
248 observed under 40 Hz excitation as the roller speed was varied. However, between the 190th
249 and 200th passes, operating the roller at a reduced speed of 2 km/h under a motor frequency of
250 40 Hz resulted in a marked increase in surface settlement, underscoring the influence of
251 reduced speed on compaction effectiveness. These findings indicate that lower operational
252 speeds, when combined with higher motor frequencies, enabling more energy to be transferred
253 into the regolith simulant.

254 Following this, the motor frequency was reduced to 20 Hz, and compaction continued
255 at varying roller speeds up to the 270th pass. During this period, no further surface settlement
256 was observed, indicating that a lower motor frequency had negligible influence on surface
257 deformation at this stage. This suggests that 20 Hz was insufficient to induce additional
258 compaction, potentially due to reduced energy input and diminished dynamic interaction with
259 the lunar highland regolith simulant.

260 Again, as mentioned above, following the completion of the 270th pass, the roller mass
261 was increased from its initial configuration of 2.92 kg to 4.45 kg. The same 21 unique
262 combinations of roller speed and motor frequency were subsequently applied using the heavier
263 drum to further assess the effects of increased mass. It can be observed from Figure 5. Surface
264 settlement versus number of passes. that the majority of surface deformation occurred during

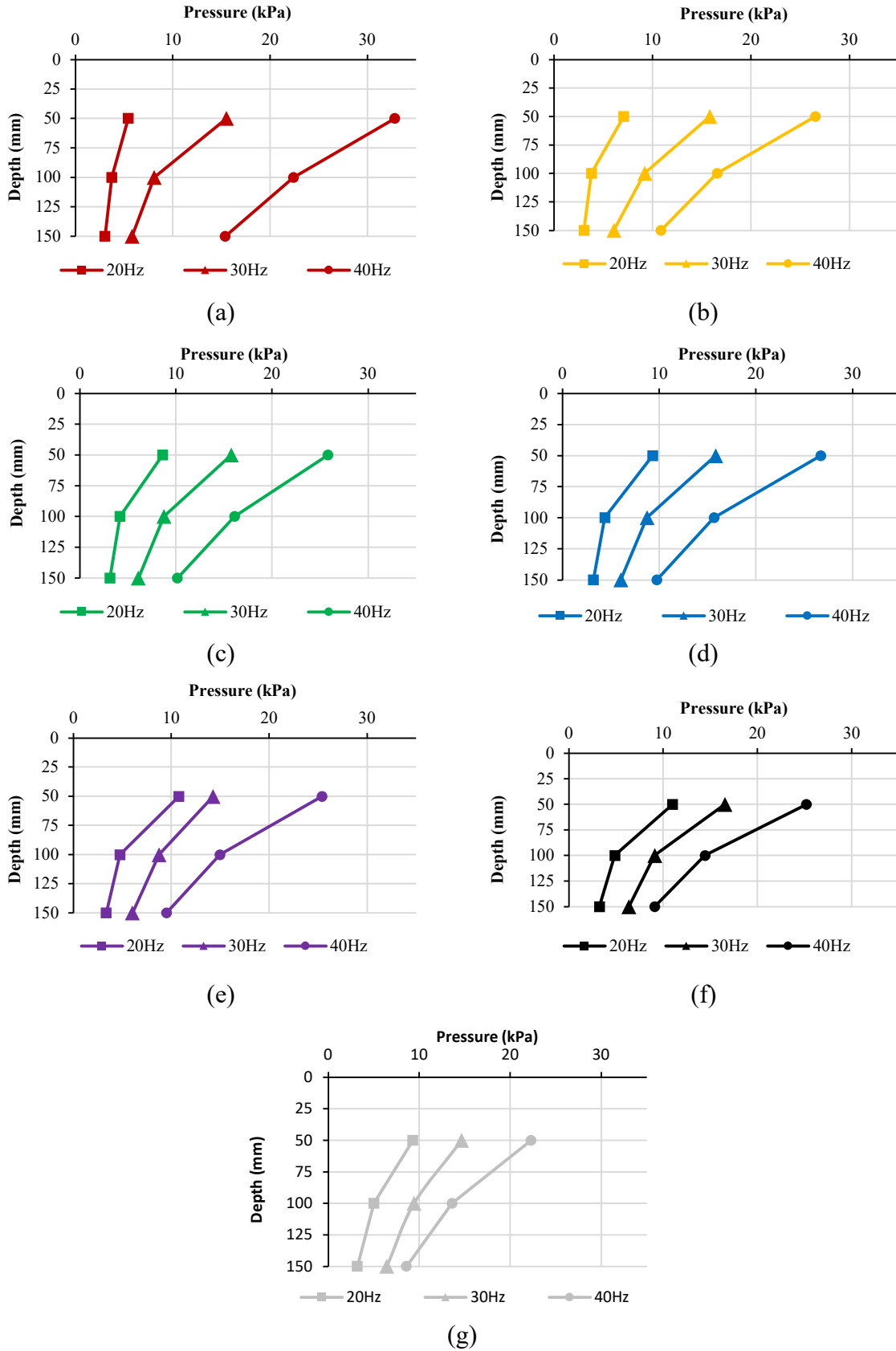
265 the initial passes, with a clear plateauing trend observed in the later stages. This is corroborated
 266 by the measured surface settlements at the 270th and 480th passes, which were approximately
 267 34 mm and 36 mm, respectively. As the number of passes increased, the material exhibited
 268 greater stiffness, resulting in a progressive reduction in the incremental settlement generated
 269 with each subsequent pass. These results confirm the diminishing marginal effect of additional
 270 passes on surface settlement and highlight the enhanced compaction efficiency of LVDR-1,
 271 particularly under conditions of lower roller speed and higher motor frequency. Overall, the
 272 data indicate that reduced operational speed combined with elevated motor frequency
 273 significantly improves compaction performance on lunar highland regolith simulant.
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275
 276 **Figure 5. Surface settlement versus number of passes.**
 277

278 Figure 6. Peak pressure with depth for different operating speeds of LVDR-1 under
 279 motor frequencies of 20, 30, and 40 Hz, using a 2.92 kg drum roller: (a) 2 km/h, (b) 4 km/h,
 280 (c) 6 km/h, (d) 8 km/h, (e) 10 km/h, (f) 12 km/h, and (g) 14 km/h. presents the measured peak
 281 pressures plotted against depth below the ground surface using a drum roller of 2.92 kg mass
 282 for different operating speeds. For each test combination, the peak pressure readings from all
 283 six EPCs were averaged over ten roller passes at each depth. Figures 6a – 6g compare the
 284 results obtained from LVDR-1 at three motor frequencies i.e., 20 Hz, 30 Hz, and 40 Hz applied
 285 to the simulant at operating speeds of 2, 4, 6, 8, 10, 12, and 14 km/h, respectively. In this
 286 analysis, the speed is kept constant within each plot.

287 It can be observed that, at every speed, the 40 Hz frequency consistently produces the
 288 highest roller-induced pressure with depth. For example, in the case of the 2 km/h speed (Figure
 289 6a), the 40 Hz curve exhibits a clear peak pressure higher than those of 20 Hz and 30 Hz. This
 290 trend remains consistent across all speeds, indicating that an increase in motor frequency of
 291 LVDR-1 leads to higher peak pressure irrespective of the operating speed.

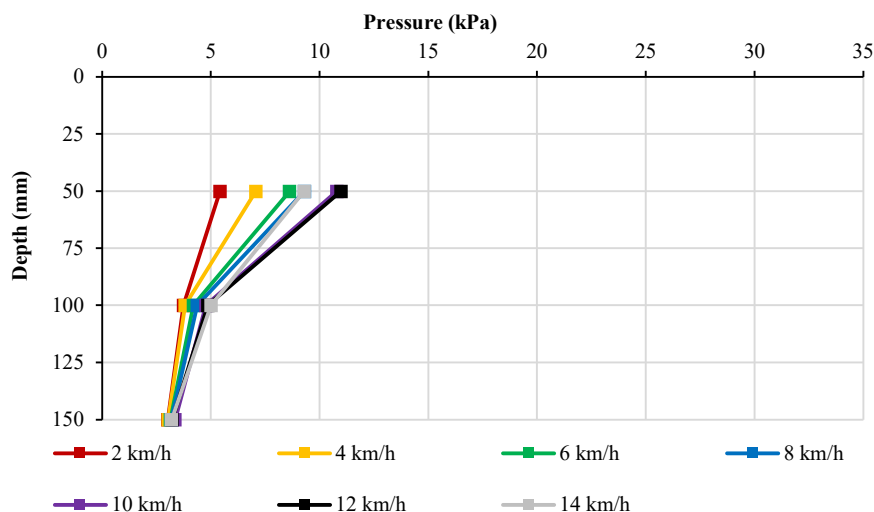


292 **Figure 6. Peak pressure with depth for different operating speeds of LVDR-1 under**
 293 **motor frequencies of 20, 30, and 40 Hz, using a 2.92 kg drum roller: (a) 2 km/h, (b) 4**
 294 **km/h, (c) 6 km/h, (d) 8 km/h, (e) 10 km/h, (f) 12 km/h, and (g) 14 km/h.**

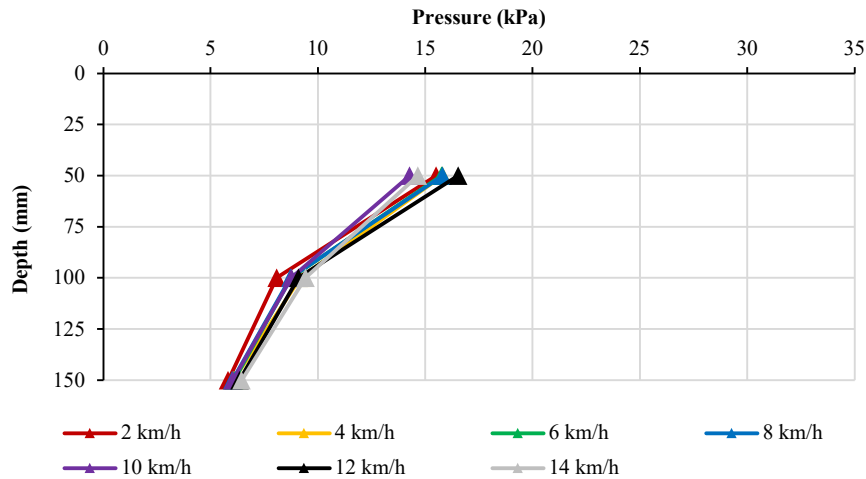
295 Figure 7. Peak pressure with depth for different operating speeds of LVDR-1 using a
 296 2.92 kg drum roller, at fixed motor frequencies: (a) 20 Hz, (b) 30 Hz, and (c) 40 Hz. illustrates
 297 the measured peak pressures plotted against depth when the motor frequency is kept constant
 298 and the operating speed is varied across 2, 4, 6, 8, 10, 12, and 14 km/h using a drum roller of
 299 2.92 kg mass. Figures 7a, 7b, and 7c represent the pressure–depth response of the simulant
 300 corresponding to a constant motor frequency of LVDR-1, i.e., 20 Hz, 30 Hz, and 40 Hz,
 301 respectively. In this analysis, the motor frequency is held constant within each figure, enabling
 302 a direct evaluation of the influence of the LVDR-1 operating speed on the pressure–depth
 303 response. It is clearly observed from Figures 7a and 7b that, for motor frequencies of 20 Hz
 304 and 30 Hz, the highest peak pressures occur at an operating speed of 12 km/h, with average
 305 values of 11 kPa and 17 kPa, respectively, at a depth of 50 mm. However, at the higher motor
 306 frequency of 40 Hz, as shown in Figure 7c, the maximum peak pressure of 33 kPa was recorded
 307 at the lowest operating speed of 2 km/h, indicating that lower speeds combined with higher
 308 motor frequencies generate greater peak pressures. This behaviour may be attributed to the
 309 lower operating speed, which—when combined with a higher vibration frequency—allows the
 310 roller to remain in contact with the simulant for a longer duration per cycle, thereby imparting
 311 greater energy into the simulant.

312 Both Figures 6 and 7 illustrate that the roller-induced pressure is typically greatest near
 313 the ground surface and diminishes with depth, consistent with the findings of Rinehart &
 314 Mooney (2009) and Agarwal et al. (2025, under review). Figures 6 and 7 clearly demonstrate
 315 that an operating speed of 2 km/h at a motor frequency of 40 Hz produces the highest peak
 316 pressure at all measured depths compared with the other speed–frequency combinations with
 317 the roller mass of 2.92 kg. This observation aligns with the surface settlement measurements,
 318 which exhibited noticeable settlement at the roller speed of 2 km/h, further confirming the
 319 enhanced compaction efficiency achieved under these conditions.

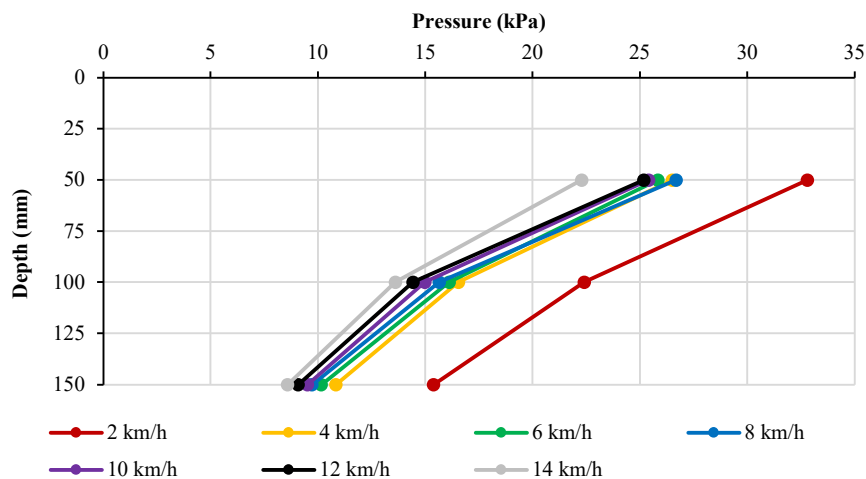
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(a)



(b)



(c)

321 **Figure 7. Peak pressure with depth for different operating speeds of LVDR-1 using a**
 322 **2.92 kg drum roller, at fixed motor frequencies: (a) 20 Hz, (b) 30 Hz, and (c) 40 Hz.**

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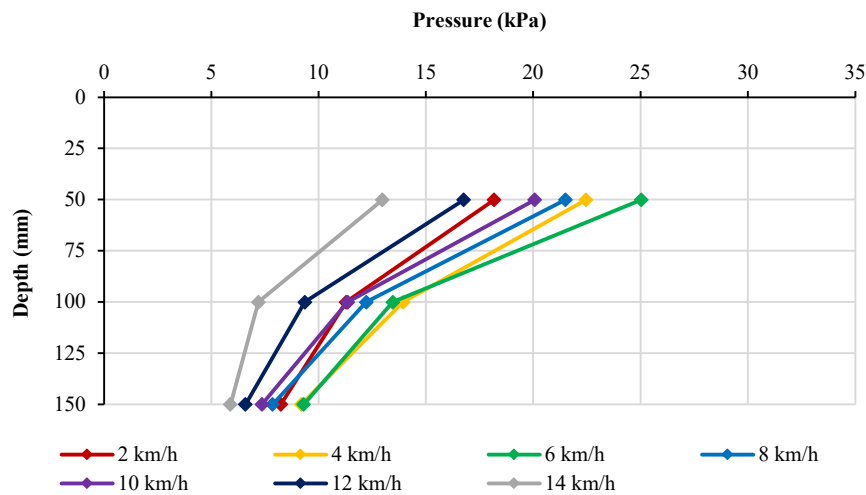
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A similar analysis conducted with the heavier drum roller mass of 4.45 kg exhibited the same behaviour as the 2.92 kg roller, with higher motor frequencies producing greater peak pressures at depth across all speeds tested. Figure 8. Peak pressure with depth for different operating speeds of LVDR-1 using a 4.45 kg drum roller, at 40 Hz motor frequencies. shows the measured peak pressures plotted against depth at a motor frequency of 40 Hz held constant, and the operating speed varied from 2 to 14 km/h, using a drum roller of 4.45 kg mass. From the data, an operating speed of 6 km/h at 40 Hz produced the highest peak pressures for the 4.45 kg mass across all combinations tested. When compared to 2.92 kg mass an operating speed of 2 km/h was found to be optimal. To better understand this discrepancy, a subsequent study is conducted below to compare the 2 km/h and 6 km/h operating speeds at constant frequency, roller mass and simulant density.

Across both roller masses, the findings confirm that motor frequency plays a dominant role in the compaction behaviour of the simulant. The results further indicate that increasing

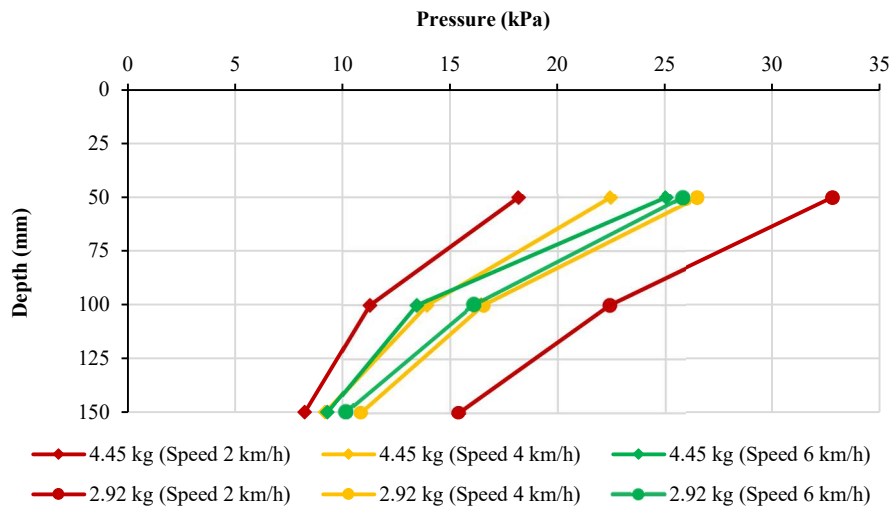
337 the roller mass does not significantly enhance pressure response, a conclusion that aligns with
 338 the surface settlement measurements, which showed negligible variation with changes in roller
 339 mass after the 270th pass. Such insights are particularly valuable for optimising roller operation
 340 in lunar soil compaction applications, where effective transmission of compaction energy to
 341 deeper layers is critical.
 342



343 **Figure 8. Peak pressure with depth for different operating speeds of LVDR-1 using a**
 344 **4.45 kg drum roller, at 40 Hz motor frequencies.**
 345
 346

347 Figure 9. Peak pressure with depth for two roller masses of LVDR-1 at 40 Hz across
 348 operating speeds of 2, 4, and 6 km/h. presents the peak pressures plotted against depth,
 349 illustrating the comparative performance of two roller masses under a constant vibration
 350 frequency of 40 Hz. Operating speeds of 2, 4, and 6 km/h are selected for both roller masses,
 351 as these conditions yielded the highest peak pressures across all tested speed–frequency
 352 combinations (refer to Figures 6, 7, and 8). Operating speeds above 6 km/h are not considered,
 353 as they did not yield effective compaction results, highlighting the importance of optimising
 354 roller speed in combination with vibration frequency to maximise energy transfer into the
 355 simulant. The results indicate that the maximum peak pressure of approximately 25 kPa was
 356 recorded at a depth of 50 mm for the 4.45 kg roller operating at 6 km/h, which is slightly lower
 357 than the 26 kPa measured for the 2.92 kg roller at 2 km/h. At a depth of 150 mm, the peak
 358 pressures were 9 kPa and 11 kPa for the 4.45 kg roller at 6 km/h and the 2.92 kg roller at 2
 359 km/h, respectively, demonstrating that the lower-mass roller consistently produced higher
 360 pressures throughout the depth profile. This effect is particularly pronounced when compacting
 361 low-cohesion granular materials with a relatively high internal angle of friction, such as loosely
 362 packed lunar regolith simulant, where energy transfer and particle rearrangement are strongly
 363 influenced by the amplitude of vibration. (Look, 2022; Mooney & Rinehart, 2009). The 2.92 kg
 364 roller consistently exhibited higher peak pressures throughout the depth profile, suggesting that
 365 greater vibration amplitude facilitated more efficient energy transfer and particle
 366 rearrangement within the simulant. With increasing passes, the stiffness of the material
 367 increased, thereby reducing the magnitude of additional settlement per pass (Anderegg et al.,

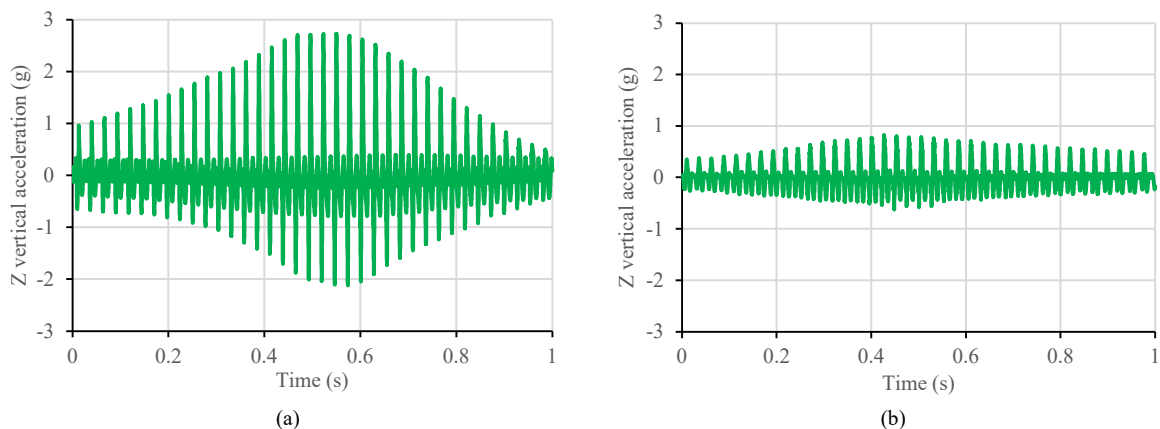
368 2004). This change in surface stiffness in turn affects the vibration amplitude, influencing
 369 energy transmission and overall compaction performance.



370
 371 **Figure 9. Peak pressure with depth for two roller masses of LVDR-1 at 40 Hz across**
 372 **operating speeds of 2, 4, and 6 km/h.**
 373

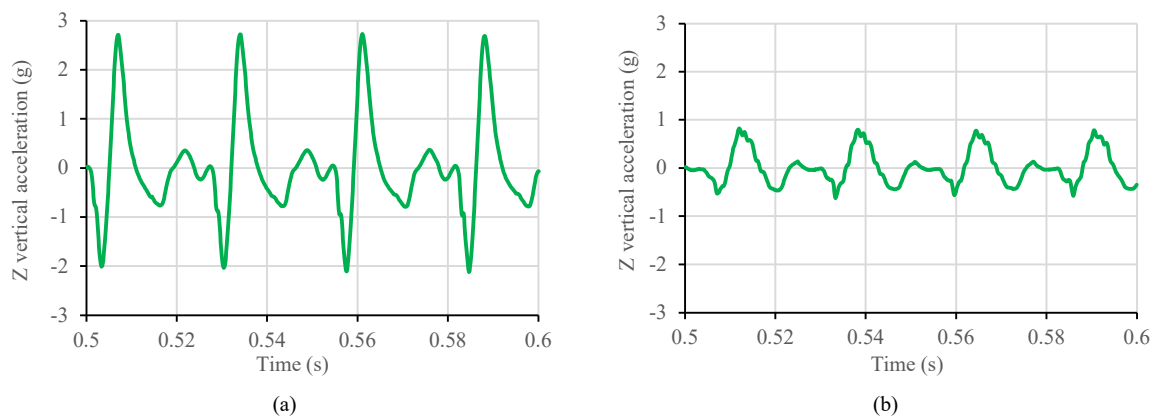
374 Figure 10. Vertical acceleration responses beneath the roller over a 1-second window
 375 at 50 mm depth and constant roller speed (6 km/h) and frequency (40 Hz) a) 2.92 kg and b)
 376 4.45 kg. indicates that increasing the roller mass does not correlate with higher measured
 377 pressures. Figures 10(a) and 10(b) present the vertical acceleration response over a one-second
 378 window, spanning 0.5 seconds on either side of the time corresponding to the peak pressure.
 379 In Figure 10, the pressure cell depth (50 mm), roller speed (6 km/h), and motor frequency (40
 380 Hz) were held constant, with only the roller mass varied.

381 It is evident from Figure 10(a) that the vertical (z-direction) acceleration amplitude is
 382 approximately three times greater than that observed in Figure 10(b). This indicates that the
 383 lighter roller mass generates a substantially larger dynamic response and therefore produces a
 384 greater in-ground acceleration response compared with the heavier roller mass.
 385



386 **Figure 10. Vertical acceleration responses beneath the roller over a 1-second window at**
 387 **50 mm depth and constant roller speed (6 km/h) and frequency (40 Hz) a) 2.92 kg and**
 388 **b) 4.45 kg.**
 389

390 To further examine the behaviour of the simulant within the influence zone beneath the
391 roller, a 0.1-second window of the responses shown in Figures 10(a) and 10(b) are zoomed in
392 and presented in Figures 11(a) and 11(b) for the 2.92 kg and 4.45 kg roller masses, respectively.
393 Figures 11(a) and 11(b) illustrate the vertical acceleration time response, where an initial
394 downward (negative) acceleration occurs as the simulant is loaded by the roller impacting the
395 surface. Following this loading event, the resistance of the lunar simulant is mobilised,
396 resulting in an upward acceleration. The acceleration-time response dampens and approaches
397 zero, remaining near equilibrium until the next vibration cycle occurs approximately 0.025
398 seconds later, corresponding to the 40 Hz operating frequency of the roller. Future studies
399 should therefore focus on measuring amplitude variations under different operating conditions
400 to establish a clearer relationship between vibrational energy input, material stiffness changes
401 and compaction performance.
402



403 **Figure 11. Vertical acceleration responses beneath the roller over a 0.1-second window**
404 **at 50 mm depth and constant roller speed (6 km/h) and frequency (40 Hz) a) 2.92 kg**
405 **and b) 4.45 kg.**
406

407 5. CONCLUSION

408 This study presents the outcomes of an experimental investigation into the influence of
409 operational parameters speed and motor frequency on the compaction performance of LHS-1E
410 (engineering-grade lunar highland simulant) used as a terrestrial analogue for actual lunar
411 highland regolith. The experiments were conducted using a scale vibrating smooth drum roller,
412 incorporating two drum roller configurations with masses of 2.92 kg and 4.45 kg, respectively.
413 The results demonstrate that both roller masses effectively densify the dry simulant, thereby
414 affirming the feasibility of compacting lunar highland regolith simulant and highlighting the
415 potential applicability of ground improvement techniques for lunar surface construction. The
416 experimental investigation demonstrates that motor frequency has a more pronounced
417 influence on simulant compaction than roller speed when using a vibrating smooth drum roller.
418 Across all tests, a higher motor frequency of 40 Hz consistently produced the highest peak
419 pressures at all measured depths, irrespective of operating speed and mass. The optimal
420 combination of motor frequency and operating speed differs with roller mass: the lighter
421 2.92 kg mass achieved best results at 2 km/h and 40 Hz, whereas for the heavier 4.45 kg mass,
422 6 km/h and 40 Hz were found to be optimal. These findings emphasize that amplitude and
423

424 frequency of vibration are more influential than mass in achieving effective energy transfer to
425 the regolith simulant and efficient subsurface compaction. While the method of increasing mass
426 in this study is recognised as a simplification, future research will address this limitation
427 through more advanced modelling and experimental approaches.

428 Overall, this study has tested key operational parameters, including speed, vibration
429 and roller mass, thereby advancing the understanding of lunar highland regolith simulant
430 behaviour. The results were consistent for drum roller masses of 2.92 kg and 4.45 kg,
431 confirming that frequency is the dominant factor controlling dynamic soil response, while roller
432 speed and mass play secondary roles. These findings highlight the importance of optimising
433 motor frequency in lunar highland regolith compaction operations to achieve maximum peak
434 pressure and effective depth penetration, providing valuable guidance for future lunar
435 infrastructure construction and regolith stabilisation strategies. This knowledge is pivotal in
436 informing the design of future in situ construction and surface operations on the Moon, where
437 efficient material handling and compaction will be essential to mission success.

438

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440

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453

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