

Regarding the Case of a 1-Dimensional Tube with a Vibration Source and a Hole in It

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Abstract

While research on automobile whistling and aspiration noise has been conducted in the past, we believe that understanding the properties of whistling and aspiration sounds using a 1-dimensional tube as a simple model is necessary in fields such as automobiles. Therefore, we discussed the case where there is a vibration source in a 1-dimensional tube with a hole. We derived the system of equations that must be satisfied and showed that the results derived using actual numerical examples are valid.

Nomenclature

S, S_H : Cross-sectional area of the 1-dimensional tube and area of the hole in the 1-dimensional tube

l : Position of the hole in the 1-dimensional tube

L : Length of the 1-dimensional tube

V : Volume of the 1-dimensional tube

p_1, v_1, ϕ_1 : Sound pressure and particle velocity and velocity potential for $0 \leq x \leq l$

A_1, B_1 : Amplitude of velocity potential for $0 \leq x \leq l$

p_2, v_2, ϕ_2 : Sound pressure and particle velocity and velocity potential for $l \leq x \leq L$

A_2, B_2 : Amplitude of velocity potential for $l \leq x \leq L$

p_H, v_H : Sound pressure and particle velocity at $x = l$

δ_H : Thickness of the 1-dimensional tube

ρ, c : Density of air and speed of sound in air

ω : Angular frequency

f : Frequency

k : Wavenumber

t : Time

j : Imaginary unit

p_E, v_E, δ_E : Sound pressure, particle velocity, and thickness of a virtual air wall.

1. Introduction

$$\rho\delta_E \frac{\partial v_E}{\partial t} = p_E(t) \quad (4)$$

Here, we show the equations satisfied by the velocity potential, vibration source, hole, and air wall.

$$\phi_1(x, t) = A_1 e^{j(\omega t - kx)} + B_1 e^{j(\omega t + kx)} \quad (5)$$

$$\phi_2(x, t) = A_2 e^{j(\omega t - kx)} + B_2 e^{j(\omega t + kx)} \quad (6)$$

$$v_1(0, t) = v_0 e^{j\omega t} \quad (7)$$

$$v_H = C_H e^{j\omega t} \quad (8)$$

$$v_2(L, t) = C_E e^{j\omega t} \quad (9)$$

We show the equations that the velocity potential, particle velocity, and sound pressure satisfy.

$$v = -\frac{\partial \phi}{\partial x} \quad (10)$$

$$p = \rho \frac{\partial \phi}{\partial t} + C \quad (11)$$

From these equations, the following six simultaneous equations are obtained.

$$A_1 e^{-jkl} + B_1 e^{jkl} = A_2 e^{-jkl} + B_2 e^{jkl} \quad (12)$$

$$S(-jkA_1 e^{-jkl} + jkB_1 e^{jkl}) = S(-jkA_2 e^{-jkl} + jkB_2 e^{jkl}) + S_H C_H \quad (13)$$

$$-jkA_1 + jkB_1 = v_0 \quad (14)$$

$$-jkA_2 e^{-jkl} + jkB_2 e^{jkl} = C_E \quad (15)$$

$$\delta_H C_H = A_1 e^{-jkl} + B_1 e^{jkl} \quad (16)$$

$$\delta_E C_E = A_2 e^{-jkl} + B_2 e^{jkl} \quad (17)$$

The solution obtained from the system of six equations is as follows:

$$D = (jk)^2 S \delta_E \delta_H \sin(kL) - S_H \sin(kl) \sin(k(L-l)) + jk S_H \delta_E \sin(kl) \cos(k(L-l)) + jk S \delta_H \cos(kL) \quad (18)$$

$$C_H = \frac{v_0 S}{D} \{ \sin(k(L-l)) + jk \delta_E \cos(k(L-l)) \} \quad (19)$$

$$C_E = \frac{v_0 S \delta_H (jk)}{D} \quad (20)$$

$$A_1 = \frac{1}{2} \left\{ \delta_H C_H e^{jkl} - \frac{v_0}{jk} \right\} \quad (21)$$

$$B_1 = \frac{1}{2} \left\{ \delta_H C_H e^{-jkl} + \frac{v_0}{jk} \right\} \quad (22)$$

$$A_2 = \frac{1}{2} C_E \left(\delta_E - \frac{1}{jk} \right) e^{jkl} \quad (23)$$

$$B_2 = \frac{1}{2} C_E \left(\delta_E + \frac{1}{jk} \right) e^{-jkl} \quad (24)$$

When equation (18) is 0, the resonance frequency can be found. The fact that the imaginary unit remains in the equation for finding the resonance frequency reflects the physical reality that, unless there is energy dissipation or a complete phase inversion, there is no resonance frequency where the amplitude is exactly infinite, and it is a finite value.

2.2. In the Case of a Single-Sided Open End

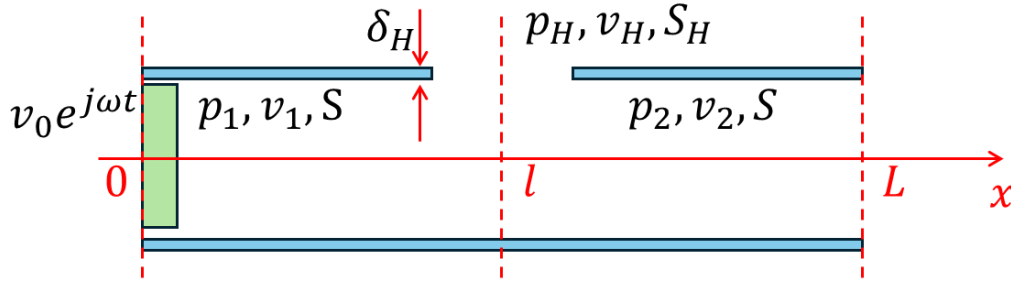


Fig. 2 1-Dimensional Tube which Right Side Opens

Fig. 2 shows a 1-dimensional tube with one end open. The following equations hold true based on the conditions for continuity of sound pressure and particle velocity.

$$p_1(l, t) = p_2(l, t) \quad (25)$$

$$Sv_1(l, t) = Sv_2(l, t) + S_H v_H \quad (26)$$

The equation of motion for the hole is as follows:

$$\rho \delta_H \frac{\partial v_H}{\partial t} = p_H(t) \quad (27)$$

Here, we show the equations satisfied by the velocity potential, vibration source, hole, and open end.

$$v_1(0, t) = v_0 e^{j\omega t} \quad (28)$$

$$v_H = C_H e^{j\omega t} \quad (29)$$

$$p_2(L, t) = 0 \quad (30)$$

From these equations, the following five simultaneous equations are obtained.

$$A_1 e^{-jkl} + B_1 e^{jkl} = A_2 e^{-jkl} + B_2 e^{jkl} \quad (31)$$

$$S(-jkA_1 e^{-jkl} + jkB_1 e^{jkl}) = S(-jkA_2 e^{-jkl} + jkB_2 e^{jkl}) + S_H C_H \quad (32)$$

$$-jkA_1 + jkB_1 = v_0 \quad (33)$$

$$A_2 e^{-jkl} + B_2 e^{jkl} = 0 \quad (34)$$

$$\delta_H C_H = A_1 e^{-jkl} + B_1 e^{jkl} \quad (35)$$

The solution obtained from the system of five equations is as follows:

$$D_{open} = jkS\delta_H \cos(kL) - S_H \sin(kl) \sin(k(L-l)) \quad (36)$$

$$C_H = \frac{v_0 S}{D_{open}} \sin(k(L-l)) \quad (37)$$

$$A_1 = \frac{1}{2} \left\{ \delta_H C_H e^{jkl} - \frac{v_0}{jk} \right\} \quad (38)$$

$$B_1 = \frac{1}{2} \left\{ \delta_H C_H e^{-jkl} + \frac{v_0}{jk} \right\} \quad (39)$$

$$A_2 = -\frac{1}{2} \frac{v_0 S \delta_H}{D_{open}} e^{jkl} \quad (40)$$

$$B_2 = \frac{1}{2} \frac{v_0 S \delta_H}{D_{open}} e^{-jkl} \quad (41)$$

These equations are equivalent to those obtained when $\delta_E = 0$ in the solution of Section 2.1. Furthermore, the physical significance of the imaginary unit being included in equation (36) is as described in Section 2.1.

2.3. In the Case of a Single-Sided Closed End

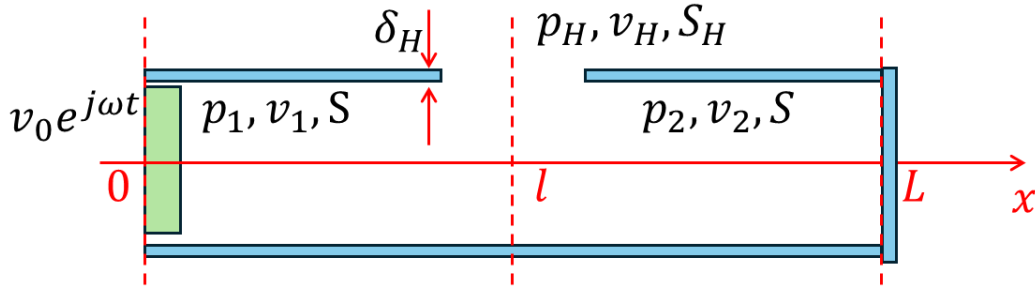


Fig. 3 1-Dimensional Tube which Right Side Closes

Fig. 3 shows a 1-dimensional tube with one end closed. The following equations hold true based on the conditions for continuity of sound pressure and particle velocity.

$$p_1(l, t) = p_2(l, t) \quad (42)$$

$$Sv_1(l, t) = Sv_2(l, t) + S_H v_H \quad (43)$$

The equation of motion for the hole is as follows:

$$\rho\delta_H \frac{\partial v_H}{\partial t} = p_H(t) \quad (44)$$

Here, we show the equations satisfied by the velocity potential, vibration source, hole, and closed end.

$$v_1(0, t) = v_0 e^{j\omega t} \quad (45)$$

$$v_H = C_H e^{j\omega t} \quad (46)$$

$$v_2(L, t) = 0 \quad (47)$$

From these equations, the following five simultaneous equations are obtained.

$$A_1 e^{-jkl} + B_1 e^{jkl} = A_2 e^{-jkl} + B_2 e^{jkl} \quad (48)$$

$$S(-jkA_1 e^{-jkl} + jkB_1 e^{jkl}) = S(-jkA_2 e^{-jkl} + jkB_2 e^{jkl}) + S_H C_H \quad (49)$$

$$-jkA_1 + jkB_1 = v_0 \quad (50)$$

$$-jkA_2 e^{-jkl} + jkB_2 e^{jkl} = 0 \quad (51)$$

$$\delta_H C_H = A_1 e^{-jkl} + B_1 e^{jkl} \quad (52)$$

The solution obtained from the system of five equations is as follows:

$$D_{close} = jkS\delta_H \sin(kL) - S_H \sin(kl) \cos(k(L-l)) \quad (53)$$

$$C_H = \frac{v_0 S}{D_{close}} \cos(k(L-l)) \quad (54)$$

$$A_1 = \frac{1}{2} \left\{ \delta_H C_H e^{jkl} - \frac{v_0}{jk} \right\} \quad (55)$$

$$B_1 = \frac{1}{2} \left\{ \delta_H C_H e^{-jkl} + \frac{v_0}{jk} \right\} \quad (56)$$

$$A_2 = \frac{1}{2} \frac{v_0 S \delta_H(jk)}{D_{close}} e^{jkl} \quad (57)$$

$$B_2 = \frac{1}{2} \frac{v_0 S \delta_H(jk)}{D_{close}} e^{-jkl} \quad (58)$$

These equations are equivalent to those obtained when $\delta_E \rightarrow \infty$ in the solution of Section 2.1. Furthermore, the physical significance of the imaginary unit being included in equation (53)(36) is as described in Section 2.1.

3. Numerical Examples

First, Fig. 4 shows the difference in resonance frequencies between the open end (section 2.2) and the closed end (section 2.3) when there is no hole and when there is a hole.

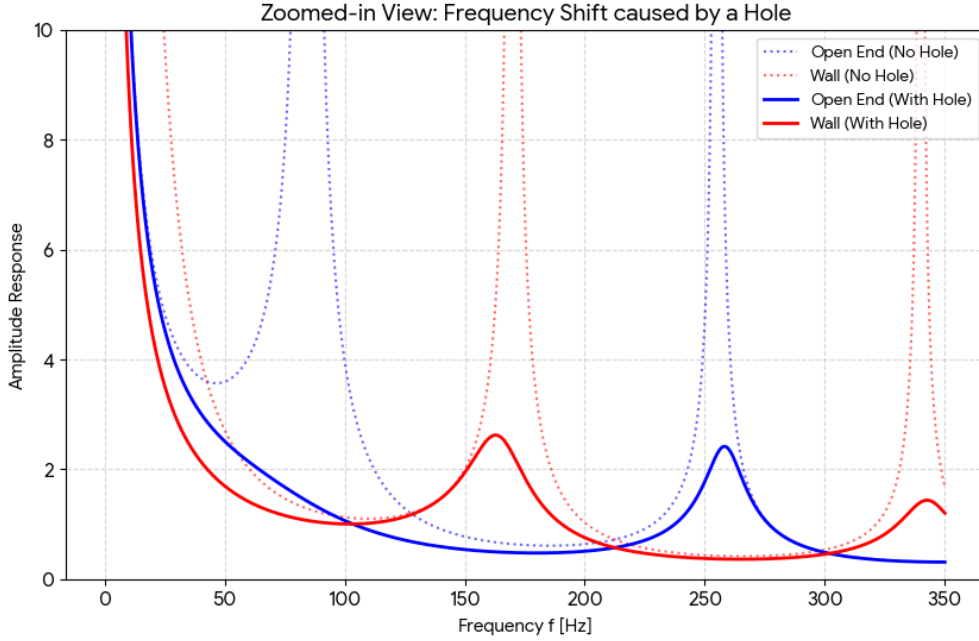


Fig. 4 Resonant Frequency Shift Caused by a Hole

($L = 1.0(\text{m})$, $l = 0.4(\text{m})$, $S = 1.0(\text{m}^2)$, $S_H = 1.5(\text{m}^2)$, $\delta_E = 0.5(\text{m})$, $c = 340(\text{m/s})$)

As seen in Fig. 4, the blue line for the open end shows that the presence of the hole shifts the resonance frequency to the higher frequency side. Conversely, the red line for the closed end shows that the presence of the hole shifts the resonance frequency to the lower frequency side. These results are consistent with the properties of the actual phenomenon of aspiration noise at a closed end and the whistling noise at an open end, indicating that the model is valid.

Next, the table below shows how the resonance frequency changes when the value of δ_E is varied in the results from Section 2.1. The table also shows the results for both the open end and the closed end.

Table 1 Resonant Frequency by changing δ_E
 ($L = 1.0(\text{m})$, $l = 0.4(\text{m})$, $S = 1.0(\text{m}^2)$, $S_H = 1.5(\text{m}^2)$, $c = 340(\text{m/s})$)

δ_E (m)	Resonant Frequency (Hz)
D_{open}	258.2
0.0	258.2
0.05	253.7
0.2	231.4
1.0	185.0
10.0	165.2
D_{close}	162.9

Table 1 shows that as δ_E increases from the open end to the closed end, the resonance frequency decreases, and in the limit, the results are the same as or converge to those of the open and closed ends. This indicates that the model is valid.

4. Conclusion

While research on automobile whistling and aspiration noise has been conducted in the past, we believe that understanding the properties of whistling and aspiration sounds using a 1-dimensional tube as a simple model is necessary in fields such as automobiles. Therefore, we discussed the case where there is a vibration source in a 1-dimensional tube with a hole. We derived the system of equations that must be satisfied and showed that the results derived using actual numerical examples are valid.

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