

# HESR: A Hysteresis-Enhanced Symbolic Regression Framework for Dynamic Friction Modeling and Compensation in Robot Manipulators

**Abstract**—Precise friction compensation in harmonic drives is hindered by complex nonlinear hysteresis, the structural rigidity of traditional physical models, and the limited out-of-distribution generalization of neural networks. This paper proposes HESR, a three-stage symbolic regression framework that introduces a hysteresis-enhanced state variable to address these limitations. Utilizing the ParFam-H algorithm, this approach generates explicit and interpretable friction equations. Experimental validation on a UFACTORY-850 manipulator shows that HESR reduces Root Mean Square Error (RMSE) by over 30% compared to LuGre and RBF neural network baselines. Additionally, the method demonstrates stable cross-frequency generalization (0.1–1.0 Hz) and reduces real-time trajectory tracking errors by 49.3%, offering an effective and transparent compensation strategy.

## I. INTRODUCTION

The construction of precise inverse dynamics models is essential for high-performance robotic control. Although rigid-body dynamics formulations [1], [2] are well-established, significant discrepancies remain between theoretical models and physical systems due to complex joint friction. This issue is particularly acute in collaborative robots utilizing harmonic drives, where flexspline deformation and gear meshing introduce nonlinear stiffness and position-dependent torque ripples [3]. These factors couple with friction, resulting in pronounced hysteretic effects that are comprehensively analyzed in classical mechatronic studies [4], yet remain challenging to model accurately [5].

Current friction compensation strategies face a fundamental trade-off between accuracy and interpretability, a challenge that continues to drive recent advancements in continuous dynamic modeling for high-precision tasks [6]. Conventional model-based approaches [7], [8] offer physical insight but struggle to capture the complex, coupled nonlinearities of harmonic drives due to their rigid structures. Conversely, data-driven methods, such as neural networks [9], [10], [11], [12], achieve high approximation accuracy but function as “black boxes,” often generalizing poorly to out-of-distribution motion frequencies.

To bridge this gap, Symbolic Regression (SR) has emerged as a promising tool for discovering explicit physical laws from data [13], [14], [15]. However, existing SR applications in robotics are largely limited to static friction mappings [16], failing to account for the dynamic hysteresis loops observed during velocity reversals. Furthermore, traditional GP-based SR suffers from computational inefficiency and equation bloating [17], hindering the discovery of compact models for complex dynamic systems.

Addressing these limitations, this paper proposes HESR,

a hysteresis-enhanced symbolic regression framework. We introduce a novel three-stage optimization pipeline that replaces traditional GP with the gradient-based ParFam algorithm [18]. Importantly, we incorporate a physics-informed hysteresis state variable as prior knowledge, enabling the automatic evolution of explicit, compact equations capable of capturing non-local memory effects. Comparative experiments on the first three joints of a UFACTORY-850 manipulator demonstrate that ParFam-H significantly outperforms traditional physical models and neural networks in both predictive accuracy and cross-frequency generalization capabilities.

The HESR methodology (Fig. 1) utilizes a UFACTORY-850 platform for data collection and comprises a three-part pipeline: 1) data preprocessing and rigid-body decoupling; 2) ParFam-H optimization for evolving the hysteresis-enhanced friction equation; and 3) experimental validation on the hardware platform. The remainder of this paper is organized as follows: Section II reviews related work. Section III details the rigid-body dynamics identification and friction decoupling process. Section IV introduces the proposed HESR methodology. Section V provides comprehensive experimental validation, and Section VI concludes the paper.

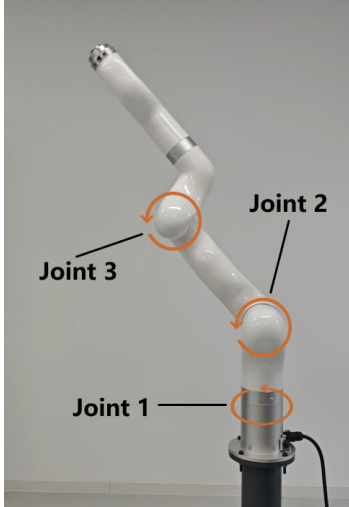
## II. RELATED WORK

### A. Rigid Body Dynamics Identification

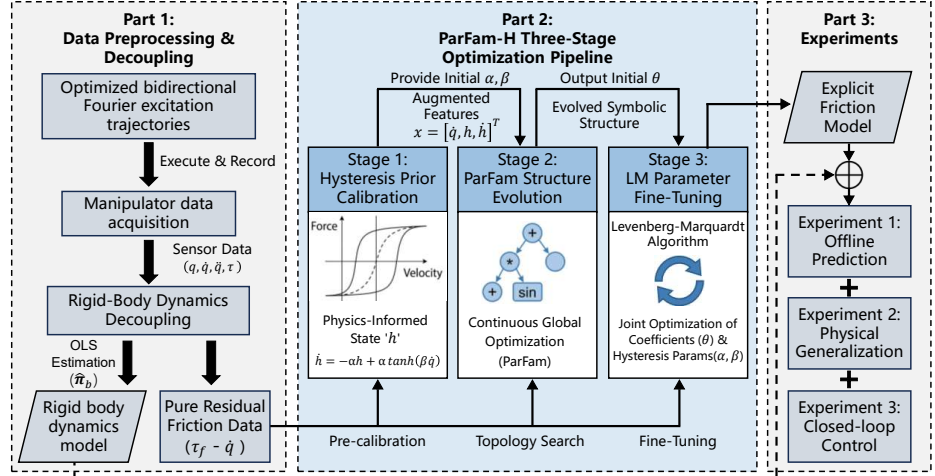
Accurate friction modeling necessitates the isolation of friction torque from the total actuation torque, which relies on a precise rigid-body dynamics model. Typically, dynamics are derived via Newton-Euler or Lagrangian formulations and linearized into a regressor form with respect to inertial parameters. To address parameter redundancy, minimum inertial parameters are identified using QR decomposition or Singular Value Decomposition (SVD). To enhance robustness against measurement noise, optimal excitation trajectories based on finite Fourier series are widely adopted, minimizing the condition number of the observation matrix. Furthermore, to prevent nonlinear friction from contaminating the identification results, techniques such as forward-reverse average motion are commonly employed to cancel out friction terms, thereby isolating the rigid-body dynamics.

### B. Friction Modeling Strategies

Friction modeling is generally categorized into static and dynamic approaches. Static models, such as the Stribeck model [19], assume friction depends solely on velocity, failing to capture position-dependent effects and dynamic memory. To address this, dynamic models like the LuGre



(a) UFACTORY-850 platform.



(b) HESR three-stage optimization pipeline.

Fig. 1. The overall framework of the proposed HESR methodology. (a) The 6-DOF robotic manipulator (UFACTORY-850) used for data collection. (b) The detailed modeling and validation pipeline, comprising three parts: Part 1 handles data preprocessing and rigid-body dynamics decoupling; Part 2 executes the ParFam-H optimization (including hysteresis prior calibration, structure evolution, and parameter fine-tuning) to discover the explicit friction model; and Part 3 conducts experimental evaluation.

model [8] introduce an internal state variable  $z$ , representing the average deflection of bristles between contact surfaces. The friction torque  $\tau_f$  is modeled as:

$$\tau_f = \sigma_0 z + \sigma_1 \dot{z} + \sigma_2 \dot{q} \quad (1)$$

where  $\sigma_0$ ,  $\sigma_1$ , and  $\sigma_2$  represent the stiffness coefficient, microscopic damping coefficient, and viscous friction coefficient, respectively. The evolution of the bristle state  $z$  is governed by:

$$\dot{z} = \dot{q} - \frac{\sigma_0 |\dot{q}|}{g(\dot{q})} z \quad (2)$$

Here, the function  $g(\dot{q})$  captures the Stribeck effect:

$$g(\dot{q}) = F_c + (F_s - F_c) e^{-(\dot{q}/v_s)^2} \quad (3)$$

where  $F_c$  is Coulomb friction,  $F_s$  is static friction, and  $v_s$  is the Stribeck velocity. While the LuGre model unifies pre-sliding, hysteresis, and steady-state characteristics, its fixed mathematical structure limits its flexibility in modeling the complex, coupled nonlinearities specific to harmonic drives.

Conversely, data-driven approaches primarily rely on Neural Networks (NN) [9], [20] due to their universal approximation capabilities. Various architectures, ranging from Radial Basis Function Neural Networks (RBFNN) for static mapping to Long Short-Term Memory (LSTM) networks [10], [21], [22], have been employed to capture dynamic hysteresis. While recent advancements like Physics-Informed Neural Networks (PINNs) [23], [24] attempt to embed physical constraints, these deep architectures remain limited in real-time edge deployment. Deep recurrent structures require intensive matrix multiplications, posing a significant computational burden that introduces unacceptable latency in high-frequency (e.g.,  $\geq 200$  Hz) closed-loop industrial controllers. In contrast, our proposed symbolic framework

explicitly evolves a highly compact, analytical algebraic equation. This explicit formulation provides  $\mathcal{O}(1)$  computational complexity, enabling direct integration into standard microprocessors and reconstructing memory effects with a low computational footprint.

### C. Symbolic Regression in Robotics

Symbolic Regression (SR) bridges mechanistic and data-driven methods by discovering explicit, interpretable analytical equations from data. Recently, SR has been explored for robot inverse dynamics [25] and physical law discovery, with cutting-edge frameworks demonstrating its potential for data-efficient adaptive control in real-world robotics [26]. However, most existing SR applications rely on Genetic Programming (GP), which suffers from slow convergence and a tendency to produce bloated formulas. Critically, current SR research in friction modeling typically establishes only static mappings [27], lacking mechanisms to handle dynamic hysteresis. The memory effect near velocity zero-crossings is vital for high-precision force control but remains unaddressed by simple static SR. Motivated by these challenges, this work combines physics-informed priors (hysteresis states) with the gradient-based ParFam algorithm to achieve dynamic friction modeling that is both accurate and interpretable.

## III. RIGID-BODY DYNAMICS IDENTIFICATION AND FRICTION DECOUPLING

To isolate the complex nonlinear friction dynamics from the total actuation torque for subsequent symbolic regression modeling, a highly accurate rigid-body inverse dynamics model must first be established. This section outlines the dynamics linearization, excitation trajectory design, and parameter identification process.

### A. Dynamics Modeling and Linearization

For the spatial 3-DOF manipulator configuration investigated in this study, the rigid-body dynamics are described by the standard Euler-Lagrange equation:

$$\mathbf{M}(\mathbf{q})\ddot{\mathbf{q}} + \mathbf{C}(\mathbf{q}, \dot{\mathbf{q}})\dot{\mathbf{q}} + \mathbf{G}(\mathbf{q}) = \boldsymbol{\tau}_{rb} \quad (4)$$

where  $\mathbf{q}, \dot{\mathbf{q}}, \ddot{\mathbf{q}} \in \mathbb{R}^3$  are the joint position, velocity, and acceleration vectors, respectively.  $\mathbf{M}(\mathbf{q})$  is the inertia matrix,  $\mathbf{C}(\mathbf{q}, \dot{\mathbf{q}})$  contains Coriolis and centrifugal terms,  $\mathbf{G}(\mathbf{q})$  represents gravity, and  $\boldsymbol{\tau}_{rb}$  is the rigid-body torque.

Exploiting the linear property of robot dynamics with respect to inertial parameters, (4) can be rewritten in a regressor form:

$$\mathbf{Y}(\mathbf{q}, \dot{\mathbf{q}}, \ddot{\mathbf{q}})\boldsymbol{\pi} = \boldsymbol{\tau}_{rb} \quad (5)$$

where  $\boldsymbol{\pi} \in \mathbb{R}^{30}$  is the standard inertial parameter vector (including mass, center of mass, and inertia tensor components for each link), and  $\mathbf{Y}$  is the regression matrix.

Due to kinematic constraints, linear dependencies exist within  $\boldsymbol{\pi}$ , making direct Least Squares estimation ill-posed. Therefore, a minimum parameter set  $\boldsymbol{\pi}_b$  must be extracted. By constructing an extended observation matrix using randomly generated valid joint states, we employ numerical QR decomposition to eliminate redundant columns. For our spatial 3-DOF configuration, the original 30 standard parameters are reduced to a base set of 15 parameters. The minimal dynamics model is thus formulated as:

$$\mathbf{Y}_b(\mathbf{q}, \dot{\mathbf{q}}, \ddot{\mathbf{q}})\boldsymbol{\pi}_b = \boldsymbol{\tau}_{rb} \quad (6)$$

### B. Excitation Trajectory and Decoupling

To enhance the robustness of parameter identification against measurement noise, finite Fourier series are adopted to construct periodic excitation trajectories. The full observation matrix  $\mathbf{W}_b$  is constructed by vertically stacking the regressor  $\mathbf{Y}_b$  calculated at each sampling instant along the trajectory. Consequently, the fundamental frequency and Fourier coefficients are optimized by minimizing the condition number of  $\mathbf{W}_b$ .

During the estimation phase, a forward-reverse symmetric motion strategy is implemented to decouple friction from the rigid-body dynamics. The manipulator is controlled to execute a forward trajectory  $\mathbf{q}^+(t)$  and its time-reversed counterpart  $\mathbf{q}^-(t)$ . Theoretically, the odd-symmetric nature of friction with respect to velocity allows it to be canceled out by averaging the torques from both directions.

However, due to nonlinear hysteresis effects, the friction torque exhibits significant time delay and asymmetry near velocity zero-crossings. To address this, we introduce a data screening mechanism during the decoupling process: data points corresponding to near-zero velocities are deliberately discarded. By retaining only the data segments with well-preserved friction symmetry, the high-confidence rigid-body torque  $\bar{\boldsymbol{\tau}}_{rb}$  is obtained. Finally, the optimal estimate of the base parameters  $\hat{\boldsymbol{\pi}}_b$  is computed using Ordinary Least Squares (OLS):

$$\hat{\boldsymbol{\pi}}_b = (\mathbf{W}_b^\top \mathbf{W}_b)^{-1} \mathbf{W}_b^\top \bar{\boldsymbol{\tau}}_{rb} \quad (7)$$

Once  $\hat{\boldsymbol{\pi}}_b$  is precisely determined, we can extract the high-fidelity residual friction data. Specifically, utilizing the identified  $\hat{\boldsymbol{\pi}}_b$  and the regressor matrix  $\mathbf{Y}_b$ , data is collected under a continuous sinusoidal sweep trajectory with varying frequencies and amplitudes. Assuming the measurement noise follows a zero-mean Gaussian distribution, the residual friction torque  $\boldsymbol{\tau}_f$  is explicitly decoupled by:

$$\boldsymbol{\tau}_f = \boldsymbol{\tau}_{meas} - \mathbf{Y}_b(\mathbf{q}, \dot{\mathbf{q}}, \ddot{\mathbf{q}})\hat{\boldsymbol{\pi}}_b \quad (8)$$

where  $\boldsymbol{\tau}_{meas}$  is the actual total torque measured via joint current feedback. This isolated residual  $\boldsymbol{\tau}_f$  predominantly encapsulates Coulomb, viscous, and complex nonlinear hysteresis friction, serving as the pure data foundation for the subsequent modeling.

## IV. HYSTERESIS-ENHANCED SYMBOLIC FRICTION MODELING

Based on the decoupled residual friction  $\boldsymbol{\tau}_f$ , this section details the proposed HESR framework. To overcome the structural rigidity of conventional parametric models and the lack of interpretability in data-driven neural networks, HESR formulates the modeling process as a systematic three-stage optimization pipeline: 1) *Hysteresis Prior Pre-calibration*, where the shape parameters ( $\alpha$  and  $\beta$ ) of the physics-informed hysteresis state variable are pre-calibrated for each joint to capture non-local memory effects; 2) *Symbolic Structure Evolution*, utilizing the continuous global optimization-based ParFam algorithm to automatically discover an explicit, compact, and mathematically interpretable friction equation; and 3) *Parameter Fine-Tuning*, employing the Levenberg-Marquardt (LM) algorithm to globally refine all numerical coefficients for maximum predictive accuracy.

### A. Hysteresis State Formulation

Most existing symbolic regression models rely solely on joint velocity  $\dot{q}$  as the input, implicitly assuming a static mapping  $\tau_f = f(\dot{q})$ . This fundamentally fails to capture the path-dependent memory effects inherent in actual harmonic drives. Inspired by the classic LuGre model [8], which employs an internal differential state to describe dynamic bristle deflection, we recognize the importance of integrating a dynamic state variable.

However, the LuGre model presents two major limitations in practical harmonic drive control: 1) *Structural rigidity*: it is mathematically hardcoded strictly for idealized friction, restricting its capacity to absorb other complex, unmodeled nonlinearities (e.g., flexspline elastic deformation); and 2) *Parameter overload*: its numerous coupled coefficients are notoriously difficult to identify accurately.

Instead of pre-defining a rigid, parameter-heavy physical equation, we formulate a minimalist, physics-inspired hysteresis state variable  $h$ . This state is governed by a simple first-order nonlinear differential equation utilizing only two physically intuitive parameters ( $\alpha$  and  $\beta$ ), as illustrated in Fig. 2:

$$\dot{h} = -\alpha h + \alpha \tanh(\beta \dot{q}) \quad (9)$$

where  $\alpha > 0$  and  $\beta > 0$  are tunable physical parameters. In a discrete real-time control system with sampling time  $\Delta t$ , the state is iteratively updated via first-order Euler discretization:

$$\begin{cases} h_k = (1 - \alpha\Delta t)h_{k-1} + \alpha\Delta t \tanh(\beta\dot{q}_k) \\ \dot{h}_k = -\alpha h_k + \alpha \tanh(\beta\dot{q}_k) \end{cases} \quad (10)$$

where  $0 < 1 - \alpha\Delta t < 1$  is maintained to ensure numerical stability and non-oscillatory behavior.

This minimalist design effectively separates the physical priors from the data-driven symbolic evolution. By employing (9), we actively delegate the generation of the fundamental phase-delayed memory effect to this simple ODE, overcoming the parameter identification bottleneck of the LuGre model. Simultaneously, the discovery of the exact, complex algebraic mapping—which encapsulates both dynamic friction and other coupled harmonic drive nonlinearities—is entirely offloaded to the ParFam-H symbolic regression engine.

Furthermore, the structural design of (9) is based on the microscopic physical interactions of harmonic drives. From a tribological perspective, the gear meshing contact surfaces can be modeled as interacting elastic asperities (bristles). First, the transition term  $\tanh(\beta\dot{q})$  captures the forced macroscopic sliding, where a larger  $\beta$  reflects the abrupt asperity break-away during the Stribeck transition. Second, the decay term  $-\alpha h$  governs the micro-elastic relaxation process. When the joint velocity reverses and crosses zero, the microscopically deformed bristles do not instantly snap back to a neutral state. Instead, they release their stored elastic strain energy exponentially. The parameter  $\alpha$  explicitly acts as the inverse of this relaxation time constant. By mathematically preserving this residual micro-deformation state,  $h_k \approx (1 - \alpha\Delta t)h_{k-1}$ , the variable  $h$  physically reconstructs the path-dependent hysteresis loop without the structural complexity of traditional models.

Ultimately, both  $h$  and its derivative  $\dot{h}$  are utilized as augmented features to provide crucial dynamic transition information to the regression engine.

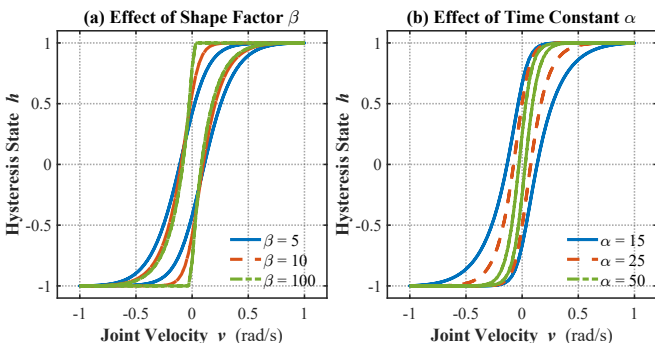


Fig. 2. Composite hysteresis analysis of the proposed friction model governed by Equation (9). (a) The shape factor  $\beta$  dictates the slope of the tanh function near zero velocity. (b) The time constant  $\alpha$  determines the decay rate of the state variable  $h$ , reflecting the memory depth and the width of the hysteresis loop.

### B. Three-Stage Optimization Strategy

Friction modeling using SR is inherently a mixed-integer optimization problem involving both function topology search and parameter identification. To ensure physical consistency and avoid local optima, we propose a physics-guided three-stage optimization framework.

**Stage 1: Hysteresis Prior Pre-calibration.** Before feeding features into the SR algorithm, the physical parameters  $\alpha$  and  $\beta$  are pre-calibrated based on genuine  $\tau_f - \dot{q}$  observations. By geometrically matching the generated virtual state  $h$  to the actual hysteresis loop width and slope at velocity zero-crossings, this pre-calibration ensures the augmented input encapsulates sufficient nonlinear dynamic priors.

**Stage 2: ParFam-based Global Structure Evolution.** Unlike traditional GP, which searches discrete symbolic spaces and is often prone to “code bloat” and poor constant optimization, we employ the ParFam algorithm. ParFam transforms the discrete SR task into a continuous global optimization problem utilizing a Parametric Family of functions, structurally resembling a residual neural network where linear connections are replaced by rational functions  $Q_i$  and nonlinear basis functions  $g_i$ :

$$f_{\theta}(\mathbf{x}) = Q_{k+1}(\mathbf{x}, g_1(Q_1(\mathbf{x})), \dots, g_k(Q_k(\mathbf{x}))) \quad (11)$$

where  $\mathbf{x} = [\dot{q}, h, \dot{h}]^T$  is the augmented input vector, and  $\theta$  denotes the set of all learnable algebraic coefficients. The optimization objective is defined as:

$$\mathcal{L}(\theta) = \frac{1}{N} \sum_{i=1}^N (\tau_{f,i} - f_{\theta}(\mathbf{x}_i))^2 + \gamma \|\theta\|_1 \quad (12)$$

The  $L_1$  regularization (coefficient  $\gamma$ ) prunes the structure by forcing redundant coefficients to zero. We globally optimize  $\theta$  via Basin-Hopping and L-BFGS-B local search. To prevent non-physical bloating, polynomial degrees are restricted to 1, cross-unit operations are prohibited, and  $\mathcal{O} = \{+, -, \times, /, \sin, \cos, \exp, \tanh\}$ . Finally, coefficients below a threshold are truncated, followed by BFGS refinement to yield a concise equation.

**Stage 3: LM-based Global Parameter Joint Fine-Tuning.** Since the pre-calibration in Stage 1 relies on geometric observation, it inevitably introduces initial bias. Therefore, after the symbolic structure is fixed in Stage 2, we release all constraints and employ the Levenberg-Marquardt (LM) algorithm to co-optimize the physical hysteresis parameters  $(\alpha, \beta)$  alongside the discovered algebraic coefficients  $\theta$  as a unified set. This step eliminates cascading errors caused by pre-calibration biases, achieving a globally optimal alignment between physical priors and data-driven fitting.

## V. EXPERIMENTAL EVALUATION

Experiments on the first three joints of a UFACTORY-850 cobot (harmonic drives, 200 Hz) validate the HESR framework (its evolved model is hereafter termed ParFam-H). Evaluations include: 1) Offline Prediction Accuracy against baselines; 2) Physical Generalization via out-of-distribution 0.1–1.0 Hz Chirp signals; and 3) Closed-loop Control tracking performance.

### A. Experimental Setup and Baseline Identification

To isolate pure friction, rigid-body dynamics were first identified using an optimized Fourier trajectory and a forward-reverse symmetric motion strategy. Data within the velocity reversal region ( $|\dot{q}| < 0.1$  rad/s) were discarded to eliminate hysteresis contamination. QR decomposition yielded 15 minimum base inertial parameters, which accurately track the macroscopic rigid-body torque (Fig. 3).

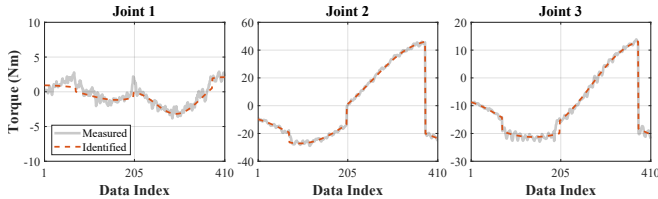


Fig. 3. Comparison of the measured total torque and the identified macroscopic rigid-body torque under the Fourier excitation trajectory.

Subsequently, a sinusoidal sweep (frequency 0.25 Hz, amplitude 1.2 rad/s) was executed. By subtracting the identified rigid-body torque from the measured total torque, high-fidelity residual friction data was extracted. Finally, the ParFam-H algorithm, utilizing augmented features  $\mathbf{x} = [\dot{q}, h, \dot{h}]^T$ , was applied to evolve an explicit friction equation. The specific analytical formulas discovered for each joint, alongside the co-optimized physical hysteresis parameters ( $\alpha, \beta$ ), are detailed in Table I.

TABLE I

EXPLICIT ANALYTICAL FRICTION EQUATIONS EVOLVED BY PARFAM-H

Friction Model Equation	
<b>Joint 1:</b>	
$\tau_{f1} = 16.237h_1 - 0.163\dot{h}_1 - 2.128$	
$+ 37.715 \tanh(\theta_1)$	
where $\theta_1 = 0.193\dot{q}_1 - 0.606h_1 + 0.064\dot{h}_1 + 0.057$	
$(\alpha_1, \beta_1) = (11.37, 256.84)$	
<b>Joint 2:</b>	
$\tau_{f2} = 3.710\dot{q}_2 + 2.936h_2 + 1.220\dot{h}_2 - 0.279$	
$+ 8.290 \cos(\theta_2)$	
where $\theta_2 = -0.865\dot{q}_2 + 1.531h_2 + 0.351\dot{h}_2 + 1.485$	
$(\alpha_2, \beta_2) = (19.62, 143.91)$	
<b>Joint 3:</b>	
$\tau_{f3} = 0.051\dot{q}_3 + 6.614h_3 + 0.238\dot{h}_3 - 0.280$	
$+ 6.527 \tanh(\theta_3)$	
where $\theta_3 = 1.014\dot{q}_3 - 2.130h_3 + 0.115\dot{h}_3 + 0.167$	
$(\alpha_3, \beta_3) = (8.45, 278.23)$	

### B. Offline Prediction Accuracy

To evaluate the fitting capability of ParFam-H on complex friction dynamics, an offline validation was conducted using an untrained, dynamically rich Fourier series trajectory (duration: 20 s, 4000 samples). Actual joint angles and torques were recorded, with velocity and acceleration derived numerically.

ParFam-H was quantitatively compared against four baselines: 1) classic LuGre and 2) Stribeck models (parameters

optimized via Genetic Algorithm); 3) an RBF Neural Network (RBFNN-H), enhanced with the hysteresis state input for fair comparison; and 4) ParFam-V, an ablation symbolic model using only velocity input. These baselines are specifically selected for their balance between approximation capacity and computational efficiency, ensuring strict feasibility for 200 Hz real-time control. While higher-capacity architectures like LSTMs or PINNs are available, their significant recursive computational overhead on standard industrial microprocessors often precludes their use in zero-latency feedforward tasks. Thus, RBFNN-H is employed as a representative high-capacity neural baseline within the bounds of practical real-time deployment.

As shown in Fig. 4 and Table II, ParFam-H achieves the highest fidelity across all three joints. While traditional models (LuGre, Stribeck) yield overly smoothed predictions and RBFNN-H exhibits severe oscillations near velocity reversals, ParFam-H accurately reconstructs the transient torque peaks induced by static friction and hysteresis. The ablation comparison with ParFam-V further proves the indispensability of the hysteresis state  $h$  for capturing reversal dynamics.

Quantitatively, ParFam-H consistently yields the lowest RMSE. For the most dynamically complex Joint 1, ParFam-H achieves an RMSE of 4.282 N·m, outperforming the sub-optimal RBFNN-H (5.186 N·m) by 17.4% and the classic LuGre model (6.370 N·m) by 32.8%. Notably, as an explicit, interpretable mathematical equation, ParFam-H surpasses the fitting-oriented black-box neural network, demonstrating an outstanding balance between predictive accuracy and physical interpretability.

TABLE II

OFFLINE TORQUE PREDICTION RMSE (N·M) COMPARISON

Model	Joint 1	Joint 2	Joint 3
LuGre	6.3703	5.9744	2.4627
Stribeck	6.4110	6.1804	3.2741
RBFNN-H	5.1865	7.7713	2.3954
ParFam-V	6.0660	5.2261	2.6997
ParFam-H	4.2823	5.1197	1.9257

### C. Physical Generalization Capability

As illustrated in Fig. 5, to assess out-of-distribution (OOD) extrapolation, a variable-frequency Chirp signal (0.1–1.0 Hz) was applied, intentionally exceeding the 0.25 Hz training frequency. The data are bisected at 0.6 Hz ( $t \approx 2.6$  s) into an in-distribution (low-frequency) region and an OOD (high-frequency) extrapolation region.

In the low-frequency region, RBFNN-H and ParFam-H exhibit comparable precision (e.g., Joint 1 RMSE: 5.777 N·m vs. 5.616 N·m), confirming the neural network's strong in-distribution approximation ability. However, in the high-frequency region, their performances diverge. The prediction error of RBFNN-H increases significantly; its RMSE on Joint 2 increases from 6.451 N·m to 18.968 N·m (a nearly  $3\times$  increase). This highlights the fundamental limitation of black-

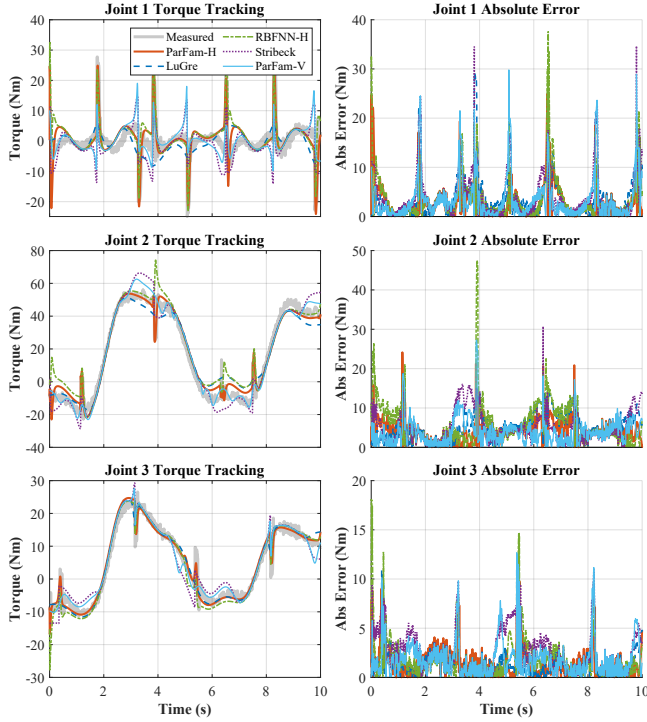


Fig. 4. Torque prediction accuracy and absolute error comparison among different friction models under an untrained, dynamically complex test trajectory.

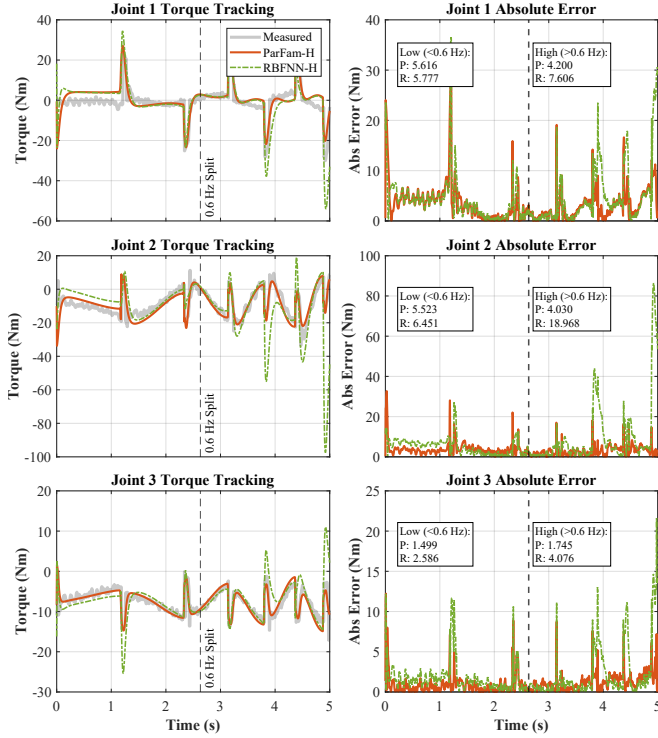


Fig. 5. Segmented frequency analysis under a variable-frequency Chirp signal. The vertical dashed line at 0.6 Hz separates the in-distribution and out-of-distribution regions. ParFam-H maintains stability in the high-frequency region, whereas RBFNN-H degrades significantly.

box models: severe extrapolation failure when encountering unseen dynamic frequencies.

Conversely, ParFam-H maintains robustness in the high-frequency region, with its Joint 2 RMSE even decreasing from 5.523 N·m to 4.030 N·m. Observing the zero-velocity crossings, RBFNN-H exhibits non-physical overshoots and numerical oscillations, whereas the ParFam-H curve remains smooth and tightly tracks the measured values. This demonstrates that the evolved symbolic equations effectively capture the underlying frequency-invariant dynamics, enabling precise predictions beyond the training operating points.

#### D. Real-time Closed-loop Control Performance

To evaluate the ParFam-H model in practical closed-loop scenarios, dynamic trajectory tracking experiments were conducted. Since the commercial UFACTORY-850 cobot, like most of its counterparts, restricts direct joint torque control for safety, we developed a torque-error-based adaptive velocity feedforward controller.

The core architecture utilizes the identified rigid-body and friction models to compute the theoretical total torque  $\tau_d$  for the desired trajectory. The torque error  $e_\tau = \tau_d - \tau_{meas}$  (where  $\tau_{meas}$  is the noisy actual torque estimated from motor currents) is fed into a PD controller to generate a velocity compensation term  $\dot{q}_{comp}$ . To mitigate the amplification of high-frequency noise and maintain closed-loop stability, an adaptive gain scheduling mechanism is introduced (as depicted in the control block diagram, see Fig. 6): the PD gains decay exponentially with respect to the desired velocity  $|\dot{q}_d|$ . This scheduling is governed by dynamics dominance: near zero-velocity crossings, the system transitions through “stick-slip” phases where massive static friction and hysteresis peak. Here, high gains are critical to punch through the static friction barrier and prevent severe phase lags. Conversely, at high speeds, motion is dominated by rigid-body inertia and smooth viscous friction. Maintaining high gains in this region would erroneously amplify high-frequency current sensor noise and potentially excite unmodeled flexible structural resonances. The final commanded velocity sent to the robot is  $\dot{q}_{cmd} = \dot{q}_d + \dot{q}_{comp}$ .

Four compensation strategies were compared: Baseline (rigid-body only, no friction compensation), LuGre, RBFNN-H, and ParFam-H. As illustrated in Fig. 7, the uncompensated baseline suffers from severe phase lags and amplitude deviations during high-acceleration segments (e.g., Joint 2 at 1–3 s), confirming friction as a dominant dynamic disturbance. While all friction models improve tracking, ParFam-H achieves the tightest trajectory adherence. Notably, at  $t \approx 8$  s on Joint 2, the absolute tracking error of the baseline and LuGre models exceeds 0.1 rad, whereas ParFam-H successfully suppresses the peak error to approximately 0.04 rad.

Quantitatively (Table III), ParFam-H yields the lowest RMSE across all three joints. On Joint 2, which experiences the most significant dynamic coupling and gravitational load, ParFam-H achieves an RMSE of 0.0377 rad. This represents a substantial tracking error reduction of 49.3% compared to

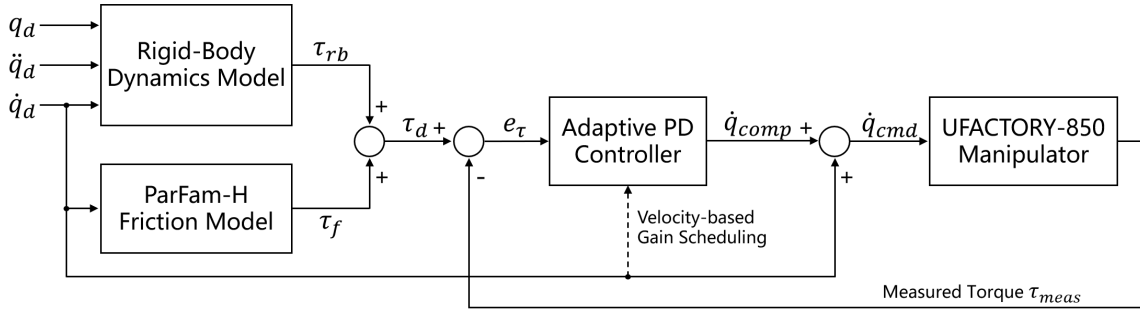


Fig. 6. Control block diagram of the torque-error-based adaptive velocity feedforward controller. The ParFam-H model and rigid-body dynamics generate the reference torque  $\tau_d$ . The adaptive PD controller, scheduled by the reference velocity, compensates for residual errors.

the baseline (0.0744 rad), 38.4% against LuGre (0.0612 rad), and 21.0% against RBFNN-H (0.0477 rad). These results confirm that integrating the explicit ParFam-H model into real-time feedforward compensation significantly enhances the precision and robustness of dynamic trajectory tracking.

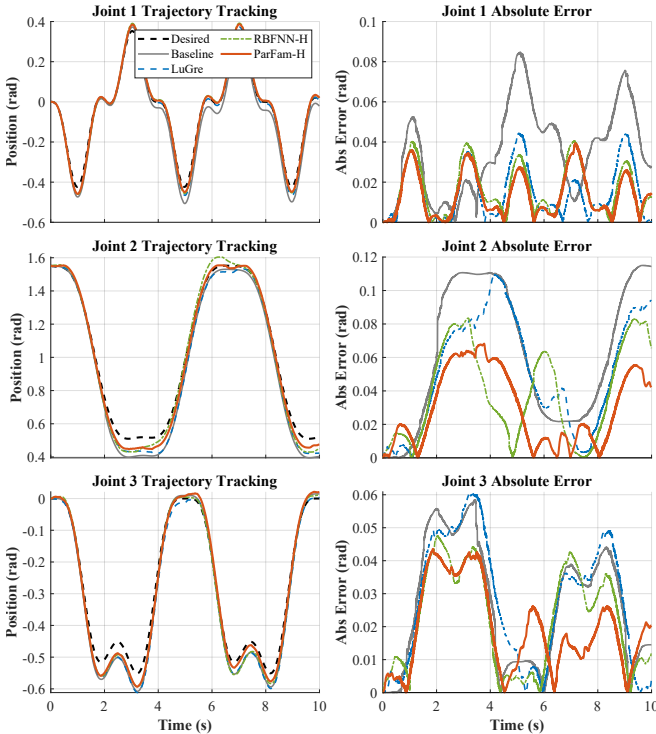


Fig. 7. Dynamic trajectory tracking performance and absolute error comparison among different controllers. ParFam-H minimizes phase lags and peak errors significantly.

## VI. CONCLUSIONS AND FUTURE WORK

This paper presents HESR, a three-stage hysteresis-enhanced symbolic regression framework for manipulator friction modeling. ParFam-H systematically integrates rigid-body dynamics decoupling and a physics-informed hysteresis state. This approach overcomes the structural limitations of traditional parametric models and the opacity of black-box neural networks. Experimental results on a UFACTORY-850 manipulator indicate that the derived explicit equations

TABLE III  
TRAJECTORY TRACKING RMSE UNDER DIFFERENT FRICTION  
COMPENSATION STRATEGIES

Model	Joint 1	Joint 2	Joint 3
Baseline	0.0417	0.0744	0.0324
LuGre	0.0202	0.0612	0.0336
RBFNN-H	0.0204	0.0477	0.0267
<b>ParFam-H</b>	<b>0.0177</b>	<b>0.0377</b>	<b>0.0233</b>

capture dynamic memory effects, reducing offline torque prediction RMSE by over 30% compared to LuGre and RBFNN-H baselines. Variable-frequency Chirp tests demonstrate out-of-distribution physical generalization, avoiding the severe overfitting commonly observed in neural networks. Furthermore, integrating ParFam-H into a real-time feedforward controller reduced closed-loop trajectory tracking errors by 49.3%.

Beyond robotic friction, using a physics-informed hysteresis state variable provides a structured approach for symbolic regression to model systems with non-local memory effects by translating path-dependent dynamics into augmented Markovian features. Future work will investigate the framework's robustness against temperature fluctuations and payload variations, potentially integrating online learning for real-time parameter adaptation. We also plan to evaluate this methodology on other robotic systems governed by complex coupled dynamics, including hydraulic actuators and flexible-joint robots.

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